INTERNATIONAL STANDARD

ISO 6971

Second edition 2002-09-15

Cranked-link drag chains of welded construction, attachments and sprockets

Chaînes racleuses en acier, de construction soudée, à maillons coudés, plaques-attaches et roues dentées



Reference number ISO 6971:2002(E)

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ISO 6971:2002(E)

Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

International Standards are drafted in accordance with the rules given in the ISO/IEC Directives, Part 3.

The main task of technical committees is to prepare International Standards. Draft International Standards adopted by the technical committees are circulated to the member bodies for voting. Publication as an International Standard requires approval by at least 75 % of the member bodies casting a vote.

Attention is drawn to the possibility that some of the elements of this International Standard may be the subject of patent rights. ISO shall not be held responsible for identifying any or all such patent rights.

ISO 6971 was prepared by Technical Committee ISO/TC 100, Chains and chain wheels for power transmission and conveyors.

This second edition cancels and replaces the first edition (ISO 6971:1982), which has been technically revised. In particular, the references and terminology have been revised to bring them into conformity with other International Standards and current practice. Furthermore, in clause 5, the pitch line clearance of sprockets has been altered for consistency with current industry practice.

Cranked-link drag chains of welded construction, attachments and sprockets

1 Scope

This International Standard specifies the characteristics of cranked-link¹⁾ drag chains of welded construction suitable for conveying bulk materials, together with associated attachments and chain sprockets. The chain dimensions specified in this International Standard ensure interchangeability of both complete chains and individual links for repair purposes.

This International Standard is applicable to sprockets with between 5 and 20 teeth.

Specifications are also given for five types of attachment for use with the conveyor chains conforming to this International Standard.

2 Normative reference

The following normative document contains provisions which, through reference in this text, constitute provisions of this International StandardInternational Standard. For dated references, subsequent amendments to, or revisions of, any of these publications do not apply. However, parties to agreements based on this International Standard International Standardare encouraged to investigate the possibility of applying the most recent edition of the normative document indicated below. For undated references, the latest edition of the normative document referred to applies. Members of ISO and IEC maintain registers of currently valid International Standards.

ISO 286-2, ISO system of limits and fits — Part 2: Tables of standard tolerance grades and limit deviations for holes and shafts

3 Chains

3.1 General

The chain is designed to operate with the closed end of each link in the forward direction to produce the maximum scraping action against the material to be conveyed.

3.2 Nomenclature

The nomenclature of the chains and their component parts is specified in Figures 1 and 2.

3.3 Dimensions

Conveyor chain dimensions shall conform to those given in Table 1 (see Figure 3). Both maximum and minimum dimensions are specified to ensure interchangeability of links made by different chain manufacturers. Although these represent limits for interchangeability, they shall not necessarily be regarded as limits of tolerance for manufacture.

¹⁾ In the USA, the term "offset sidebar" is used in place of "cranked link".

3.4 Tensile strength

3.4.1 Minimum tensile strength

The minimum tensile strength is that value which shall be exceeded when a tensile force is applied to a sample which is tested to destruction in accordance with 3.4.2.

NOTE This minimum tensile strength is not a working force. It is intended primarily as a comparative figure between chains of different construction. For application information, it is necessary to consult the manufacturers or their published data.

3.4.2 Tensile testing

A tensile force, not less than the minimum tensile strength specified in Table 1, shall be applied slowly to the ends of a chain, containing a minimum of three free pitches, by means of shackles so designed as to allow universal movement. The actual test method is at the discretion of the manufacturer.

Failure shall be considered to have occurred at the first point where increasing extension is no longer accompanied by increasing force, i.e. the summit of the force/extension diagram.

Any test in which failure occurs adjacent to the shackles shall be disregarded.

3.5 Length accuracy

Finished chains shall be measured either in the dry state or after light lubrication.

The standard nominal length for measurement shall be that nearest to 3 048 mm.

