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INTERNATIONAL STANDARD

ISO 5864

Second edition 1993-12-15

ISO inch screw threads — Allowances and tolerances

Filetages ISO en inches — Jeux et tolérances



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Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

Draft International Standards adopted by the technical committees are circulated to the member bodies for voting. Publication as an International Standard requires approval by at least 75 % of the member bodies casting a vote.

International Standard ISO 5864 was prepared by Technical Committee ISO/TC 1, Screw threads, Sub-Committee SC 2, Tolerances.

This second edition cancels and replaces the first edition (ISO 5864:1978), tables 2 and 4 of which have been technically revised.

ISO inch screw threads — Allowances and tolerances

1 Scope

This International Standard specifies a system of allowances and tolerances for standard thread series, covering the range of diameters from 0,06 in to 6 in, and pitches from 80 to 4 threads per inch.

2 Normative references

The following standards contain provisions which, through reference in this text, constitute provisions of this International Standard. At the time of publication, the editions indicated were valid. All standards are subject to revision, and parties to agreements based on this International Standard are encouraged to investigate the possibility of applying the most recent editions of the standards indicated below. Members of IEC and ISO maintain registers of currently valid International Standards.

ISO 263:1973, ISO inch screw threads — General plan and selection for screws, bolts and nuts — Diameter range 0.06 to 6 in.

ISO 725:1978, ISO inch screw threads — Basic dimensions.

ISO 5408:1983, Cylindrical screw threads — Vocabulary.

3 Definitions

For the purposes of this International Standard, the definitions given in ISO 5408 apply.

In this International Standard, the terms "external threads" and "internal threads" are synonymous with bolt threads and nut threads, respectively, as used in some other International Standards.

4 Symbols and abbreviations

- Major diameter of external thread (nominal diameter)
- d₂ Pitch diameter of external thread
- d_1 Minor diameter of external thread
- Major diameter of internal thread (nominal diameter)
- D₂ Pitch diameter of internal thread
- D₁ Minor diameter of internal thread
- P Pitch
- n Number of threads per inch
- Length of thread engagement (for formula)
- LE Length of thread engagement (for designation)
- SE Special length of thread engagement (for designation)
- PD Pitch diameter (for designation)
- MOD Modified diameter limits (for designation)
- Height of fundamental triangle
- T_d
- T_{d2}
- T_{D2} Tolerances for d, d_2 , D_2 and D_1
- T_{D1}

5 Sizes and series

Nominal sizes, basic dimensions and the thread pitches available in each size and the pitch series in which they are classified appear in ISO 263 and ISO 725.

Thread series are groups of diameter-pitch combinations distinguished from each other by the number of threads per inch applied to series of specific diameters. The various diameter-pitch combinations are of three series with graded pitches, coarse (UNC), fine (UNF) and extra fine (UNEF) and eight series with constant pitches, 4 UN, 6 UN, 8 UN, 12 UN, 16 UN, 20 UN, 28 UN and 32 UN.

6 Thread classes

6.1 Thread classes are distinguished from each other by the amounts of tolerance and allowance. The function of these classes is to provide for various grades of fit when threaded parts are assembled. Three classes of external threads (1A, 2A and 3A) and three classes of internal threads (1B, 2B and 3B) have been established for general purpose use.

Thread classes 1A and 1B are applicable for bolts and nuts where easy assembly is required or where rough handling and foreign material may restrict assembly of a closer tolerance class. This class provides a liberal tolerance and is applicable only to sizes 0,25 inch and over the UNC and UNF series.

Thread classes 2A and 2B are applicable to general usage, including production of bolts, screws, nuts and similar threaded fasteners. The maximum material diameters of class 2A (external) uncoated threads are less than basic by the amount of the allowance.

Customarily, for class 2A threads having an additive finish, the maximum allowable diameter is increased to the basic size, the value being the same as for class 3A. The allowance minimizes galling and seizing in high-cycle wrench assembly, or it can be used to accommodate plated finishes or other coatings.

Thread classes 3A and 3B are applicable where closeness of fit and accuracy of lead and angle of thread are important. They are obtainable consistently only by the use of high quality production equipment supported by a very efficient system of gauging and inspection. No allowance is provided.

6.2 Fits other than that obtained with class 2A with class 2B, for example, may be obtained by using class 2A with 1B or 3B, or class 2B with 1A or 3A.

7 Allowances and formulae

The allowance is applied negatively to the basic size to give a maximum material size below basic. An allowance is applied only to the classes 1A and 2A (external threads).

The allowance, in inches, for the thread classes 1A and 2A is calculated from the following formula:

$$0.3\left(0.001\ 5\ \sqrt[3]{D}\ + 0.001\ 5\sqrt{L_{\rm e}}\ + 0.015\ \sqrt[3]{P^2}\right)$$

Class 3A: zero allowance.

8 Tolerances and formulae

8.1 Major diameter tolerances

The major diameter tolerance T_d varies with the pitch and the thread series, but is independent of the length of engagement.

The tolerance for the major diameter for the thread classes indicated is calculated from the following.

a) External thread

$$T_{d}$$
, class 1A: 0,09 $\sqrt[3]{P^2}$

$$T_d$$
, classes 2A and 3A: 0,06 $\sqrt[3]{P^2}$

b) Internal thread

No tolerance is given for the major diameter of the nut thread. (See clause 13.)

