# INTERNATIONAL STANDARD

ISO 15242-4

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# Rolling bearings — Measuring methods for vibration —

Part 4:

Radial cylindrical roller bearings with cylindrical bore and outside surface

Roulements — Méthodes de mesurage des vibrations —

Partie 4: Roulements radiaux à rouleaux cylindriques, à alésage et surface extérieure cylindriques



Reference number ISO 15242-4:2007(E)

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#### **Foreword**

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

International Standards are drafted in accordance with the rules given in the ISO/IEC Directives, Part 2.

The main task of technical committees is to prepare International Standards. Draft International Standards adopted by the technical committees are circulated to the member bodies for voting. Publication as an International Standard requires approval by at least 75 % of the member bodies casting a vote.

Attention is drawn to the possibility that some of the elements of this document may be the subject of patent rights. ISO shall not be held responsible for identifying any or all such patent rights.

ISO 15242-4 was prepared by Technical Committee ISO/TC 4, Rolling bearings.

ISO 15242 consists of the following parts, under the general title *Rolling bearings* — *Measuring methods for vibration*:

- Part 1: Fundamentals
- Part 2: Radial ball bearings with cylindrical bore and outside surface
- Part 3: Radial spherical and tapered roller bearings with cylindrical bore and outside surface
- Part 4: Radial cylindrical roller bearings with cylindrical bore and outside surface

#### Introduction

Vibration of rotating rolling bearings is a complex physical phenomenon dependent on the conditions of operation. Measuring the vibration output of an individual bearing under a certain set of conditions does not necessarily characterize the vibration output under a different set of conditions or when the bearing becomes part of a larger assembly. Assessment of the audible sound generated by the mechanical system incorporating the bearing is complicated further by the influence of the interface conditions, the location and orientation of the sensing device, and the acoustical environment in which the system operates. Assessment of airborne noise, which for the purpose of ISO 15242 (all parts) can be defined as any disagreeable and undesired sound, is further complicated by the subjective nature of the terms "disagreeable" and "undesired". Structure-borne vibration can be considered the driving mechanism that ultimately results in the generation of airborne noise. Only selected methods for the measurement of the structure-borne vibration of rotating rolling bearings are addressed in the current edition of ISO 15242 (all parts).

Vibration of rotating rolling bearings can be assessed by any of a number of means using various types of transducers and test conditions. No simple set of values characterizing vibration of a bearing is adequate for the evaluation of the vibratory performance in all possible applications. Ultimately, a knowledge of the type of bearing, its application and the purpose of the vibration testing (e.g. as a manufacturing process diagnostic or an assessment of the product quality) is required to select the most suitable method for testing. The field of application for standards on bearing vibration is, therefore, not universal. However, certain methods have established a wide enough level of application to be considered as standard methods for the purposes of this part of ISO 15242.

This part of ISO 15242 serves to define the detailed method for assessing vibration of single-row and double-row radial cylindrical roller bearings with cylindrical bore and outside surface on a test rig.

# Rolling bearings — Measuring methods for vibration —

### Part 4:

# Radial cylindrical roller bearings with cylindrical bore and outside surface

#### 1 Scope

This part of ISO 15242 specifies vibration measuring methods for single-row and double-row radial cylindrical roller bearings, under established test conditions.

It covers single-row and double-row radial cylindrical roller bearings with cylindrical bore and outside surface.

#### 2 Normative references

The following referenced documents are indispensable for the application of this document. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies.

ISO 286-2, ISO system of limits and fits — Part 2: Tables of standard tolerance grades and limit deviations for holes and shafts

ISO 554, Standard atmospheres for conditioning and/or testing — Specifications

ISO 558, Conditioning and testing — Standard atmospheres — Definitions

ISO 1132-1, Rolling bearings — Tolerances — Part 1: Terms and definitions

ISO 2041, Vibration and shock — Vocabulary

ISO 3205, Preferred test temperatures

ISO 3448, Industrial liquid lubricants — ISO viscosity classification

ISO 5593, Rolling bearings — Vocabulary

ISO 15242-1:2004, Rolling bearings — Measuring methods for vibration — Part 1: Fundamentals

#### 3 Terms and definitions

For the purposes of this document, the terms and definitions given in ISO 1132-1, ISO 2041, ISO 5593 and ISO 15242-1 apply.

### **Measurement process**

#### Speed of rotation 4.1

The default speed of rotation shall be 30 s<sup>-1</sup> (1 800 r/min) for bearings with outside diameter up to 100 mm and 15 s<sup>-1</sup> (900 r/min) for outside diameters larger than 100 up to 200 mm. Tolerance of the speed of rotation shall be  $^{+1}_{-2}$  % of the specified nominal value.

