## ASME B29.10M-1997 (Revision of ASME B29.10M-1994)

# HEAVY DUTY OFFSET SIDEBAR POWER TRANSMISSION Roller Chains and Sprocket teeth

AN AMERICAN NATIONAL STANDARD



The American Society of Mechanical Engineers



#### Α Ν Ε R С A N Α Μ Ν Α Т 0 Ν Α L S Т Α Ν D Α R D

## HEAVY DUTY OFFSET SIDEBAR POWER TRANSMISSION Roller Chains and Sprocket teeth

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#### FOREWORD

(This Foreword is not part of ASME B29.10M-1997.)

Chains of the type covered by this Standard were introduced in the United States late in the 19th Century. As their popularity increased, the number of chains and manufacturers grew. As one manufacturer developed a successful size chain, or family of chains, others soon duplicated it and a large number of chains came into being.

In spite of efforts at standardization, the Corps of Engineers, United States Army, found during the Korean incident of 1950-53 that lack of adequate standardization of power transmission chains still resulted in the incapacitation of many cranes and shovels. For this reason, the Corps of Engineers requested the Society of Automotive Engineers to expand the scope of ASA Sectional Committee B29 to cover this group of chains. On May 13, 1956, the Corps of Engineers request was approved by the ASA B29 Committee and the ASA B29.10 Subcommittee was formed. Upon approval by the American Standards Association in April 1962; the first standard for ASA B29.10 chains and sprockets was published. The Standard described the physical dimensions of the chain components and sprockets, and defined the minimum static properties for the chains.

In 1970, this Standard was revised to include Drive Selection information in a Supplementary Section and to convert fractions to decimal-inch, and include metric dimensions in SI Units.

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In September 1962, the Engineering Steel Chain Division of the ACA began a research program to develop ratings for the ANSI B29.10 chains. Capacity information on the smaller B29.1 chains already developed in research programs sponsored by the Roller Chain Division of the ACA was made available to the Engineering Steel Chain Division. Member companies contributed information from their own research programs. Special dynamic test equipment was built to enable obtaining wear and capacity data on very large chains. The horsepower capacities published as supplementary information in this Standard resulted from this combined research effort; they are considerably higher than previously accepted by the industry. At all speeds and for all sprocket sizes, the capabilities of ANSI B29.10 chain drives exceeded those previously accepted by the industry.

This Standard was approved as an American National Standard on December 27, 1972. The 1981 revision included updating to the current ANSI standards format covering chains and sprockets. In the process of updating, the supplemental information (Appendices A and B) was reviewed and found to be inadequate with respect to selection information. The information was incomplete and could result in an unsatisfactory selection. Chain standards generally contained only dimensional and strength information to allow chains to be intercoupled. Therefore, the horsepower capacity information was deleted.

The American National Standards Institute approved that revision on April 3, 1981.

The 1991 revision added definitions for minimum ultimate strength (para. 2.2), measuring load (para. 2.3), and strand length tolerance (para. 2.4). M.U.T.S. values for chains 4020, 4824, and 5628 were increased. Other clarifications were made without changing the basic content.

The 1994 revision was approved by the American National Standards Institute on November 17, 1994.

This 1997 revision of ASME B29.10M was approved by the American National Standards Institute on July 22, 1997.

#### ASME COMMITTEE B29 Chains, Attachments, and Sprockets for Power Transmission and Conveying

(The following is the roster of the Committee at the time of approval of this Standard.)

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#### HEAVY DUTY OFFSET SIDEBAR POWER TRANSMISSION ROLLER CHAINS AND SPROCKET TEETH

#### **1 NOMENCLATURE**

A series of identical offset links in which the pins articulate inside the bushings and the rollers are free to turn on the bushings. Pins and bushings are fixed in their respective sidebar holes [Fig. 1, sketches (a) and (b)].

In addition to press fits, other types of locks such as flats are sometimes used to prevent rotation of pins and bushings in their respective sidebar holes.

#### **2 GENERAL CHAIN DESIGNATORS**

#### 2.1 Dimensions for Chain Links

To assure interchangeability of links as produced by different makers of chain, standard maximum and minimum dimensions are adopted. They are not actual dimensions used in manufacturing, but limiting dimensions, maximum or minimum, required to assure the desired interchangeability.

All dimensions are given in a decimal-inch system. The metric equivalent dimensions are for reference only.