8.2 Pitch diameter tolerances

The pitch diameter tolerances T_{d2} and T_{D2} vary with the diameter, pitch length of engagement and thread class

The tolerance for the pitch diameter for the thread classes indicated is calculated from the following.

a) External thread

 T_{d2} , class 2A:

$$0,001\ 5\sqrt[3]{D} + 0,001\ 5\sqrt{L_{\rm e}} + 0,015\ \sqrt[3]{P^2}$$

 T_{d2} , class 1A: 1,5 (tolerance of class 2A)

 T_{d2} , class 3A: 0,75 (tolerance of class 2A)

b) Internal thread

 T_{D2} class 1B: 1,95 (tolerance of class 2A)

 $T_{\rm D2}$ class 2B: 1,30 (tolerance of class 2A)

 T_{D2} class 3B: 0,975 (tolerance of class 2A)

8.3 Minor diameter tolerances

The minor diameter tolerance, T_{D1} , varies with diameter, pitch and thread class but is independent of the length of engagement. Modifications may be made for special applications.

The tolerance for the minor diameter for lengths of engagement up to 1,5D of the thread classes indicated is calculated from the following formulae.

a) External thread

No tolerance is given for the minor diameter of the bolt thread. For root radius control, see clause 13.

b) Internal thread

 T_{D1} , classes 1B and 2B for all sizes below 0,25 in: $\left(0.05 \sqrt[3]{P^2} + \frac{0.03}{D}P\right) - 0.002$

The resultant value should not exceed 0,394P, or be less than 0,25P – 0,4 P^2 .

 T_{D1} , classes 1B and 2B for all sizes 0,25 in and larger: 0,25P – 0,4 P^2

$$T_{D1}$$
, class 3B: $\left(0.05 \sqrt[3]{P^2} + \frac{0.03}{D}P\right) - 0.002$

The resultant value for class 3B should not exceed 0.394P, or be less than $0.23P-1.5P^2$ for 80 to 13 threads per inch. For 12 threads per inch and coarser, the tolerance shall not be less than 0.12P, which is, in effect, the tolerance for all sizes 1 in and larger having 12 threads per inch and coarser.

For applications having lengths of thread engagement less than 0,667D or more than 1,5D, the values should be modified in accordance with rules given in clause 11.

9 Accuracy

The final dimension is rounded off by standard conventional means, after actual calculation to the eighth decimal place, giving the final answer to the fourth place; for example, 0,003 291 81 is rounded off to 0,003 3 when the dimension is required to the fourth decimal place.

10 Design profiles (maximum material profile) and disposition of tolerances

- **10.1** The design profiles are the maximum material limit of the class 3A external and classes 1B, 2B and 3B internal threads and are shown in figure 1 (nut) and figure 2 (bolt).
- **10.2** For classes 1A and 2A, the maximum material limit differs from the design profile by the amount of the allowance.
- 10.3 Tolerances are applied to the maximum material limit to determine the minimum material limit.
- **10.4** The dispositions of tolerances, allowances and crest clearances are shown in figure 3 (class 1A, 2A, 1B and 2B) and figure 4 (classes 3A and 3B).

10.5 The root contours of the design profiles are designed to clear a crest width of 0,125*P* on the external threads and crest width of 0,250*P* on the internal thread (see clause 13).

11 Modified threads

Modification of minor diameter tolerances of the internal thread is permitted for special lengths of engagement.

Occasionally, there are applications where the length of engagement of the mating threads, or the combinations of materials used for mating threads, are such that the internal maximum minor diameter may not provide the desired strength of the threads. Experience has shown that for length of engagement less than 0,667D (the minimum thickness of standard nuts), the minor diameter tolerance can be reduced without causing tapping difficulties.

In other applications, the length of engagement of mating threads may exceed 1,5D because of design considerations or the combination of materials used for mating. As the threads engaged increase in number, their depth of engagement can be shallower and still develop stripping strength greater than the external thread breaking strength. In these cases, the internal thread minor diameter tolerance is increased. By working to the minimum permissible material limit, the possibility of tapping difficulties is reduced.

To reduce the number of minor diameter tolerances to a practical minimum, tolerances for a selection of recommended diameters, lengths of engagement and pitches are given in table 2 for thread classes 1B and 2B, and tables 3 and 4 for thread class 3B.

In these tables, the tolerances for lengths of engagement less than 0.333D are 0.5 of the formula values. For lengths of engagement from 0.333D to 0.667D, the tolerances are 0.75 of the formula values; for lengths of engagement from 0.667D to 1.5D the tolerances are equal to the formula values; and for lengths of engagement 1.5D to 3.0D, the tolerances are 1.25 times the formula values. Where the tolerance value so computed is more than 0.394P, which corresponds to a resulting minimum thread height of 53% of 0.75H, the value is adjusted to equal 0.394P(=0.455H). (See clause 8.)

Nut threads requiring modified minor diameters for lengths of thread engagement less than 0.667D to develop the optimum strength of the fastening, or longer than 1.5D to reduce tapping difficulties, should be designated MOD in the screw thread designation (See clause 14.)

12 Lengths of thread engagement

The pitch diameter tolerances for the UNC, UNF, 4 UN, 6 UN and 8 UN thread series are based on a length of engagement equal to the basic major (nom-

inal) diameter and are applicable for lengths of engagement up to 1,5 diameters.

The pitch diameter tolerances for the UNEF, 12 UN, 16 UN, 20 UN, 28 UN and 32 UN thread series are based on a length of engagement of 9 pitches and are applicable for lengths of engagement up to 15 pitches.

Where the length of engagement exceeds that for which these tolerances are applicable, the tolerances shall be calculated by the method shown in table 1.

Table 1 — Pitch diameter tolerances

	,		
Thread	_	of thread ement	
series	Above	Up to and including	Tolerance
UNC, UNF,		1,5 <i>D</i>	Formula value calcu- lated from 8.2
4 UN, 6 UN and	1,5 <i>D</i>	3 <i>D</i>	1,25 times the for- mula value
8 UN	3 <i>D</i>		1,5 times the for- mula value
UNEF, 12 UN,	_	15 <i>P</i>	Formula value calcu- lated from 8.2
16 UN, 20 UN, 28 UN	15 <i>P</i>	30 <i>P</i>	1,25 times the for- mula value
and 32 UN	30 <i>P</i>	_	1,5 times the for- mula value

13 Root radius control

13.1 General

Where there is no allowance, the design profile establishes the maximum material profiles of both the external and internal threads.