The chain shall be supported throughout its length and the measuring force specified in Table 1 shall be applied.

The finished chain length shall be equal to the nominal chain length $^{+}$ 0,32 % .

Chains that work in parallel may be matched by agreement between the purchaser and the manufacturer.

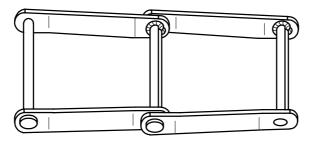
3.6 Designation

The designation numbers for welded-steel-type cranked link drag chains are based on the ISO chain numbers given in Table 1. These numbers are derived from those given to the cast type which they replace and have been given the prefix WD to indicate that they are of welded design.

3.7 Marking

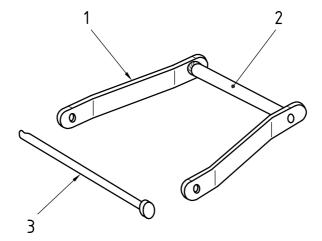
The chains shall be marked with the manufacturer's name or trademark and in addition should be marked with the appropriate ISO chain number given in Table 1.

The marking of the chain shall not be obscured by the attachments.



NOTE The illustration does not define the actual form of the cranked link.

Figure 1 — Cranked link chain assembly



Key

- 1 Cranked plate
- 2 Barrel
- 3 Connecting pin

NOTE The illustration does not define the actual form of the cranked link.

Figure 2 — Typical cranked link components

The overall width of connecting links is

Łj

 $b_4 + b_5$, when there is a fastener at $b_5 + b_6$, when riveted

ď

one side $2b_{4}$, when there is a fastener at both

7-7

The line of cranking, or offset, between I_1 and I_2 is straight.

٤p d d 14 1 $p_{\rm S}$ p_{ϵ} p_{t} ^ζP φ

91

 ϕd_{j}^{a}

The illustration does not define the actual form of the cranked link. NOTE

Direction of travel

(ref.)

Pitch line

Figure 3 — Chain dimensions and symbols (see Table 1)

Table 1 — Chain dimensions, measuring forces and tensile strengths

Tensile strength	betsert treated		min.		245	245	245	245	253	262	351	351	351	
Tensile	Pin heat treated		min.	kN	170	170	170	170	213	245	311	311	311	
	Measuring force				2,7	1,8	3,1	2,7	3,6	3,6	2,8	4	4,4	
	Chain plate thickness	С	nom.		2'6	2,6	2,6	2,6	12,7	2,6	12,7	12,7	12,7	
	Width over rivet to centreline	9_q	max.		127,8	94	157,5	157,5	165,1	205,2	220	162,1	193,8	
6	Width over pin head to centreling	p_{5}	max.		117,6	87,4	151,1	151,1	157,2	200,7	211,1	152,4	184,2	
əui	Width over pin fastening to centrel	b_4	max.		127,8	94	157,5	157,5	165,1	205,2	220	162,1	193,8	
р	Width between plates at outer en	p_3	min.		197,6	137,4	264,4	264,4	270,7	359,6	378,7	261,1	324,6	dual links.
	Width over link at inner end	b_2	тах.		1,261	136,9	263,9	263,9	270,2	359,1	378,2	260,6	324,1	οf indivic
	ו ומנפ בוות בובמומונים חווובוים וווי	1/4	max.		25,4	25,4	25,4	25,4	25,4	28,4	35	35	35	inspection
	Plate end clearance dimensions	ار3	max.		25,4	25,4	25,4	25,4	25,4	28,4	35	35	31,7	use in the
	CIONSI ANIM ANIM MANA MINIMANA	l_2	min.		25,6	25,6	25,6	25,6	25,6	28,7	35,3	35,3	35,3	nded for u
Crank clearance dimensions		۲,	min.		25,6	25,6	25,6	25,6	25,6	28,7	35,3	35,3	32	is not inte
	Barrel diameter or plate depth	h_2	max.	mm	38,12	38,12	38,12	38,12	38,12	45,21	51,8	51,8	51,8	is and sprocket dimensions, and is not intended for use in the inspection of individual links
	Chain path depth	h_1	max.		39,6	39,6	39,6	39,6	39,6	46	52,3	52,3	52,3	ocket dimer
	Connecting pin body diameter	d_2	max.		19,13	19,13	19,13	19,13	22,3	19,13	22,3	22,3	22,3	
1:	Width between plates for sprocke contact at inner end	b_1	min.		162	104,6	228,6	228,6	228,6	330,2	336,5	222,2	282,4	The pitch, $ ho$ is a theoretical reference dimension used in the calculation of chain lengt
		8/	max.		19,6	19,6	19,6	19,6	19,6	22,6	25,9	25,9	25,9	alculation
		4			17,5	17,5	17,5	17,5	17,5	20,6	23,9	23,9	23,9	d in the co
shape		9,			14,2	14,2	14,2	14,2	15,7	16	20,6	20,6	20,6	nsion use
Barrel shape	Plate hole for barrel bore	d_3	min.		19,25	19,25	19,25	19,25	22,43	19,25	22,43	22,43	22,43	ence dime
	Z-Z noitoes of		тах.		39,1	39,1	39,1	39,1	39,1	45,2	51,8	51,8	51,8	tical refere
	Width accross the barrel with respect	d,	nom.		38,1	38,1	38,1	38,1	38,1	44,45	8'09	8'09	8,03	s a theore
Pitch		$p^{\mathbf{a}}$			127	152,4	152,4	203,2	152,4	203,2	203,2	203,2	203,2	pitch, p, is
OSI	chain number	_			WD102	WD104	WD110	WD112	WD113	WD116	WD118	WD122	WD480	a The