#### 2.2 Numbering System

The standard chain numbers were based upon the inch system. The first two, or left hand digits in the chain designation denote the number of  $\frac{1}{8}$  in. in the pitch. The last two, or right hand digits, denote the number of  $\frac{1}{16}$  in. in the pin diameter.

### 2.3 Minimum Ultimate Tensile Strength (M.U.T.S.)

Minimum Ultimate Tensile Strength (M.U.T.S.), for chain covered by this Standard, is the minimum force at which an unused, undamaged chain could fail when subjected to a single tensile loading test.

#### WARNING:

(a) The minimum ultimate tensile strength is not a "working load." The M.U.T.S. greatly exceeds the maximum force that may be safely applied to the chain.

(b) Test Procedure: A tensile force is slowly applied, in uniaxial direction, to the ends of the chain sample.





#### FIG. 1 HEAVY DUTY OFFSET SIDEBAR POWER TRANSMISSION ROLLER CHAINS

(c) The tensile test is a destructive test. Even though the chain may not visibly fail when subjected to the "Minimum Ultimate Tensile Force," it will have been damaged and will be unfit for service.

CAUTION: This load is beyond the yield strength of the chain and would render the chain unsuitable for application. For application guidance, consult manufacturers' catalogs or the American Chain Association Handbook, "Engineering Steel Chains for Conveyors, Elevators and Drives."

#### 2.4 Measuring Load

The measuring load, in pounds (or kilonewtons) in Table 1, is the load under which a dry or lightly lubricated chain should be measured for length.

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#### 2.5 Strand Length Tolerance

The length of new chains subject to the specified measuring load must fall within the plus and minus tolerances. Specific maximum and minimum strand lengths are shown in Table 1, for each chain.

#### **3 CHAIN ASSEMBLY**

See Fig. 2 and Tables 1, 2, and 3.

#### **4 SPROCKET TOOTH FORM**

See Fig. 3.

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- B = inside diameter of bushing
- D = pin diameter
- F = overall chain height
- = roller diameter Н
- = pin head to centerline J
- Κ = pin end to centerline
- Ρ = assembled chain pitch (this is a theoretical reference dimension used for basic calculations)
- T = sidebar thickness
- V = sidebar end clearance radius, open end of link
- V, = sidebar end clearance radius, closed end of link
- W = inside width
- X = width of link at closed end extending through end clearance zone "Y"
- Y = sidebar end clearance zone, closed end of link
- $Y_a$  = sidebar end clearance zone, open end of link Z = width between sidebars at open end of link extending through sidebar end clearance zone " $Y_a$ "

#### FIG. 2 CHAIN ASSEMBLY

			Dimens	ions, in.				
Chain No.	2010	2512	2814	3315	3618	4020	4824	5628
Р	2.500	3.067	3.500	4.073	4.500	5.000	6.000	7.000
D	0.625	0.750	0.875	0.938	1.100	1.250	1.500	1.750
т	0.31	0.38	0.50	0.56	0.56	0.62	0.75	0.88
F	1.75	2.25	2.25	2.38	3.00	3.50	4.00	5.00
Ĥ	1 25	1.62	1 75	1 78	2 25	2 50	3.00	3.50
Ŵ	1.50	1.56	1.50	1.94	2.06	2.75	3.00	3.25
M.U.T.S., Ib (See para. 2.2) [Note (1)]	57,000	77,000	106,000	124,000	171,000	222,000	315,000	425,000
No. of Chain Pitches in Standard Measuring Length	48	39	34	30	27	24	20	17
Standard Measuring Length, in.	100.00		110.00	100.57	101.00	100.00	100.00	110.20
iviax.	120.38	119.99	119.38	122.57	121.88	120.38	120.38	119.38
Min.	120.00	119.61	119.00	122.19	121.50	120.00	120.00	119.00
Measuring								
Load, Ib	400	600	800	900	1300	1700	2200	3300
			Dimensi	ons, mm				
Р	63.50	77.90	88.00	103.45	114.30	127.00	152.40	177.80
D	15.88	19.05	22.22	23.83	27.94	31.75	38.10	44.45
T	7.9	9.7	12.7	14.2	14.2	15.7	19.1	22.4
F	44.5	57.2	57.2	60.5	76.2	88.9	101.6	127.0
н	31.8	41 1	44.5	45.2	57.2	63.5	76.2	88.9
Ŵ	38.1	39.6	38.1	49.3	52.3	69.9	76.2	82.6
M.U.T.S., kN (See para. 2.2) [Note (1)]	254	342	471	552	761	987	1401	1890
No. of Chain Pitches in Standard Measuring Length	48	39	34	30	27	24	20	17
Standard Measuring Length, mm								
Max.	3057.7	3047.7	3032.3	3113.3	3095.8	3057.7	3057.7	3032.3
Min.	3048.0	3038.1	3022.6	3103.6	3086.1	3048.0	3048.0	3022.6
Measuring Load, kN	1.8	2.7	3.6	4.0	5.8	7.6	9.8	14.7