At the maximum material limits of the internal and external threads, the roots are shown as rounded contours approximating tool crest wear. These contours are sometimes represented by uniform radii tangent to the flanks of the thread. The maximum value of such a radius for the root contour of the internal thread is 0,072*P*. For the root contour of the external thread it is 0,144*P*.

The root contour for the internal thread is designed to clear the design profile of the external thread which is a flat of width 0,125P but, in practice, to avoid sharp corners, is normally rounded and cleared beyond the flat. Similarly, the maximum material profile of the root of external thread is designed to clear the design profile of the crest of the internal thread which is a flat of width 0,25P.

At the maximum and minimum material limits of the internal thread, the contour of the root is bounded by flats of 0,125*P* and 0,041 7*P* respectively. In practice, crest form of the threading tool is relied upon for compliance with these limits.

13.2 Internal thread root contour

The major (root) diameter of internal threads is considered satisfactory if it will accept external threads having a major diameter and a crest conforming to the design profile (maximum material profile).

13.3 External thread root contour

13.3.1 The minor (root) diameter of external threads is generally considered satisfactory if it will accept internal threads having a minor diameter and a crest conforming to the design profile. However, when a radius root is required, the contour may be checked for compliance with its minimum radius limit.

13.3.2 For applications requiring high strength and resistance to fatigue, it is recommended that the root of the external thread should be a smoothly rounded radius (a faired curved root) without reverse curvature or flats within the limits on minimum radius 0,108*P* and maximum radius 0,144*P*, the former being tangent to a root flat of 0,125*P* and tangent to the thread flank.

13.4 Specification of tolerances and symbols

13.4.1 In view of the above discussion, root diameter tolerances as such are not provided in this International Standard.

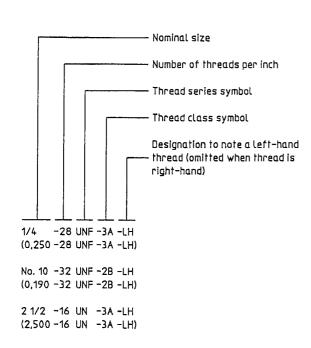
13.4.2 The designation for specifying the rounded root is at the discretion of the user; however, some countries have established the symbol UNR as the method of designating a thread with a mandatory rounded root.

14 Designation

A complete designation for a screw thread for standard length of engagement comprises the nominal size, threads per inch, and designations for the thread series and thread class. To this may be appended other supplementary symbols for threads with modified major (crest) diameter limits, long lengths of engagement, etc, when and as applicable.

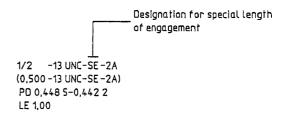
EXAMPLES

a) For standard lengths of engagement (see clause 12)



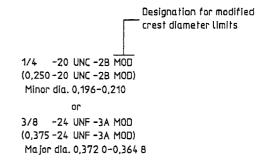
b) For special lengths of engagement (see clause 12)

(Actual pitch diameter and length of engagement are shown.)



c) For threads with modified crest diameter limits (see clause 11)

(Actual minor and major diameter limits are shown.)



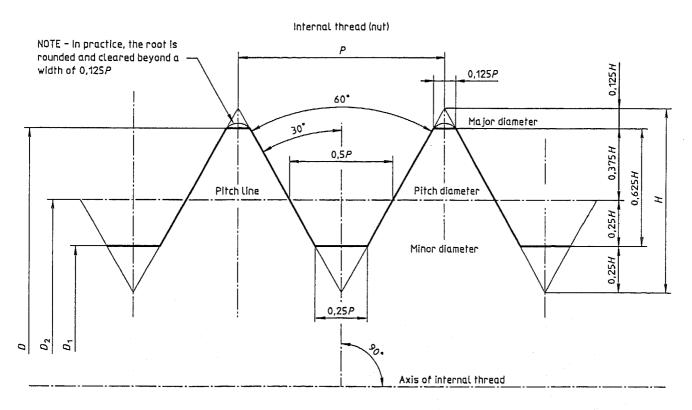


Figure 1 — Unified internal thread design profile (same as basic profile given in ISO 68) (maximum material condition MMC)

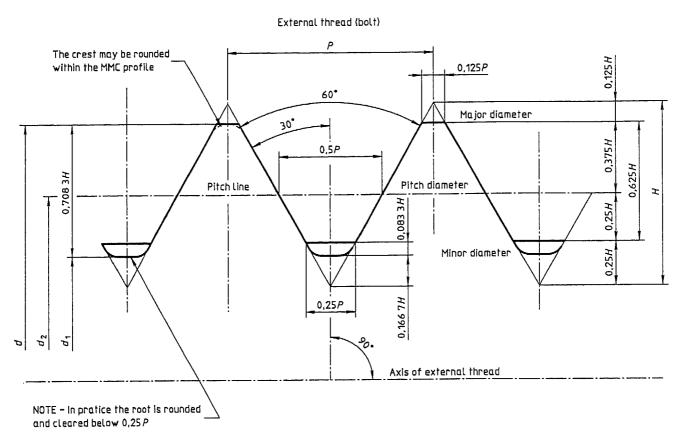


Figure 2 — Unified external thread design profile (maximum material condition MMC)

