ASME B18.2.3.4M-2001 [Revision of ANSI/ASME B18.2.3.4M-1984 (R1995)]

METRIC HEX FLANGE SCREWS

AN AMERICAN NATIONAL STANDARD





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ASME B18.2.3.4M-2001 [Revision of ANSI/ASME B18.2.3.4M-1984 (R1995)]

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FOREWORD

American National Standards Committee B18 for the standardization of bolts, screws, nuts, rivets, and similar fasteners was organized in March 1922 as Sectional Committee B18 under the aegis of the American Engineering Standards Committee (later the American Standards Association, then the United States of America Standards Institute and, as of October 6, 1969, the American National Standards Institute, Inc.), with the Society of Automotive Engineers and the American Society of Mechanical Engineers as joint sponsors. Subcommittee 2 was subsequently established and charged with the responsibility for technical content of standards covering wrench head bolts and nuts.

At its meeting on December 4, 1974, Committee B18 authorized preparation of a series of standards for metric fasteners. Subcommittee 2 was assigned responsibility for developing standards for metric hex bolts, screws, and nuts.

At a meeting on September 22, 1976, Subcommittee 2 organized the contents of a standard covering eight different hex head screw and bolt products. Actual drafting was postponed until ISO/TC2 could reach final decisions relating to basic dimensions and characteristics of hex bolts, screws, and nuts. At ISO/TC2 meetings held in April 1977, final actions were taken. Committee B18 affirmed the TC2 decisions at a meeting on June 29, 1977, and drafting of this Standard was started.

In February 1978, Committee B18 established a cooperative program with the Department of Defense to draft American National Standards for metric fasterners in such a way that they could be used directly by the government for procurement purposes. The Department of Defense requested that each of the eight products be covered in separate standards, and Subcommittee 2 accepted this approach at its meeting on June 27, 1978. The standard was approved by the secretariat and submitted to the American National Standards Institute for designation as an American National Standard. This was granted on May 17, 1979.

ANSI B18.2.3.4M-1979 was introduced to ISO/TC2 by the United States of America with the recommendation that it be considered as the basis for development of ISO standards for metric hex flange screws. At the March 1980 ISO/TC2 meetings, the Canadian delegation was invited to accept responsibility for an analytical research study to examine the relationships between wrenchability, head material volume, head proportioning, and manufacturing convenience. Stelco, Inc., conducted the study with E. M. Alexander supervising.

Following these ISO/TC2 meetings, the Canadian research was reviewed, and based on the findings, designs of metric hex flange and metric heavy hex flange screws were established. Preparation of ISO standards for both series of flange screws was authorized.

Following these ISO/TC2 decisions, B18 subcommittee 2 initiated this revision of ANSI B18.2.3.4M-1979 with the purpose being to bring it into conformance with the proposed ISO standards. Relatively few changes were necessary; the principal ones included:

- (a) discontinuation of size M20
- (b) modest adjustments in values for flange diameters bearing circle diameters, head heights, and wrenching heights
 - (c) slight change in width across corners tolerances
 - (d) a more effective and simplified technique for gaging the head

This revised Standard is in essential accord with ISO standards now under development.

This revision was approved by letter ballot of Subcommittee 2. Following its approval by letter ballot of the B18 Committee and the American Society of Mechanical Engineers, the revision was submitted to the American National Standards Institute for recognition as an American National Standard. This was granted on March 23, 1984.

The changes in this 2001 revision follow those made for the metric hex cap screw and include the following:

- (a) The government standard items and part numbering system of Appendix III is deleted and replaced by a B18 standard part numbering reference to ASME B18.24.1 found in para. 28.1.
 - (b) Paragraph 7 makes clear that the head height does not include raised markings.
- (c) Bearing face runout is now to be measured on the actual bearing circle instead of at a minimum bearing circle as described in para. 13.
- (d) Straightness at maximum material condition is specified. A rail gage replaces the sleeve gage as specified in para. 18.
- (e) The transition threads shall have a rounded root contour. This replaces the requirement of a rounded root contour, no radius of which shall be less than the specified minimum at the root of the full form thread for Property Class 10.9 and higher, per para. 20.4.
- (f) In Tables 1 and 5 for M16, the head height, K max., the wrenching height, K_w min., and the thickness of Gage A, T_a max., and T_a min., are each increased by 0.1 mm to agree with ISO/DIS 4162.2.
- (g) A new Nonmandatory Appendix A details a comparison with ISO 4162 and ISO/DIS 4162.2.
 - (h) Dimensional conformance has been specified in para. 26.
 - (i) Workmanship and accordance with ASTM F 788/F 788M is specified in para. 24. This Standard was approved by the American National Institute on March 9, 2001.