#### TABLE 1 GENERAL CHAIN DIMENSIONS, M.U.T.S., STRAND LENGTH, AND MEASURING LOAD

CAUTION: The numerical values set forth in this Table do not afford a sufficient or appropriate basis for determining chain application and must be read in conjunction with the definitions and explanatory notes on page 1.

NOTE:

(1) M.U.T.S. is a factored statistical value for standards, which does not necessarily reflect the typical ultimate strength of the chain. Manufacturer should be consulted for additional information.

#### HEAVY DUTY OFFSET SIDEBAR POWER TRANSMISSION ROLLER CHAINS AND SPROCKET TEETH

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Dimensions, in.									
Chain No.		2010	2512	2814	3315	3618	4020	4824	5628
Chain Pitch	( <i>P</i> )	2.500	3.067	3.500	4.073	4.500	5.000	6.000	7.000
Pin Diameter, Max.	(D)	0.626	0.751	0.876	0.939	1.101	1.251	1.501	1.751
Inside Diameter of Bushing, Min.	( <i>B</i> )	0.628	0.753	0.879	0.942	1.105	1.255	1.506	1.757
Roller Diameter, Max.	(H)	1.250	1.625	1.750	1.781	2.250	2.500	3.000	3.500
Sidebar End Clearance Zone Closed End of Link, Min. [Note (1)]	( <i>Y</i> )	0.88	1.06	1.25	1.31	1.56	1.88	2.19	2.56
Sidebar End Clearance Zone Open End of Link, Min. [Note (1)]	( <i>Y<sub>a</sub></i> )	0.94	1.16	1.31	1.38	1.62	2.06	2.31	2.68
Sidebar End Clearance Radius Open End of Link, Max. [Note (1)]	( <i>V</i> )	0.88	1.06	1.25	1.31	1.56	1.88	2.19	2.56
Sidebar End Clearance Radius Closed End of Link, Max. [Note (1)]	( <i>V</i> <sub>a</sub> )	0.94	1.16	1.31	1.38	1.62	2.06	2.31	2.68
Width of Link at Closed End, Extend- ing Through End Clearance Zone "Y," Max.	( <i>X</i> )	2.141	2.328	2.520	3.082	3.207	4.031	4.531	5.031
Width Between Sidebars at Open End of Link Extend- ing Through Sideba End Clearance Zone "Y <sub>a</sub> ," Min.	( <i>Z</i> )	2.146	2.333	2.525	3.087	3.212	4.036	4.536	5.036
Inside Width, Min.	(W)	1.44	1.50	1.44	1.86	1.98	2.64	2.88	3.12

#### TABLE 2 MAXIMUM AND MINIMUM CONTROLLING LINK DIMENSIONS FOR INTERCHANGEABLE CHAIN LINKS

(Table 2 continues on next page)

Dimensions, mm									
Chain No.		2010	2512	2814	3315	3618	4020	4824	5628
Chain Pitch	( <i>P</i> )	63.50	77.00	88.90	103.45	114.30	127.00	152.40	177.80
Pin Diameter, Max.	(D)	15.90	19.08	22.25	23.85	27.97	31.78	38.13	44.48
Inside Diameter of Bushing, Min.	(B)	15.95	19.13	22.33	23.93	28.07	31.88	38.25	44.63
Roller Diameter, Max.	( <i>H</i> )	31.75	41.28	44.45	45.24	57.15	63.50	76.20	88.90
Sidebar End Clearance Zone Closed End of Link, Min. [Note (1)]	( <i>Y</i> )	22.4	26.9	31.8	33.3	39.6	47.8	55.6	65.0
Sidebar End Clearance Zone Open End of Link, Min. [Note (1)]	(Y <sub>a</sub> )	23.9	29.5	33.3	35.1	41.1	52.3	58.7	68.1
Sidebar End Clearance Radius Open End of Link, Max. [Note (1)]	( <i>V</i> )	22.4	26.9	31.8	33.3	39.6	47.8	55.6	65.0
Sidebar End Clearance Radius Closed End of Link, Max. [Note (1)]	( <i>V</i> <sub>a</sub> )	23.9	29.5	33.3	35.1	41.1	52.3	58.7	68.1
Width of Link at Closed End, Extend ing Through End Clearance Zone "Y," Max.	(X)	54.38	59.13	64.01	78.28	81.46	102.39	115.09	127.79
Width Between Sidebars at Open End of Link Extend- ing Through Sideba End Clearance Zone "Y <sub>a</sub> ," Min.	( <i>Z</i> ) ir	54.51	59.26	64.14	78.41	81.58	102.51	115.21	127.91
Inside Width, Min.	(W)	36.6	38.1	36.6	47.2	50.3	67.1	73.2	79.2