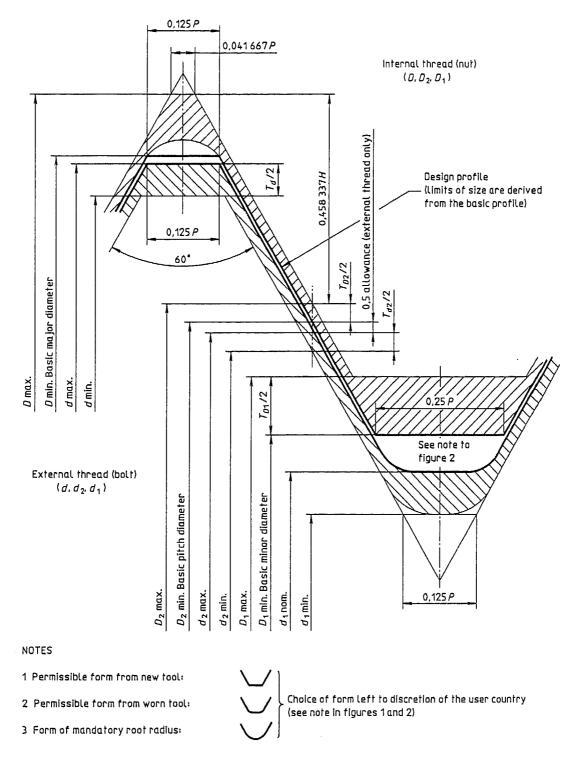


Figure 3 — Disposition of tolerances, allowances and crest clearances for classes 1A, 2A, 1B and 2B.

1

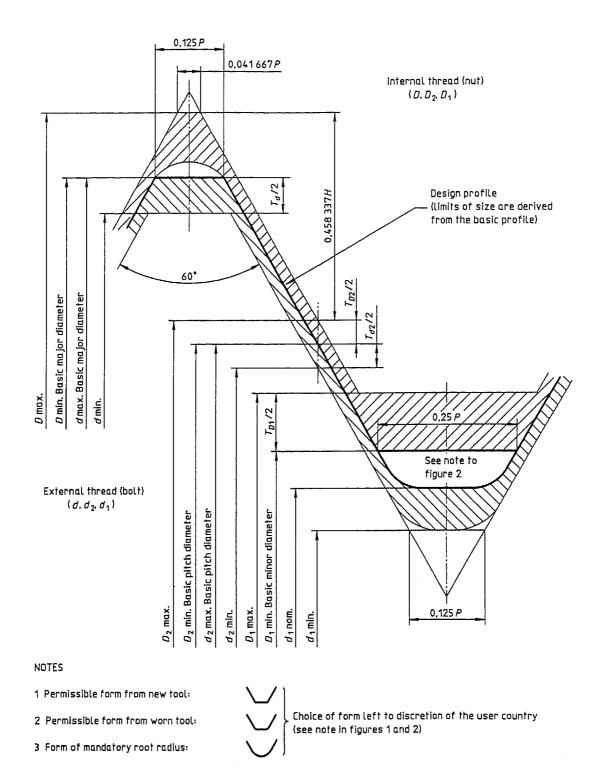


Figure 4 — Disposition of tolerances and crest clearances for classes 3A and 3B

Table 2 — Minor diameter tolerances for internal threads, T_{D1} , for special length of engagement: Classes 1B and 2B

			,								Toleran	ces in 0,	000 1 ir
Threads per inch	Length o	of thread ement					Thi	read siz	es.				,
			0	1	2	3	4	5	6	8	10	12	
n	above	up to and including	0,060	0,073	0,086	0,099	0,112	0,125	0,138	0,164	0,190	0,216	0,250 to 6
								T_{D1}					
		0,333 <i>D</i>	35	29	25	22	20	18	17	16	16	16	16
80	0,333 <i>D</i>	0,667 <i>D</i>	49	44	38	34	30	28	26	23	23	23	23
00	0,667 <i>D</i>	1,5 <i>D</i>	49	49	49	45	40	37	34	31	31	31	31
	1,5 <i>D</i>	3,0 <i>D</i>	49	49	49	49	49	46	43	39	39	39	39
		0,333 <i>D</i>	39	33	29	26	23	21	20	17	17	17	17
72	0,333 <i>D</i>	0,667 <i>D</i>	55	49	43	38	35	32	29	26	26	26	26
	0,667 <i>D</i>	1,5 <i>D</i>	55	55	55	51	46	42	39	34	34	34	34
	1,5 <i>D</i>	3,0 <i>D</i>	55	55	55	55	55	53	49	43	42	42	42
		0,333 <i>D</i>	45	38	33	29	27	24	23	20	19	19	19
64	0,333 <i>D</i>	0,667 <i>D</i>	62	57	49	44	40	37	34	20	28	28	28
	0,667 <i>D</i>	1,5 <i>D</i>	62	62	62	59	53	49	45	40	38	38	38
	1,5 <i>D</i>	3,0 <i>D</i>	62	62	62	62	62	61	-57	50	48	48	48
	_	0,333 <i>D</i>	-	44	38	34	31	29	26	23	22	22	22
56	0,333 <i>D</i>	0,667 <i>D</i>		66	57	51	46	43	40	35	32	32	32
	0,667 <i>D</i>	1,5 <i>D</i>		70	70	68	62	57	53	47	43	43	43
	1,5 <i>D</i>	3,0 <i>D</i>	_	70	70	70	70	70	66	59	54	54	54
	-	0,333 <i>D</i>			45	40	37	34	32	28	25	25	25
48	0,333 <i>D</i>	0,667 <i>D</i>	_		68	61	55	51	47	42	38	38	38
	0,667 <i>D</i>	1,5 <i>D</i>	-	,	82	81	74	68	63	56	51	50	50
	1,5 <i>D</i>	3,0 <i>D</i>			82	82	82	82	79	70	63	62	62
	*****	0,333 <i>D</i>			50	44	40	38	35	31	28	28	28
44	0,333 <i>D</i>	0,667 <i>D</i>	-		75	67	61	56	52	46	42	41	41
	0,667 <i>D</i>	1,5 <i>D</i>	-		90	89	81	75	50	62	56	55	55
	1,5 <i>D</i>	3,0 <i>D</i>	_	_	89	89	89	89	88	78	70	69	69
		0,333 <i>D</i>	-	_		49	45	41	39	34	31	30	30
40	0,333 <i>D</i>	0,667 <i>D</i>				74	67	62	58	51	47	45	45
	0,667 <i>D</i>	1,5 <i>D</i>	-	_		98	90	83	77	68	62	60	60
	1,5 <i>D</i>	3,0 <i>D</i>	_			98	98	98	96	86	78	75	75
		0,333 <i>D</i>					50	46	43	38	35	33	33
36	0,333 <i>D</i>	0,667 <i>D</i>	_		_	—	75	69	65	58	52	50	50
"	0,667 <i>D</i>	1,5 <i>D</i>	-	-			100	93	86	80	70	66	66
	1,5 <i>D</i>	3,0 <i>D</i>	_		_		109	109	108	96	87	82	82