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(The following is the roster of the Committee at the time of approval of this Standard.)

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> Secretary, B18 Standards Committee The American Society of Mechanical Engineers Three Park Avenue New York, NY 10016-5990

Proposing Revisions. Revisions are made periodically to the Standard to incorporate changes that appear necessary or desirable, as demonstrated by the experience gained from the application of the Standard. Approved revisions will be published periodically.

The Committee welcomes proposals for revisions to this Standard. Such proposals should be as specific as possible, citing the paragraph number(s), the proposed wording, and a detailed description of the reasons for the proposal, including any pertinent documentation.

Interpretations. Upon request, the B18 Committee will render an interpretation of any requirement of the Standard. Interpretations can only be rendered in response to a written request sent to the Secretary of the B18 Standards Committee.

The request for interpretation should be clear and unambiguous. It is further recommended that the inquirer submit his/her request in the following format:

Subject: Cite the applicable paragraph number(s) and the topic of the inquiry. Edition:

Cite the applicable edition of the Standard for which the interpretation

is being requested.

Phrase the question as a request for an interpretation of a specific Question:

> requirement suitable for general understanding and use, not as a request for an approval of a proprietary design or situation. The inquirer may also include any plans or drawings which are necessary to explain the question; however, they should not contain proprietary names or

information.

Requests that are not in this format will be rewritten in this format by the Committee prior to being answered, which may inadvertently change the intent of the original request.

ASME procedures provide for reconsideration of any interpretation when or if additional information that might affect an interpretation is available. Further, persons aggrieved by an interpretation may appeal to the cognizant ASME Committee or Subcommittee. ASME does not "approve," "certify," "rate," or "endorse" any item, construction, proprietary device, or activity.

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METRIC HEX FLANGE SCREWS

1 SCOPE

- (a) This Standard covers the complete dimensional and general data for metric series hex flange screws recognized as American National Standard.
- (b) The inclusion of dimensional data in this Standard is not intended to imply that all products described are stock production items. Consumers should consult with suppliers concerning availability of products.

2 COMPARISON WITH ISO 4162

(a) Hex flange screws as presented in this Standard are harmonized to the extent possible with ISO 4162: 1990, and with its draft revision, ISO/DIS 4162.2: 1998. Dimensional differences between this Standard and ISO 4162 are few, relatively minor, and none will affect functional interchangeability of screws manufactured to the requirements of either.

This Standard specifies some requirements that are not included in ISO 4162. Dimensional requirements shown in bold type are in addition to, or differ from, ISO 4162: 1990 and ISO/DIS 4162.2: 1998. The technical differences between this Standard and the ISO documents are described in Nonmandatory Appendix A.

(b) Letter symbols designating dimensional characteristics are in accord with ISO 225 and ISO 4162, except where capitals have been used instead of lowercase letters used in the ISO standards.

3 REFERENCED STANDARDS

The following is a list of publications referenced in this Standard. Unless otherwise specified, the standard(s) referenced shall be the most recent issue at the time of order placement.

ASME B1.3M Screw Thread Gaging Systems for Dimensional Acceptability—Inch and Metric Threads (UN, UNR, UNJ, M, and MJ)

ASME B1.13M Metric Screw Threads—M Profile ASME B18.2.8 Clearance Holes for Inch and Metric Bolts and Screws