Other speeds and tolerances may be used by agreement between the manufacturer and the customer; e.g. it may be necessary to use a higher speed for bearings in the smaller size range [40 s<sup>-1</sup> to 60 s<sup>-1</sup> (2 400 r/min to 3 600 r/min)] in order to obtain an adequate vibration signal. Conversely, it may be necessary to use a lower speed for bearings in the larger size range [7,5 s<sup>-1</sup> to 10 s<sup>-1</sup> (450 r/min to 600 r/min)] to avoid possible roller and raceway damage.

#### Bearing radial and axial loads 4.2

The bearing load shall be in the radial direction with default values as specified in Table 1.

Bearing outside diameter Single-row radial cylindrical **Double-row radial cylindrical** roller bearings roller bearings D Default values for bearing radial load ≤ min. max. min. max. Ν Ν mm 165 165 195 30 50 135 50 70 195 225 275 165 70 100 225 275 315 385 100 140 315 385 430 520 140 170 430 520 565 685 170 200 685 720 565 880

Table 1 — Default values for bearing radial load

Other radial loads and tolerances may be used by agreement between the manufacturer and the customer; e.g. depending on bearing design and lubricant used, it may be necessary to use a higher load to prevent roller/raceway slip, or a lower load to avoid possible roller, rib and raceway damage.

For bearings capable of taking axial load, an axial load of up to 30 N shall be applied on the outer ring to ensure stable operation.

The method of applying the radial and axial loads is described in 6.3.3.

NOTE Default values for radial loads are resultant values. Actual values depend on load angle used (see Figure 3).

#### 5 Measurement and evaluation methods

#### 5.1 Physical quantity measured

The default physical quantity to be measured is vibration velocity,  $v_{r.m.s.}$  (µm/s), in the radial direction.

#### 5.2 Frequency range

The velocity signal shall be measured in one or more bands with default frequency ranges as specified in Table 2.

Table 2 — Default frequency ranges

Speed of rotation		Low band (L) <sup>a</sup>		Medium band (M) <sup>a</sup>		High band (H) <sup>a</sup>	
		Default frequencies					
min.	max.	$f_{low}$	$f_{high}$	$f_{low}$	$f_{high}$	$f_{low}$	$f_{high}$
r/min		Hz		Hz		Hz	
882	909	50	150	150	900	900	5 000
1 764	1 818	50	300	300	1 800	1 800	10 000

<sup>&</sup>lt;sup>a</sup> For rotational speeds other than the nominal 900 r/min or 1 800 r/min, the frequency range should be adjusted in proportion to the speed. For practical reasons, frequencies lower than 50 Hz or higher than 10 000 Hz should not be used, unless agreed upon between the manufacturer and the customer.

NOTE Other frequency ranges may be considered by agreement between the manufacturer and the customer in those instances where specific ranges have greater importance to successful operation of the bearing.

The use of spectral analysis of the vibration signal is an alternative.

#### 5.3 Peak measurement

Detection of peaks or spikes in the time domain velocity signal, usually due to surface defects and/or contamination in the test bearing, may be considered as a supplementary option by agreement between the manufacturer and the customer. Various evaluation methods exist depending on the bearing type and the application.

#### 5.4 Testing sequence

Single-row and double-row radial cylindrical roller bearings shall be tested with the radial load applied in a radial direction on the outer ring and perpendicular to the inner ring axis. An axial load may be necessary to ensure stable operation. If the axial load is used, it is applied from one side of the outer ring. For double-row radial cylindrical roller bearings the test should be repeated, if the design allows, with the axial load on the other side of the outer ring.

For diagnostic purposes, performing multiple measurements with the outer ring in different angular positions relative to the pick-up is appropriate.

For acceptance of the bearing, the highest vibration reading for the appropriate frequency band shall be within the limits mutually agreed between the manufacturer and the customer.

For test duration, see ISO 15242-1:2004, 6.5.

#### 6 Conditions for measurement

#### 6.1 Bearing conditions for measurement

#### 6.1.1 Prelubrication

Prelubricated (greased, oiled or solid lubricated) bearings, including sealed and shielded types, shall be tested in the as-delivered condition.

NOTE Some greases, oils and solid lubricants increase or decrease bearing vibration levels in comparison with the reference conditions in 6.1.2 and 6.1.3.

The following reference condition procedures (6.1.2 and 6.1.3) normally apply for bearings that are not prelubricated. However, they may also be used in cases of dispute regarding the source of unacceptable vibration levels.