Threads per inch		of thread ement					Thi	read siz	:es				
n	above	up to and including	0 0,060	1 0,073	2 0,086	3 0,099	4 0,112	5 0,125	6 0,138	8 0,164	10 0,190	12 0,216	0,250 to 6
								T_{D1}					
32	 0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i>	0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i> 3,0 <i>D</i>		1 1 1 1	-				49 73 98 122	43 65 87 108	39 59 79 99	37 79 74 92	37 75 74 92
28	— 0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i>	0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i> 3,0 <i>D</i>		1 1 1 1		1 1 1 1		-	— — —		45 68 91 113	42 63 90 105	42 63 84 105
24	— 0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i>	0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i> 3,0 <i>D</i>					<u> </u>	-			53 79 106 132	49 73 98 122	48 73 97 121
20		0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i> 3,0 <i>D</i>			 		— — —	<u>-</u>			1 1 1 1		53 86 115 144
18	 0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i>	0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i> 3,0 <i>D</i>	_ _ _	-	_		- I - I - I				 	_	64 95 127 159
16	 0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i>	0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i> 3,0 <i>D</i>					<u>-</u>	 			_ _ _	 	70 106 141 176
14	— 0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i>	0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i> 3,0 <i>D</i>	_ _ _	_ _ _	- - -		_ _ _		_	_ _ _	 		79 118 158 198
13	0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i>	0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i> 3,0 <i>D</i>	 			— — —	 		<u>-</u>		<u>-</u>		85 128 170 213

Threads per inch		of thread gement					Th	read siz	zes				
n	above	up to and including	0 0,060	1 0,073	2 0,086	3 0,099	4 0,112	5 0,125	6 0,138	8 0,164	10 0,190	12 0,216	0,250 to 6
								T_{D1}					
12	— 0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i>	0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i> 3,0 <i>D</i>			_			_	_			_ _ _ _	90 135 181 226
11	— 0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i>	0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i> 3,0 <i>D</i>			_		_	_ _ _	_		_ _ _	_	97 146 194 242
10	— 0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i>	0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i> 3,0 <i>D</i>					_			_	<u>-</u> - -		105 158 210 262
9	 0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i>	0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i> 3,0 <i>D</i>										_ _ _ _	114 171 228 286
8	 0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i>	0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i> 3,0 <i>D</i>						_ _ _ _			_		125 188 250 312
7	— 0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i>	0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i> 3,0 <i>D</i>	_ _ _			_				_ _ _ _			138 207 276 344
6	 0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i>	0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i> 3,0 <i>D</i>	 							 		_ _ _	153 230 306 382
5	 0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i>	0,333 <i>D</i> 0,667 <i>D</i> 1,5 <i>D</i> 3,0 <i>D</i>			_		_ _ _ _				_ _ _		170 255 340 425

Threads per inch	Length o						Thi	ead siz	es.				
111011			0	1	2	3	4	5	6	8	10	12	
n	above	up to and including	0,060	0,073	0,086	0,099	0,112	0,125	0,138	0,164	0,190	0,216	0,250 to 6
				<u></u>				T_{D1}					
		0,333 <i>D</i>			-		_		_				179
4.5	0,333 <i>D</i>	0,667 <i>D</i>			_	!			_		_	_	268
4,5	0,667 <i>D</i>	1,5 <i>D</i>		_	-	_	-	_	_				358
	1,5 <i>D</i>	3,0 <i>D</i>	_			_	_	_	_	_		+	448
		0,333 <i>D</i>			_	_		_	_				183
	0,333 <i>D</i>	0,667 <i>D</i>	_	-		_		_	_	_	_	_	281
4	0,667 <i>D</i>	1,5 <i>D</i>			_	_	_	_	_				375
	1,5 <i>D</i>	3,0 <i>D</i>	-			_	—	_	_	-			469

NOTE — If the minor diameter tolerance provided by this table is less than the pitch diameter tolerance, use the latter.