- ASME B18.12 Glossary of Terms for Mechanical Fasteners
- ASME B18.18.1M Inspection and Quality Assurance for General Purpose Fasteners
- ASME B18.18.2M Inspection and Quality Assurance for High-Volume Machine Assembly Fasteners
- ASME B18.24.1 Part Identifying Number (PIN) Code System Standard for B18 Externally Threaded Fasteners
- ASME Y14.5M Dimensioning and Tolerancing
- Publisher: The American Society of Mechanical Engineers (ASME International), Three Park Avenue, New York, NY 10016-5990; Order Department: 22 Law Drive, Box 2300, Fairfield, NJ 07007
- ASTM F 468M Nonferrous Bolts, Hex Cap Screws, and Stands for General Use (Metric)
- ASTM F 568M Carbon and Alloy Steel Externally Threaded Metric Fasteners
- ASTM B 633 Standard Specification for Electrodeposited Coations of Zinc on Iron and Steel
- ASTM F 738M Stainless Steel Metric Bolts, Screws, and Studs
- ASTM F 788/F 788M Surface Discontinuities of Bolts, Screws and Studs, Inch and Metric Series
- ASTM F 871M Standard Specification for Electrodeposited Coatings on Threaded Components [Metric] (Discontinued 2001)
- ASTM F 1137 Standard Specification for Phosphate/ Oil and Phospate/Organic Corrosion Protective Coatings for Fasterners
- Publisher: American Society for Testing and Materials (ASTM), 100 Barr Harbor Drive, West Conshohocken, PA 19428
- ISO 225 Fasteners—Bolts, Screws, Studs and Nuts— Symbols and Designations of Dimensions
- ISO 898-1 Mechanical Properties of Fasteners Made of Carbon Steel and Alloy Steel—Part 1: Bolts, Screws and Studs
- ISO 1891 Bolts, Screws, Nuts and Accessories—Terminology and Nomenclature
- ISO 4162 Hexagon Flange Bolts—Small Series ISO/DIS 4162.2 Hexagon Bolts With Flange—Small

Series—Product Grade Combination A/B

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ISO 4753 Fasterners—End of Parts With External ISO Metric Thread

ISO 4759-1 Tolerances for Fasteners—Part 1: Bolts, Screws, Studs and Nuts—Product Grades A, B and C
Publisher: International Organization for Standardization (ISO), 1 rue de Varembé, Case Postale 56, CH-1211, Genève 20, Switzerland/Suisse

SAE J1199 Mechanical and Material Requirements for Metric Externally Threaded Steel Fasteners

Publisher: Society of Automotive Engineers (SAE), 400 Commonwealth Drive, Warrendale, PA 15096

4 TERMINOLOGY

For definitions of terms relating to fasteners or features thereof used in this Standard, refer to ASME B18.12.

5 DIMENSIONS

- (a) All dimensions in this Standard are given in millimeters (mm), and apply before any coating, unless stated otherwise.
- (b) Symbols specifying geometric characteristics are in accord with ASME Y14.5M.

6 TOP OF HEAD

The top of head shall be either full form or indented at manufacturer's option, and shall be either chamfered or rounded. The diameter of the chamfer circle or start of rounding shall be equal to the maximum width across flats, S maximum, within a tolerance of -15%. If the top of the head is indented, the periphery may be rounded.

7 HEAD HEIGHT

The head height, *K*, is the distance, parallel to the axis of the screw, from the plane of the bearing circle to the top of the head, not including any raised markings. See para. 22.

8 WRENCHING HEIGHT

The wrenching height, K_w , is the distance at a corner of the hexagon from the junction of hex head with the flange to the last plane of full-formed hexagon (i.e., the plane closest to the top of the head at which the width across corners, E, of the hexagon is within its specified limits).

9 CORNER FILL

The rounding due to lack of fill at the six corners of the head shall be reasonably uniform.

10 GAGING OF HEX FLANGE HEAD

See Table 1. The head shall be gaged using two ring gages, A and B, to demonstrate the coincidental acceptability of wrenching height, corner fill, and width across corners. Gage A shall be placed over the head and shall seat on the flange. Gage B shall be placed on the top of the head normal to the screw axis. The two gages shall not be in contact.

11 POSITION OF HEAD

At maximum material condition, the axis of the hexagon of the head shall be within a positional tolerance zone of the diameter specified in Table 2 with respect to the axis of the shank over a distance under the head equal to the nominal screw diameter, *D*. The datum shall be as close to the head as practicable, but within 0.5*D* from the head, and shall be either wholly plain body or wholly the thread pitch diameter, not including the thread runout or the underhead fillet.

12 FLANGE

The top surface of the flange shall be conical or slightly rounded (convex). Radius, R_2 , applies both at the corners and at the flats of the hexagon. The contour of edge at flange periphery, between the maximum flange diameter, D_c maximum, and the minimum bearing circle diameter, D_w minimum, shall be optional provided that the minimum flange edge thickness, C minimum, is maintained at the minimum bearing circle diameter, D_w minimum.