Attachments

4.1 **Types**

This International Standard specifies five types of attachment designated C1, C3, C4, RR and wing with the following characteristics:

- C1, C3, and C4: have a scraper bar attached to the barrel perpendicular to the direction of travel, as shown in Figure 4;
- RR: has a triangular spur attached to each cranked plate, as shown in Figure 5;
- wing: has an angle section attached to the outer face of each cranked plate, as shown in Figure 6.

Dimensions 4.2

The respective dimensions of the attachments shall be as specified in Tables 2 to 6.

NOTE The actual form of the attachments is at the discretion of the manufacturer.

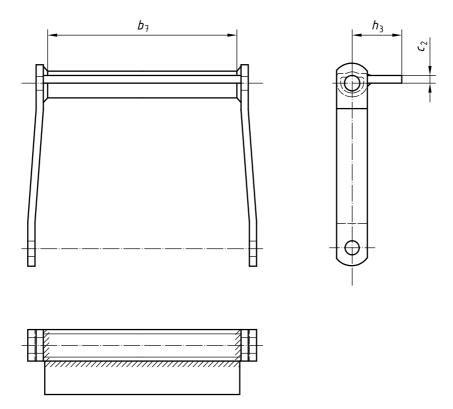


Figure 4 — C1, C3 and C4 attachments (see Tables 2, 3 and 4)

Table 2 — Dimensions of C1 attachment

Dimensions in millimetres

ISO chain number	c_2	b_7 max.	h ₃ max.
WD102	9,7	197,1	62
WD104	9,7	136,9	62
WD110	9,7	263,9	62
WD112	9,7	263,9	62
WD116	9,7	359,2	68,1

Table 3 — Dimensions of C3 attachment

Dimensions in millimetres

ISO chain number	c ₂	b ₇ max.	h ₃ max.
WD110	12,7	263,9	58,7
WD113	12,7	270,3	58,7
WD118	12,7	378,2	77,7
WD480	12,7	324,1	77,7