Table 3 — Minor diameter tolerance for internal threads, T_{D1} , for special length of engagement: Class 3B — Sizes 0 to 12

Tolerances in 0,000 1 in

Threads	Length of threa	nd engagement					Thread	l sizes				
per inch	above	up to and	0 0,060	1 0,073	2 0,086	3 0,099	4 0,112	5 0,125	6 0,138	8 0,164	10 0,190	12 0,216
n	above	including					T_{L})1			•	
		0,333 <i>D</i>	35	29	25	22	20	18	17	15	13	13
80	0,333 <i>D</i>	0,667 <i>D</i>	49	44	38	34	30	28	26	22	20	20
80	0,667 <i>D</i>	1,5 <i>D</i>	49	49	49	45	40	37	34	30	27	26
:	1,5 <i>D</i>	3,0 <i>D</i>	49	49	49	49	49	46	43	37	33	33
		0,333 <i>D</i>	39	33	29	26	23	21	20	17	15	15
70	0,333 <i>D</i>	0,667 <i>D</i>	55	49	43	38	35	32	29	26	23	22
72	0,667 <i>D</i>	1,5 <i>D</i>	55	55	55	51	46	42	39	34	31	29
	1,5 <i>D</i>	3,0 <i>D</i>	55	55	55	55	55	53	49	43	39	36
	_	0,333 <i>D</i>	45	38	33	29	27	24	23	20	18	16
64	0,333 <i>D</i>	0,667 <i>D</i>	62	57	49	44	40	37	34	30	27	25
04	0,667 <i>D</i>	1,5 <i>D</i>	62	62	62	59	53	49	45	40	36	33
	1,5 <i>D</i>	3,0 <i>D</i>	62	62	62	62	62	61	57	50	45	41
		0,333 <i>D</i>	_	44	38	34	31	29	26	23	21	19
Ec	0,333 <i>D</i>	· 0,667 <i>D</i>	-	66	57	51	46	43	40	35	32	29
56	0,667 <i>D</i>	1,5 <i>D</i>	_	70	70	68	62	57	53	47	42	39
	1,5 <i>D</i>	3,0 <i>D</i>	-	70	70	70	70	70	66	59	53	49

Threads per inch	Length of thre	ead engagement					Thread	sizes				
n	above	up to and including	0 0,060	1 0,073	2 0,086	3 0,099	4 0,112	5 0,125	6 0,138	8 0,164	10 0,190	12 0,216
							T_L)1				
		0,333 <i>D</i>	_	_	45	40	37	34	32	28	25	23
48	0,333 <i>D</i>	0,667 <i>D</i>	_	_	68	61	55	51	47	42	38	35
	0,667 <i>D</i>	1,5 <i>D</i>	-		82	81	74	68	63	56	51	47
	1,5 <i>D</i>	3,0 <i>D</i>			82	82	82	82	79	70	63	59
		0,333 <i>D</i>	_		50	44	40	37	35	31	28	26
44	0,333 <i>D</i>	0,667 <i>D</i>	_		75	67	61	56	52	46	42	39
- 1	0,667 <i>D</i>	1,5 <i>D</i>	-	_	90	89	81	75	70	62	56	52
	1,5 <i>D</i>	3,0 <i>D</i>	-		90	90	90	90	87	77	70	65
	*****	0,333 <i>D</i>	_	_		49	45	41	39	34	31	29
40	0,333 <i>D</i>	0,667 <i>D</i>	-	-		74	67	62	58	51	47	43
	0,667 <i>D</i>	1,5 <i>D</i>	-		-	98	90	83	76	68	62	57
	1,5 <i>D</i>	3,0 <i>D</i>	-	_	-	98	98	98	96	86	78	72
		0,333 <i>D</i>	_	_		_	50	46	43	38	35	32
36	0,333 <i>D</i>	0,667 <i>D</i>	-	-	_	-	75	69	65	58	52	43
30	0,667 <i>D</i>	1,5 <i>D</i>	-	-	-	_	100	93	86	76	70	64
	1,5 <i>D</i>	3,0 <i>D</i>	-	-		-	109	109	108	96	87	81
		0,333 <i>D</i>		_		_			49	43	39	36
32	0,333 <i>D</i>	0,667 <i>D</i>	-				_		73	65	59	55
-	0,667 <i>D</i>	1,5 <i>D</i>	_	-		-	-		98	87	79	73
	1,5 <i>D</i>	3,0 <i>D</i>	-		-		_	-	122	108	99	91
		0,333 <i>D</i>	-				_	_		_	45	42
28	0,333 <i>D</i>	0,667 <i>D</i>		-	-	-	-		_		68	63
~~	0,667 <i>D</i>	1,5 <i>D</i>	-	-	-	-	-	_	-		91	84
	1,5 <i>D</i>	3,0 <i>D</i>		-	-	-	-	-	-		113	105
		0,333 <i>D</i>	_		_		_	_		-	53	49
24	0,333 <i>D</i>	0,667 <i>D</i>		-	_		-		-	-	79	73
-7	0,667 <i>D</i>	1,5 <i>D</i>	-	-	-		_		-	_	106	98
	1,5 <i>D</i>	3,0 <i>D</i>	-		-	-	-		-	-	132	122

NOTE — If the minor diameter tolerance as provided by this table is less than the pitch diameter tolerance, use the latter.

Table 4 — Minor diameter tolerance for internal thread, T_{D1} , for special length of engagement: Class 3B — Sizes 0,250 to 6,000