13 BEARING SURFACE

The bearing surface shall be conical, 0.75 ± 0.50 deg concave from the plane formed by the bearing circle diameter. The plane formed by the bearing circle shall be perpendicular to the axis of the shank, over a length under the head equal to the nominal screw diameter, D, within the circular runout as specified in Table 2. The measurement of bearing face runout shall be made at the actual bearing circle (i.e., at the line of highest points on any radial line, e.g., by use of straight edge anvil). The datum shall be as close to the head as practical, but within 0.5D from the head, and shall be

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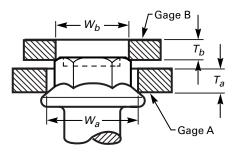


TABLE 1 GAGING OF HEX FLANGE HEAD

Nominal		Gag	e A		Gage B				
Screw Diameter	Inside D	Diameter, W _a	Thicknes	s, T _a	Inside Diam	Inside Diameter, W _b			
and Thread Pitch	Max.	Min. [Note (1)]	Max. [Note (2)]	Min.	Max. [Note (3)] Min.		Min.		
M5 × 0.8	8.09	8.08	2.30	2.29	7.43	7.42	3.0		
M6 × 1	9.25	9.24	2.90	2.89	8.55	8.54	3.0		
M8 × 1.25	11.56	11.55	3.80	3.79	10.79	10.78	4.0		
M10 × 1.5	15.02	15.01	4.30	4.29	14.07	14.06	4.0		
M12 × 1.75	17.33	17.32	5.40	5.39	16.31	16.30	5.0		
M14 × 2	20.79	20.78	5.60	5.59	19.67	19.66	5.0		
M16 × 2	24.26	24.25	6.80	6.79	22.57	22.56	6.0		

GENERAL NOTES:

- (a) Dimensions are in millimeters.
- (b) Refer to para. 10.

NOTES:

- (1) W_a min. equals theoretical maximum width across corners, E max.
- (2) T_a max. equals minimum wrenching height, K_w min.
- (3) W_b max. equals minimum width across corners, E min., -0.01 mm.

TABLE 2 TOLERANCE ZONES

Nominal Screw Diameter and Thread Pitch	Position of Head-to-Shank Tolerance Zone Diameter at MMC	Circular Runout of Bearing Circle to Shank FIM [Note (1)]
M5 × 0.8	0.44	0.16
M6 × 1	0.44	0.20
M8 × 1.25	0.44	0.26
M10 × 1.5	0.54	0.33
M12 × 1.75	0.54	0.39
$M14 \times 2$	0.54	0.46
M16 × 2	0.66	0.53

GENERAL NOTE: Dimensions are in millimeters. NOTE:

either wholly plain body or wholly thread pitch diameter, not including the thread runout or the underhead fillet.

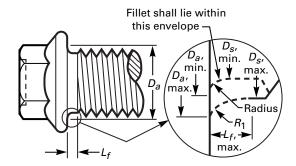
14 FILLET

The fillet configuration at the junction of the head and shank shall conform to either Type F, as shown in Table 3, which also specifies limits, or Type U, as shown in Table 4, at the option of the manufacturer unless the fillet type is specified by the purchaser. The fillet shall be a smooth and continuous curve fairing smoothly into the bearing surface and the shank within the limits specified. For Type F, no radius in the fillet contour shall be less than R_1 minimum specified in Table 3.

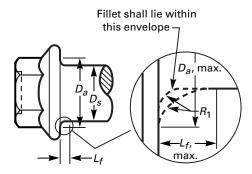
15 BODY DIAMETER

The diameter of the body, D_s , on screws that are not threaded full length shall be within the limits specified in Table 5, unless the purchaser specifies

⁽¹⁾ Circular runout of bearing circle to shank is based on 1 degree and the minimum bearing circle diameter, D_w min.