Table 4 — Dimensions of C4 attachment

Dimensions in millimetres

ISO chain number	c_2	b_7 max.	h_3 max.
WD102	9,7	197,1	96,8
WD104	9,7	136,9	96,8
WD110	9,7	263,9	96,8
WD112	9,7	263,9	96,8
WD113	12,7	270,3	122,2
WD116	9,7	359,2	125,5
WD480	12,7	324,1	128,5

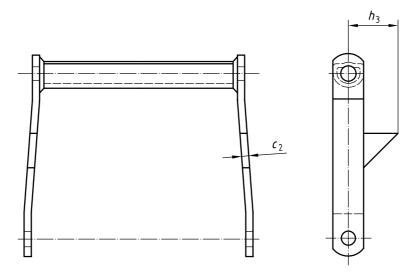


Figure 5 — RR attachment (see Table 5)

Table 5 — Dimensions of RR attachment

Dimensions in millimetres

ISO chain number	h ₃ max.	c ₂
WD102	65	9,7
WD104	65	9,7
WD110	65	9,7
WD112	65	9,7
WD113	65	12,7
WD116	77,7	9,7
WD118	79,2	12,7
WD480	84,1	12,7

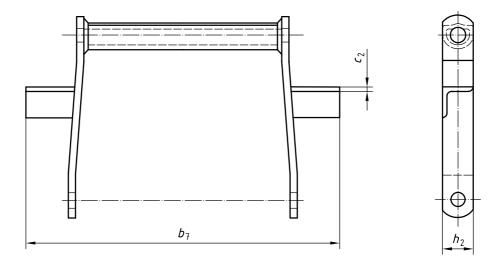


Figure 6 — Wing attachment (see Table 6)

Table 6 — Dimensions of wing attachment

Dimensions in millimetres

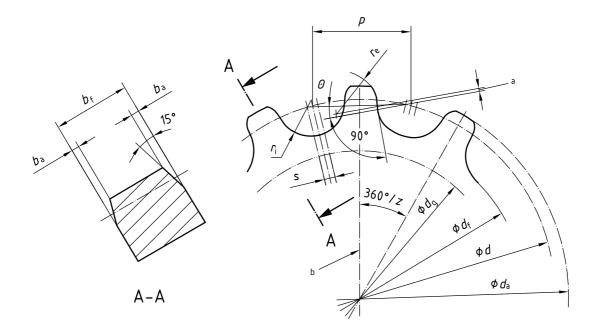
ISO chain number	c ₂	b_7 max.	h ₂ max.
WD102	9,7	365,3	39,6
WD104	9,7	295,1	39,6
WD110	9,7	434,8	39,6
WD112	9,7	434,8	39,6
WD113	9,7	434,8	39,6
WD116	9,7	561,8	46
WD480	9,7	561,8	52,3

5 Sprockets

5.1 Diametral dimensions

5.1.1 General

The sprocket diametral dimensions are shown in Figure 7 and specified in 5.1.2 to 5.1.6.



Key

- tooth side relief
- tooth width
- pitch circle diameter d
- tip diameter d_{a}
- root diameter
- maximum chain clearance diameter d_{g}

- chordal pitch, equal to chain pitch
- tooth flank (topping radius)
- barrel seating radius
- pitch line clearance
- number of teeth
- pressure angle

- а Working face
- Centreline of tooth

Figure 7 — Diametral dimensions and tooth form

5.1.2 Pitch circle diameter, d

$$d = p \times p_{cf}$$

where p_{cf} is the pitch diameter factor according to the number of teeth, as specified in 5.2.4 and Table 7.

5.1.3 Tip diameter, d_a

$$d_{a} = (p \times d_{gf}) + h_{2}$$

where d_{of} is the chain clearance diameter and outside diameter factor according to the number of teeth, as specified in 5.2.5 and Table 7.

The tip diameter may be increased to give a full height tooth when the top of the chain is clear of flights, pans, buckets, etc.

5.1.4 Measuring pin diameter, d_R

$$d_{\mathsf{R}} = d_{\mathsf{1}}$$

where d_1 is the width across the barrel, as specified in Table 1.