Tolerances in 0.000 1 in

	Length o	of thread											TOICEAN	ces in 0,0	
Threads per inch		ement						16	read siz	es		1		r	
n	above	up to and	0,250	0,312 5	0,375	0,437 5	0,500	0,562 5	0,625	0,687 5	0,750	0,812 5	0,875	0,937 5	1,0 to (
		including							TDI						
		0,333 <i>D</i>	13	13			-	_	_		-	_	*****		
80	0,333 <i>D</i>	0,667 <i>D</i>	20	20					_		_	-	-	-	_
80	0,667 <i>D</i>	1,500 <i>D</i>	26	26		_		-		-		-	_	-	_
	1,500 <i>D</i>	3,000 <i>D</i>	33	33	_	-	1	_	_	_		_	-	_	
	-	0,333 <i>D</i>	15	15	15	15		_	_	_				-	_
72	0,333 <i>D</i>	0,667 <i>D</i>	22	22	22	22	_	-	_	_	_	-			
-	0,667 <i>D</i>	1,500 <i>D</i>	29	29	29	29	_		_	-		-	_	_	_
	1,500 <i>D</i>	3,000 <i>D</i>	36	36	36	36		_							
	-	0,333 <i>D</i>	16	16	16	16	16	16	_	_	_	-	-	-	_
64	0,333 <i>D</i>	0,667 <i>D</i>	24	24	24	24	24	24	_	_	-	-		-	
	0,667 <i>D</i>	1,500 <i>D</i>	32	32	32	32	32	32		_	_	-	_	-	_
	1,500 <i>D</i>	3,000 <i>D</i>	40	40	40	40	40	40							
	_	0,333 <i>D</i>	18	18	18	18	18	18	18	18	18	18	18	_	
56	0,333 <i>D</i>	0,667 <i>D</i>	27	27	27	27	27	27	27	27	27	27	27		
••	0,667 <i>D</i>	1,500 <i>D</i>	36	36	36	36	36	36	36	36	36	36	36	-	-
	1,500 <i>D</i>	3,000 <i>D</i>	45	45	45	45	45	45	45	45	45	45	45		
	_	0,333 <i>D</i>	21	21	21	21	21	21	21	21	21	21	21	-	-
48	0,333 <i>D</i>	0,667 <i>D</i>	32	31	31	31	31	31	31	31	31	31	31	_	-
,,	0,667 <i>D</i>	1,500 <i>D</i>	43	41	41	41	41	41	41	41	41	41	41	-	-
	1,500 <i>D</i>	3,000 <i>D</i>	54	52	52	52	52	52	52	52	52	52	52		
		0,333 <i>D</i>	24	22	22	22	22	22	22	22	22	22	22	_	
44	0,333 <i>D</i>	0,667 <i>D</i>	36	33	33	33	33	33	33	33	33	33	33	_	
	0, 6 67 <i>D</i>	1,500 <i>D</i>	47	45	45	45	45	45	45	45	45	45	45		_
	1,500 <i>D</i>	3,000 <i>D</i>	59	56	56	56	56	56	56	56	56	56	56	56	_
	_	0,333 <i>D</i>	26	24	24	24	24	24	24	24	24	24	24	24	24
40	0,333 <i>D</i>	0,667 <i>D</i>	40	36	36	36	36	36	36	36	36	36	36	36	36
	0,667 <i>D</i>	1,500 <i>D</i>	58	48	48	48	48	48	48	48	48	48	48	48	48
	1,500 <i>D</i>	3,000 <i>D</i>	66	62	62	60	60	60	60	60	60	60	60	60	60
	_	0,333 <i>D</i>	30	26	26	26	26	26	26	26	26	26	26	26	20
36	0,333 <i>D</i>	0,667 <i>D</i>	44	39	39	39	39	39	39	39	39	39	39	39	3
	0,667 <i>D</i>	1,500 <i>D</i>	59	53	52	52	52	52	52	52	52	52	52	52	5
	1,500 <i>D</i>	3,000 <i>D</i>	74	66	65	65	65	65	65	65	65	65	65	65	65
	_	0,333 <i>D</i>	34	30	29	29	29	29	29	29	29	29	29	29	29
32	0,333 <i>D</i>	0,667 <i>D</i>	50	45	43	43	43	43	43	43	43	43	43	43	4
	0,667 <i>D</i>	1,500 <i>D</i>	67	60	57	57	57	57	57	57	57	57	57	57	5
	1,500 <i>D</i>	3,000 <i>D</i>	84	75	72	72	72	72	72	72	72	72	72	72	7:
	_	0,333 <i>D</i>	39	34	32	32	32	32	32	32	32	32	32	32	3:
28	0,333 <i>D</i>	0,667 <i>D</i>	58	51	47	47	47	47	47	47	47	47	47	47	4
	0,667 <i>D</i>	1,500 <i>D</i>	77	69	63	63	63	63.	63	63	63	63	63	63	63
	1,500 <i>D</i>	3,000 <i>D</i>	96	86	79	79	79	79	79	79	79	79	79	79	79