(a) Detail for Short Screws



(b) Detail for Long Screws

TABLE 3 DIMENSIONS OF TYPE F UNDERHEAD FILLETS

		ion Diameter, D _a	Body Diameter, <i>D_s</i> [Note (2)]	Fillet I	Fillet Radius, <i>R</i> 1		
Nominal Screw Diameter and Thread	For Short [Note (1)] and Long Screws	For Short [Note (1)] Screws	For Short [Note (1)] Screws	For Long Screws	For Short [Note (1)] Screws	For Short [Note (1)] and Long Screws	
Pitch	Max.	Min.	Min.	Max.	Max.	Min.	
M5 × 0.8	5.7	5.1	4.36	1.4	0.7	0.2	
M6 × 1	6.8	6.2	5.21	1.6	0.9	0.25	
M8 × 1.25	9.2	8.3	7.04	2.1	1.1	0.4	
M10 × 1.5	11.2	10.2	8.86	2.1	1.2	0.4	
M12 × 1.75	13.7	12.2	10.68	2.1	1.3	0.6	
$M14 \times 2$	15.7	14.1	12.50	2.1	1.4	0.6	
M16 × 2	17.7	16.5	14.50	3.2	1.6	0.6	

GENERAL NOTES:

- (a) Dimensions are in millimeters.
- (b) Type F was formerly named Style A.

NOTES:

- (1) Short screws are screws that are threaded full length.
- (2) Values of D_s for long screws and D_s max. for short screws are specified in Table 5.

screws with reduced diameter body. For screws threaded full length, the diameter of the unthreaded shank under the head shall neither exceed the maximum body diameter, D_s maximum, specified in Table 5 nor be less than the minimum body diameter, D_s minimum, specified in Table 3 or 4.

Screws of nominal lengths equal to or greater than the shortest nominal lengths specified in Table 6 may be obtained with reduced diameter body, if so specified. Where reduced diameter body (or "Type R") is specified, the body diameter, D_2 , shall be within the limits specified in Table 6. The screw shall have a shoulder under the head. The diameter, D_s , and length, L_2 , of the shoulder shall be as specified in Table 6.

16 LENGTH

The length of the screw is the distance, parallel to the axis of the screw, from the plane formed by the underhead bearing circle diameter to the extreme end of the shank. Tolerances for screw lengths are specified in Table 7.

17 POINTS

The end of the screw shall be chamfered or rounded from a diameter equal to or slightly less than the thread root diameter. The length of the point to the first fullformed thread at major diameter, as determined by the METRIC HEX FLANGE SCREWS ASME B18.2.3.4M-2001

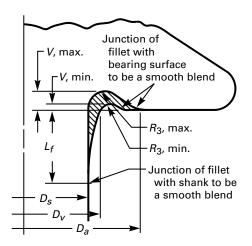


TABLE 4 DIMENSIONS OF TYPE U UNDERHEAD FILLETS

Nominal Screw Diameter and Thread	Undercut Diameter, <i>D</i> _a [Note (1)]	Fillet Diameter, <i>D_v</i> [Note (2)]	Body Diameter For Short Screws, D _s [Notes (3) and (4)]	Fillet Length, <i>L</i> _f		ercut us, <i>R</i> 3		ercut th, <i>V</i>
Pitch	Max.	Max.	Min.	Max.	Max.	Min.	Max.	Min.
M5 × 0.8	6.2	5.5	4.36	1.4	0.25	0.10	0.15	0.05
M6 × 1	7.5	6.6	5.21	1.6	0.26	0.11	0.20	0.05
M8 × 1.25	10.0	8.8	7.04	2.1	0.36	0.16	0.25	0.10
M10 × 1.5	12.5	10.8	8.86	2.1	0.45	0.20	0.30	0.15
M12 × 1.75	15.2	12.8	10.68	2.1	0.54	0.24	0.35	0.15
$M14 \times 2$	17.7	14.8	12.50	2.1	0.63	0.28	0.45	0.20
$M16 \times 2$	20.5	17.2	14.50	3.2	0.72	0.32	0.50	0.25

GENERAL NOTES:

- (a) Dimensions are in millimeters.
- (b) Type U was formerly named Style B.

NOTES:

- (1) D_a was formerly D_u .
- (2) D_v was formerly D_a .
- (3) Values of D_s for long screws and D_s max. for short screws are specified in Table 5.
- (4) Short screws are screws that are threaded full length.

distance the point enters into a cylindrical NOT GO major diameter ring gage, shall not exceed U maximum specified in Table 8. The end of the screw shall be reasonably square with the axis of the screw, and where pointed blanks are used, the slight rim or cup resulting from roll threading shall be permissible. At the manufacturer's option, the end of the screw may have a rounded point of radius, R_e , as specified in Table 8.