#### 6.1.2 Cleanliness of the bearing

Since contamination affects vibration levels, the bearings must be cleaned effectively, taking care not to introduce contamination or other sources of vibration.

NOTE Some preservatives may meet the lubrication requirements (see 6.1.3) for vibration testing. In this case, it is not necessary to remove the preservative.

#### 6.1.3 Lubrication

Before testing, bearings shall be adequately lubricated with filtered oil (0,8 µm maximum filter), having a nominal viscosity in the range of 10 mm<sup>2</sup>/s to 100 mm<sup>2</sup>/s. Additional information is given in ISO 3448.

The lubrication procedure shall include some running-in to achieve homogeneous distribution of the lubricant within the bearing.

NOTE Other lubricant viscosities may be agreed upon between the manufacturer and the customer in order to suit the application.

#### 6.2 Conditions of the test environment

The bearings shall be tested at room temperature in an environment that does not influence the bearing vibration. Additional information is given in ISO 554, ISO 558 and ISO 3205.

#### 6.3 Conditions for the test device

#### 6.3.1 Stiffness of the spindle/mandrel arrangement

The spindle (including the mandrel) used to hold and drive the bearing inner ring shall be so designed and constructed that, except for transmittal of rotary motion, it represents essentially a rigid reference system for the inner ring axis. The transmission of vibration between the spindle/mandrel arrangement and the bearing inner ring in the frequency band used shall be negligible by comparison to the velocities measured (in cases of dispute, precise values shall be agreed between the manufacturer and the customer).

#### 6.3.2 Loading mechanism

The loading system used to apply load to the bearing outer ring shall, ideally, be designed and constructed so that it leaves the ring essentially free to vibrate in all radial, axial, angular or flexural modes according to the bearing type.

#### 6.3.3 Magnitude and alignment of the external load applied to the bearing

A constant external radial load of the magnitude specified in 4.2 together with the recommended axial load, if applicable, shall be applied to the outer ring.

The distortion of the bearing rings, caused by contact with elements of the mechanical unit, shall be negligible in comparison to the inherent geometrical accuracy of the bearing under test.

The position of the externally applied radial load shall coincide with the middle of the outer ring width. The direction of the externally applied radial load shall coincide with the axis perpendicular to the spindle axis of rotation. The position and the direction shall be within the limits given in Figure 1 and Table 3. See Annex A for a measurement description.

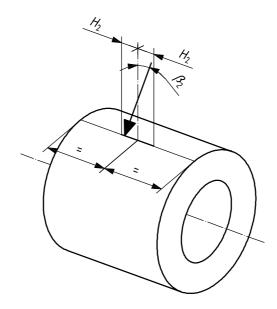
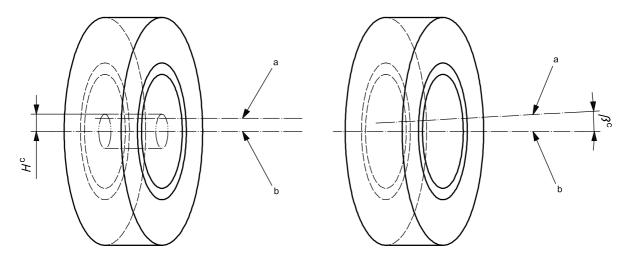


Figure 1 — Radial load direction deviation in relation to direction and axial position

Table 3 — Values for radial load direction deviation in relation to direction and axial position

Outer ring width  C		Axial deviation from the middle of the outer ring width $H_2$	Angular deviation from the axis perpendicular to the spindle axis $eta_2$	
>	$\leqslant$	max.	max.	
mm		mm	۰	
10	20	0,3		
20	40	0,5	1	
40	70	1		

The position and direction of the externally applied axial load shall coincide with the spindle axis of rotation within the limits given in Figure 2 and Table 4. See Annex B for a measurement description.



- Axis of externally applied load.
- Axis of bearing inner ring rotation.
- See Table 4.