#### TABLE 2 (CONT'D) MAXIMUM AND MINIMUM CONTROLLING LINK DIMENSIONS FOR INTERCHANGEABLE CHAIN LINKS

GENERAL NOTE: Offset bend lines may be straight or curved. If curved, their radii Y and  $Y_a$  must be greater than V and  $V_a$ , respectively, so that the mating ends of links built to maximum dimensions will allow flexing around a 7-tooth sprocket.

NOTE:

(1) Sidebar ends may be extended, provided the extension is within a 30 deg. included angle with respect to the end of sidebar as shown in Chain Assembly. Whether the sidebar ends are extended or not, the offset section of the bar must provide clearance for such an extension.

#### HEAVY DUTY OFFSET SIDEBAR POWER TRANSMISSION ROLLER CHAINS AND SPROCKET TEETH

Pin End to Center-

line, Max. Overall Chain

Height, Max.

(K)

(F)

47.8

47.8

55.6

60.5

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Dimensions, in.								
Chain No.	2010	2512	2814	3315	3618	4020	4824	5628
Pin Head to Center- (J) line, Max.	1.69	1.88	2.19	2.50	2.56	3.06	3.50	4.00
Pin End to Center- (K) line, Max.	1.88	2.19	2.44	2.81	3.00	3.56	3.88	4.50
Overall Chain (F) Height, Max.	1.88	2.38	2.38	2.50	3.12	3.62	4.12	5.25
			Dimension	is, mm				
Pin Head to Center- (J) line, Max.	42.9	47.8	55.6	63.5	65.0	77.7	88.9	101.6

62.0

60.5

71.4

63.5

76.2

79.2

90.4

91.9

98.6

104.6

114.3

133.4

#### TABLE 3 CHAIN CLEARANCE DIMENSIONS

#### 7

#### HEAVY DUTY OFFSET SIDEBAR POWER TRANSMISSION ROLLER CHAINS AND SPROCKET TEETH



The elements of a chain sprocket and the tooth form may be determined by the following:

- $C_b$  = undersize compensation (typically 0.06 in.)
- $C_c$  = chain clearance circle [Note (1)] = ( $P \times C_{cf}$ ) F
- $C_{cf}$  = chain clearance circle factor (see Table 5) = cot (180/N<sub>t</sub>) 0.05
- $C_p$  = pitch line clearance [Note (2)] =  $P \times 0.10$
- $D_b$  = bottom diameter [Note (3)] =  $D_r C_b$
- $D_{o}$  = outside diameter
- $D_{p}$  = pitch diameter =  $P \times D_{pf}$

 $D_{pf}$  = pitch diameter factor (see Table 5) = csc (180/N<sub>t</sub>)

- $D_r$  = maximum root diameter [Note (4)] = ( $P \times D_{pf}$ ) H max.
- F = maximum chain height (see Table 3)
- H = barrel height or roller diameter (see Table 1)
- $N_t$  = number of teeth
- $R_{\rho}$  = maximum pocket radius = H/2
- $R_t$  = topping radius = 0.5 × P
- $S_s$  = side slope = approximately  $0.12 \times W_t$ , not to exceed 0.38 in. (9.6 mm)
- W = inside chain width
- $W_t$  = working face [Note (5)] = 0.01 ×  $P \times N_t$
- $W_t$  = maximum tooth width = 0.95 W, min. of chain

 $\theta$  = pressure angle

GENERAL NOTE: Tooth working face length provides for approximately 6% chain pitch elongation for sprockets having less than 40 teeth. As the number of teeth increases, the possible pitch elongation is progressively reduced to less than 2% at 100 teeth.