5.1.5 Root diameter, $d_{\rm f}$

$$d_f$$
 max. = $(p \times p_{cf}) - d_1$

NOTE Root diameters exceeding the maximum obtained from this equation result in improper chain and sprocket action and excessive chain loads.

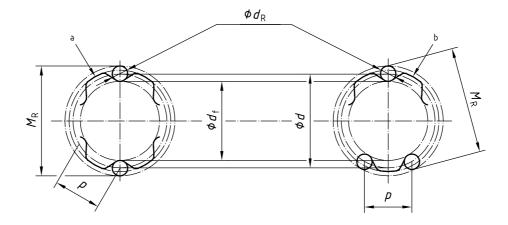
5.1.6 Measurement over measuring pins, M_R

The measurement over measuring pins is illustrated in Figure 8.

 M_R for even numbers of teeth = $d + d_R$ min.

$$M_{\rm R}$$
 for odd numbers of teeth = $d \cos \frac{90^{\circ}}{z} + d_{\rm R}$ min.

For a sprocket having an even number of teeth, measurement shall be made over the appropriate pins inserted in diametrically opposed tooth spaces. For a sprocket having an odd number of teeth, measurement shall be made over pins inserted in the two tooth spaces most nearly diametrically opposite with respect to each other. During measurement, the pins shall remain in contact with the root diameter of the corresponding teeth.



Key

- d pitch circle diameter
- d₅ root diameter
- d_{R} diameter of measuring pins
- M_{R} measurement over measuring pins
- p chordal pitch, equal to chain pitch
- a Even number of teeth
- b Odd number of teeth

Figure 8 — Measurement over measuring pins

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5.2 Tooth gap form

5.2.1 General

The tooth gap form shall have tooth flanks or a profile defined by the tooth flank (topping radius) $r_{\rm e}$, the working face length and the barrel seating radius r_i with a smooth blending from one portion to the next.

5.2.2 Working face

The working face is the functional part of the tooth form. It shall not extend beyond the line through the adjacent pitch point which is perpendicular to the working face.

The length of the working face shall be equal to $0.01p \times z$.

NOTE The working face length provides for approximately 6 % chain pitch elongation.

5.2.3 Pressure angle, θ

The pressure angle is the angle between the pitch line of the chain link and the line perpendicular to the working face at the point of barrel contact. The pressure angle at any point on the working face shall be in accordance with Table 7.

5.2.4 Pitch diameter factor, p_{cf}

$$p_{\rm cf} = {\rm cosec}\bigg(\frac{180^{\circ}}{z}\bigg)$$

Values for p_{cf} are given in Table 7.

5.2.5 Chain clearance diameter and outside diameter factor, d_{qf}

$$d_{\mathsf{gf}} = \cot\left(\frac{180^{\circ}}{z}\right)$$

Values for d_{qf} are given in Table 7.

5.2.6 Pitch line clearance, s

$$s = 0.3 p$$

5.2.7 Barrel seating radius, r_i

$$r_{i}$$
 max. = 0,5 d_{1}

5.2.8 Tooth flank (topping radius), r_e

$$r_{\rm e} = 0.5 \, p$$

Table 7 — Pitch diameter factor, chain clearance diameter and outside diameter factor and pressure angles

Number of teeth	Pitch diameter factor	Chain clearance diameter and outside diameter factor	Pressure angle
z	p_{cf}	d_{gf}	heta°
5	5 1,701		8
6	2,000	1,73	9
7	2,304	2,07	10
8	2,613	2,41	11
9	2,923	2,74	12
10	3,236	3,07	13
11	3,549	3,40	14
12	3,863	3,73	15
13	4,178	4,05	16
14	4,494	4,38	17
15	4,809	4,70	18
16	5,125	5,03	19
17	5,442	5,35	20
18	5,758	5,67	20
19	6,075	5,99	21
20	6,392	6,31	21

5.3 Rim profile

5.3.1 Tooth width, b_f

$$b_{\rm f}$$
 max. = 0,95 $b_{\rm 1}$

Values for b_1 are given in Table 1.