24	0,333 <i>D</i> 0,667 <i>D</i> 1,500 <i>D</i> 0,333 <i>D</i> 0,667 <i>D</i>	up to and including 0,333 <i>D</i> 0,667 <i>D</i> 1,500 <i>D</i> 3,000 <i>D</i> 0,333 <i>D</i>	0,250 45 68 90 113	0,312 5 40 60 80	0,375 37	0,437 5	0,500	0,562 5													
	0,667 <i>D</i> 1,500 <i>D</i> — 0,333 <i>D</i>	0,667 <i>D</i> 1,500 <i>D</i> 3,000 <i>D</i>	68 90	60	37			L			0,750	0,812 5	0,875	0,937 5	to 6						
	0,667 <i>D</i> 1,500 <i>D</i> — 0,333 <i>D</i>	0,667 <i>D</i> 1,500 <i>D</i> 3,000 <i>D</i>	68 90	60	37			1	T_{D1}			<u>'</u>	<u> </u>	<u> </u>	<u> </u>						
20			113		55 73	35 52 70	35 52 70	35 52 70	35 52 70												
20		0,333 <i>D</i>	ı	100	92	87	87	87	87	87	87	87	87	87	87						
[1,500 <i>D</i>	0,687 <i>D</i> 1,500 <i>D</i> 3,000 <i>D</i>	54 81 108 135	48 72 96 120	44 66 88 110	41 62 82 103	39 58 78 97	39 58 78 97	39 58 78 97	39 58 78 97	39 58 78 97	39 58 78 97	39 58 78 97	39 58 78 97	39 58 78 97						
18		0,333 <i>D</i> 0,667 <i>D</i> 1,500 <i>D</i> 3,000 <i>D</i>	- -	53 80 106 133	49 73 97 122	45 68 91 114	43 65 86 108	41 62 82 103	41 61 81 102	41 61 81 102	41 61 81 102	41 61 81 102	41 61 81 102	41 61 81 102	41 61 81 102						
16	 0,333 <i>D</i> 0,667 <i>D</i> 1,500 <i>D</i>	0,333 <i>D</i> 0,667 <i>D</i> 1,500 <i>D</i> 3,000 <i>D</i>		_ _ _	54 82 109 136	51 76 102 127	48 72 96 120	46 69 92 115	44 67 89 111	43 64 86 108	43 64 85 106	43 64 85 106	43 64 85 106	43 64 85 106	43 64 85 106						
14		0,333 <i>D</i> 0,667 <i>D</i> 1,500 <i>D</i> 3,000 <i>D</i>	<u>-</u> - -	-		58 86 115 144	54 82 109 136	52 78 104 130	50 75 100 125	49 73 97 122	47 71 95 118	46 69 92 116	45 68 91 113	44 67 89 111	44 66 88 110						
13	 0,333 <i>D</i> 0,667 <i>D</i> 1,500 <i>D</i>	0,333 <i>D</i> 0,667 <i>D</i> 1,500 <i>D</i> 3,000 <i>D</i>	_	_ _ _ _			58 87 117 146	56 83 111 139	54 80 107 134	52 78 104 130	50 76 101 126	50 74 99 124	49 73 97 122	48 71 95 119	47 70 94 118						
12		0,333 <i>D</i> 0,667 <i>D</i> 1,500 <i>D</i> 3,000 <i>D</i>	_ _ _ _	- - -	_ _ _ _	<u>-</u>	63 94 125 157	60 90 120 150	58 87 115 144	56 84 112 140	54 82 109 136	53 80 106 133	52 78 104 130	51 77 102 128	50 75 100 125						
11		0,333 <i>D</i> 0,667 <i>D</i> 1,500 <i>D</i> 3,000 <i>D</i>		_	_				62 94 125 156	60 91 121 151	58 88 117 146	58 86 115 144	56 84 112 140	55 82 110 138	54 82 109 136						
10		0,333 <i>D</i> 0,667 <i>D</i> 1,500 <i>D</i> 3,000 <i>D</i>	_			- - -			_	66 99 131 164	64 96 128 160	62 93 125 156	61 92 122 153	60 90 120 150	60 90 120 150						
9		0,333 <i>D</i> 0,667 <i>D</i> 1,500 <i>D</i> 3,000 <i>D</i>				- - - -		 	_			68 103 137 171	67 100 134 168	66 100 133 166	66 100 133 166						
8	0,333 <i>D</i> 0,667 <i>D</i> 1,500 <i>D</i>	0,333 <i>D</i> 0,667 <i>D</i> 1,500 <i>D</i> 3,000 <i>D</i>	=	-		_				<u>-</u>	_	75 112 150 188	75 112 150 188	75 112 150 188	75 112 150 188						

Threads per inch		of thread ement						Th	read size	B\$					
п	above	up to and including	0,250	0,312 5	0,375	0,437 5	0,500	0,562 5	0,625	0,687 5	0,750	0,812 5	0,875	0,937 5	1,0 to 6
		including				1			T _{D1}						
	_	0,333 <i>D</i>	-	-	-	-	_	-	_	-	_		_	86	86
7	0,333 <i>D</i>	0,667 <i>D</i>	-			-	_		_				_	129	129
′	0,667 <i>D</i>	1,500 <i>D</i>	_	[-	-		_				171	171
	1,500 <i>D</i>	3,000 <i>D</i>	_	_	_	_	_	-	_	-	-		-	214	214
		0,333 <i>D</i>	_	_	_	_	_	_		-	_	_	-		100
	0,333 <i>D</i>	0,667 <i>D</i>	-	_	_	_		_	_				_	-	150
6	0,667 <i>D</i>	1,500 <i>D</i>		_	_	-	-	_	_	-	_			-	200
	1,500 <i>D</i>	3,000 <i>D</i>	_	_		_	_	_		-	_	_	_	_	250
		0,333 <i>D</i>	_	_	_			_	_				_		120
5	0,333 <i>D</i>	0,667 <i>D</i>		_		_			_	-	-	-			180
5	0,667 <i>D</i>	1,500 <i>D</i>	l —	-	_	_			_	-	_		_	-	240
	1,500 <i>D</i>	3,000 <i>D</i>	_	-	_	_					_	_		_	300
	_	0,333 <i>D</i>	_	_			_					_	_	_	133
4,5	0,333 <i>D</i>	0,667 <i>D</i>	_	-			_		_	-		_		-	200
4,5	0,667 <i>D</i>	1,500 <i>D</i>	-	-		-	-	_	_	_		_	_	-	267
	1,500 <i>D</i>	3,000 <i>D</i>	_	_	_	_		_	_		_	_	_	-	333
<u> </u>		0,333 <i>D</i>	_	_		_	_		_						150
4	0,333 <i>D</i>	0,667 <i>D</i>	_	-		_	_		_	-					225
•	0,667 <i>D</i>	1,500 <i>D</i>	_	_	_	_		_	_	-	_		_		300
	1,500 <i>D</i>	3,000 <i>D</i>	_	-	_	_	_	_		_	_	_	_	-	375

NOTE — If the minor diameter tolerance as provided by this table is less than the pitch diameter tolerance, use the latter.