#### NOTES:

- (1) No portion of hub, beads, lugs, or fillets shall extend beyond this circle in the sidebar zone.
- (2) If precision cut tooth sprockets are used in clean environment, pitch line clearance can be as small as  $P \times 0.003$ .
- (3) The bottom diameter shall be smaller than the root diameter, and the pocket radius should be smaller than H/2.
- (4) Root diameters and pocket radii must not exceed the maximum obtained from these formulas. Oversize dimensions cause improper chain and sprocket action and excessive chain loads.
- (5) Limitation on length of working face the working face shall not extend beyond the line through the adjacent pitch which is perpendicular to the working face.

#### FIG. 3 SPROCKET TOOTH FORM

in.

Up to 12

12 - 36

Over 36

**Pitch Diameter** 

mm

Up to 305

305 - 915

Over 915

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TABLE 4 ECCENTRICITY OUT TOLERAN			
Tolerances -	— Total Indic	ator Reading	
in.	····	mm	
0.06		1.52	
0.06 for each 12 i 0.18 plus 0.03 for each additional	n. 1.52 f 4.56 p 12 in. addi	or each 305 m blus 0.76 for ea tional 305 mm	m ach
procket tolerance Engineering Class nsmission chains nin the realm of n	s are closer t Chains. The and require formal sprocl	than called for se B29.10M cha sprockets of ket manufactur	ron ains the ring
TABLE 5 TOOTH FORM	I FACTOR	S	
θ, deg.	C <sub>cf</sub>	<u>N</u> t	
10	2.02	7	
11	2.36	8	
12	2.69	9	
13	3.02	10	
14	3.35	11	
15	3.68	12	
16	4.00	13	
17	4.33	14	
18	4.65	15	
19	4.97	16	
20	5.29	17	
20	5.62	18	
21	5.94	19	
21	6.26	20	
22	6.58	21	
22	6 69	22	
22	0.05	~~ 22	
22	7.22	23	
23	7.54	24	

#### MAXIMUM E **RUNO**

GENERAL NOTE: The above spre current B29 standards for other Er are high precision power trans highest precision possible within methods.

Nt	<b>D</b> <sub>pf</sub>	θ, deg.	<b>C</b> <sub>cf</sub>	Nt
7	2.304	10	2.02	7
8	2.613	11	2.36	8
9	2.923	12	2.69	9
10	3.236	13	3.02	10
11	3.549	14	3.35	11
12	3.863	15	3.68	12
13	4.178	16	4.00	13
14	4.494	17	4.33	14
15	4.809	18	4.65	15
16	5.125	19	4.97	16
17	5.442	20	5.29	17
18	5.758	20	5.62	18
19	6.075	21	5.94	19
20	6.392	21	6.26	20
21	6.709	22	6.58	21
22	7.026	22	6.69	22
23	7.343	22	7.22	23
24	7.661	23	7.54	24
25	7.978	23	7.86	25
26	8.296	23	8.18	26
27	8.613	23	8.50	27
28	8.931	24	8.82	28
2 <del>9</del>	9.249	24	9.14	29
30	9.566	24	9.46	30
31	9.884	24	9.78	31
32	10.202	24	10.10	32
33	10.520	25	10.42	33
34	10.837	25	10.74	34
35	11.155	25	11.07	35
36	11.473	25	11.38	36

SPROCKET T

#### AMERICAN NATIONAL STANDARDS — CHAINS, ATTACHMENTS, AND SPROCKETS FOR POWER TRANSMISSION AND CONVEYING

Precision Power Transmission Roller Chains, Attachments, and Sprockets	B29.1M-1993
Inverted Tooth (Silent) Chains and Sprockets	B29.2M-1982(R1994)
Double-Pitch Power Transmission Roller Chains and Sprockets	B29.3M-1994
Double-Pitch Conveyor Roller Chains, Attachments, and Sprockets	B29.4M-1994
Steel Detachable Link Chains, Attachments, and Sprockets	B29.6M-1993
Leaf Chains, Clevises, and Sheaves	B29.8M-1993
Heavy Duty Offset Sidebar Power Transmission Roller Chains and	
Sprocket Teeth	B29.10M-1997
Combination Chains, Attachments, and Sprocket Teeth	B29.11M-1994
Steel Bushed Rollerless Chains, Attachments, and Sprocket Teeth	. B29.12M-1983(R1988)
"H" Type Mill Chains, Attachments, and Sprocket Teeth	B29.14M-1996
Steel Roller Type Conveyor Chains, Attachments, and Sprocket Teeth	B29.15M-1997
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