5.3.2 Tooth side relief, b_a

$$b_{a} = 0.12 b_{f}$$

 $b_{\rm a}$ shall not exceed 9,6 mm.

5.3.3 Maximum chain clearance diameter, $d_{\rm q}$

$$d_{g} = p \left(d_{gf} - 0.05 \right) - h_{2}$$

The circle corresponding to this diameter defines the limit beyond which no portion of the hubs, beads, lugs or fillets shall extend in the proximity of the chain side plates.

5.4 **Tolerances**

5.4.1 Radial run-out

The radial run-out between the bore and the root diameter shall not exceed the values given in Table 8.

5.4.2 Axial run-out

The axial run-out, measured with reference to the bore and the flat part of the side face of the teeth, shall not exceed the values given in Table 8.

5.4.3 Bore

Unless otherwise specified by agreement between the manufacturer and purchaser, bores shall be machined to the H9 limits specified in ISO 286-2.

Table 8 — Tolerances

Dimensions in millimetres

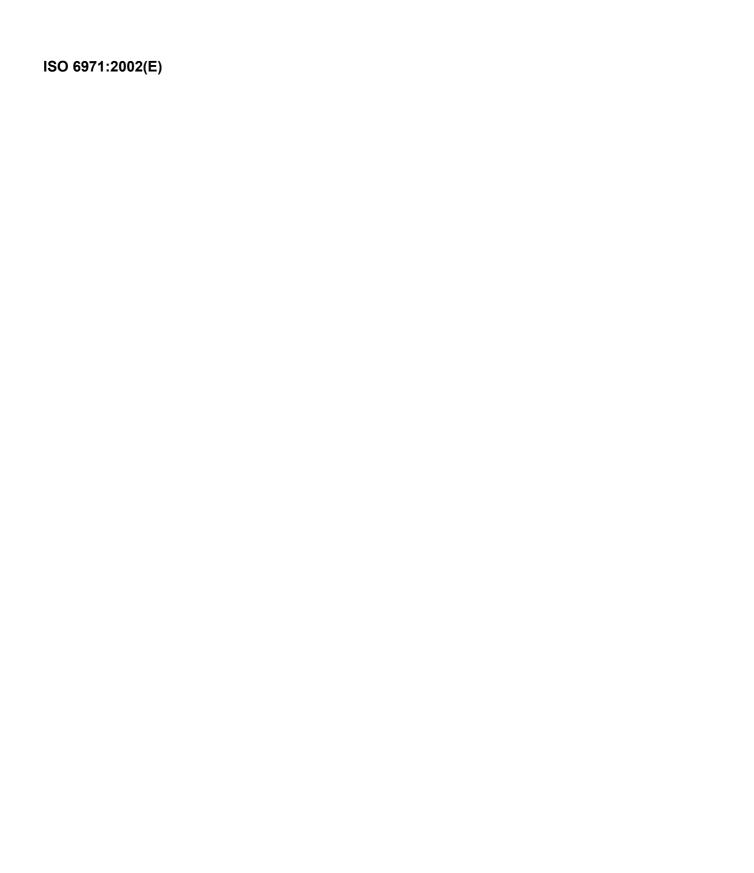
Pitch circle diameter	Root diameter radial run-out	Tooth side face axial run-out	
d			
< 305	1,524	2,286	
305 to 609	3,048	3,81	
610 to 914	5,08	5,334	
915 to 1 219	7,62	6,858	
1 220 to 1 524	8,382	8,382	
1 525 to 1 830	9,144	9,906	

5.5 Marking

It is recommended that sprockets be marked with the following information:

- manufacturer's name or trademark;
- number of teeth; b)
- ISO chain number (see Table 1). c)

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