Figure 2 — Axial load axis deviation in relation to axis of bearing inner ring rotation

Table 4 — Values for axial load axis deviation in relation to axis of bearing inner ring rotation

Bearing outside diameter  D		Radial deviation from axis of bearing inner ring rotation ${\it H}$	Angular deviation from axis of bearing inner ring rotation $eta$	
>	< <	max.	max.	
m	m	mm	٥	
30	50	0,4		
50	70	0,6		
70	100	0,8	0,5	
100	140	1,6	0,3	
140	170	2,0		
170	200	2,5		

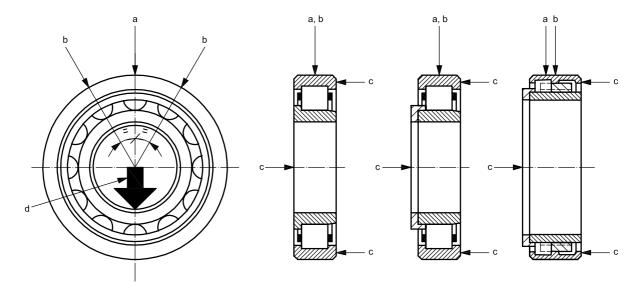
#### 6.3.4 Location of pick-up and direction of measurement

The pick-up shall be placed and orientated as follows:

Default axial location: On the outside surface of the outer ring in the plane corresponding to the middle of the loaded outer ring raceway/roller contacts (see Figure 3). The manufacturer shall supply this datum.

Default angular location: On the outside surface of the outer ring in the plane corresponding to the direction of the resultant radial load (see Figure 3).

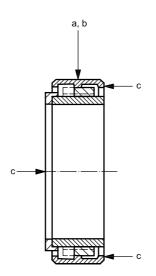
The radial loads should be applied in a manner that can be resolved into a single radial load as shown in Table 1.



- a Pick-up location.
- b Applied radial load.
- c Axial load (if any) direction.
- d Resultant of applied radial loads (see Table 1).

NOTE Other rib configurations may be tested in a manner agreed between manufacturer and customer.

Figure 3 — Measurement — Pick-up default location



- <sup>a</sup> Pick-up location.
- b Applied radial load.
- c Axial load (if any) direction.

Figure 4 — Measurement — Alternative pick-up location

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Once the pick-up position is determined, the maximum permissible axial and angular deviations are:

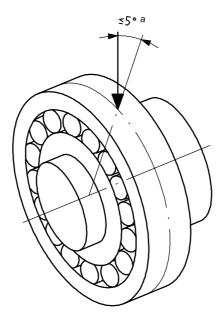
#### **Axial location:**

- for outside diameter ≤ 70 mm: ±0,5 mm;
- for outside diameter > 70 mm:  $\pm$ 1,0 mm.

#### **Angular location:**

for all outside diameters: ±5°.

Direction: Perpendicular to the axis of rotation (see Figure 5). The deviation from a radial axis shall not exceed 5° in any direction.



In any direction.

Figure 5 — Deviation from a radial axis

#### 6.3.5 Mandrel

The cylindrical surface of the mandrel, on which the inner ring of the bearing is mounted, shall have an outside diameter to tolerance class f5, according to ISO 286-2, with minimal geometric errors. This will ensure a sliding fit in the bearing bore.

#### Requirements for the operator

A competent operator shall ensure that vibration measurements are taken in accordance with this part of ISO 15242.

# Annex A

(normative)

# Measurement of external radial loading alignment

The alignment of radial loading tools shall be measured using two dial gauges, with radial distance between them, mounted on a bar attached to the spindle shaft and perpendicular to the spindle axis (see Figure A.1). The two dial gauges shall be set to zero and in same distance from the bar side face.

The measured reading deviation from the two dial gauges shall be recalculated to the radial loading tool perpendicularity deviation, to enable comparison with the limits given in Table 3. The measurement shall be carried out on both loading tool positions.

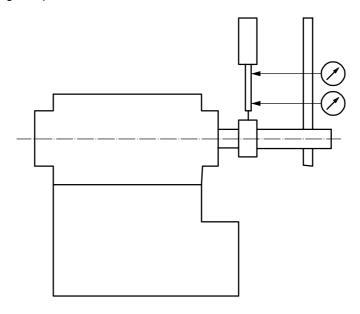


Figure A.1 — Measurement — External radial loading alignment

### **Annex B** (normative)

## Measurement of external axial loading alignment

The displacement of the loading tool shall be measured using two dial gauges, with an axial distance between them, mounted on a bar attached to the spindle shaft (see Figure B.1). The spindle shaft shall be rotated slowly and the dial gauges shall measure the radial runout of the loading piston.

The measured radial runout from the two dial gauges shall be corrected to the test bearing axial position, to enable comparison with the limits given in Table 4.

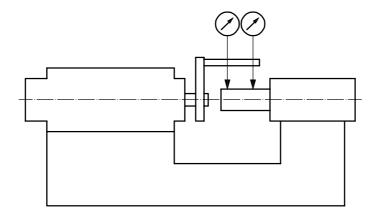


Figure B.1 — Measurement — External axial loading alignment

ICS 21.100.20

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