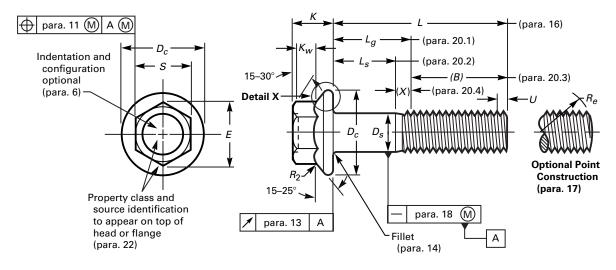
18 STRAIGHTNESS

At maximum material condition, the derived median line of the screw body and thread major diameter shall be within a straightness tolerance zone of a diameter equal to **0.006 times length**, **expressed as a two-place decimal.** A gage and gaging procedure for checking screw straightness are given in Nonmandatory Appendix B.

19 THREADS

19.1 Thread Series and Tolerance Class

Screw threads shall be general purpose metric screw threads with tolerance Class 6g conforming to ASME B1.13M, unless otherwise specified by the purchaser.



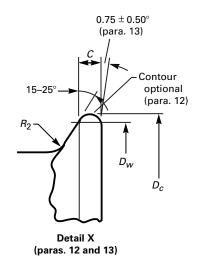


TABLE 5 DIMENSIONS OF HEX FLANGE SCREWS

Nominal Screw Diameter and Thread	Diam	dy ieter,) _s	Acr Fla	dth oss its,	Corr	dth oss ners,	Flange Diamter, <i>D_c</i>	Bearing Circle Diameter, D _w	Flange Edge Thickness, <i>C</i>	Head Height, <i>K</i>	Wrenching Height, <i>K</i> _w	Flange Top Fillet Radius, <i>R</i> ₂
Pitch	Max.	Min.	Max.	Min.	Max.	Min.	Max.	Min.	Min.	Max.	Min.	Max.
M5 × 0.8	5.00	4.82	7.00	6.64	8.08	7.44	11.4	9.4	1.0	5.6	2.3	0.3
$M6 \times 1$	6.00	5.82	8.00	7.64	9.24	8.56	13.6	11.6	1.1	6.9	2.9	0.4
M8 × 1.25	8.00	7.78	10.00	9.64	11.55	10.80	17.0	14.9	1.2	8.5	3.8	0.5
M10 × 1.5	10.00	9.78	13.00	12.57	15.01	14.08	20.8	18.7	1.5	9.7	4.3	0.6
M12 × 1.75	12.00	11.73	15.00	14.57	17.32	16.32	24.7	22.5	1.8	12.1	5.4	0.7
$M14 \times 2$	14.00	13.73	18.00	17.57	20.78	19.68	28.6	26.4	2.1	12.9	5.6	0.9
M16 × 2	16.00	15.73	21.00	20.16	24.25	22.58	32.8	30.6	2.4	15.2	6.7	1.0
See para.	1	5			8, 9	, 10	12	12, 13	12	7, 22	8, 9, 10	12

GENERAL NOTE: Dimensions are in millimeters.

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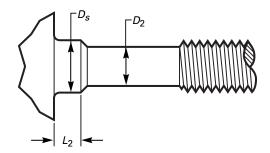


TABLE 6 DIMENSIONS OF REDUCED BODY DIAMETER (TYPE R)

Nominal Screw Diameter and	Shortest Nominal Screw Length,	Reduced Body Diameter, <i>D</i> ₂			Diameter, ote (1)]	Shoulder Length, L ₂ [Note (1)]	
Thread Pitch	Screw Length, L	Max.	Min.	Max.	Min.	Max.	Min.
M5 × 0.8	30	4.54	4.36	5.00	4.82	3.5	2.5
M6 × 1	35	5.39	5.21	6.00	5.82	4.0	3.0
M8 × 1.25	40	7.26	7.04	8.00	7.78	5.0	4.0
M10 × 1.5	45	9.08	8.86	10.00	9.78	6.0	5.0
M12 × 1.75	50	10.95	10.68	12.00	11.73	7.0	6.0
$M14 \times 2$	55	12.77	12.50	14.00	13.73	8.0	7.0
M16 × 2	60	14.77	14.50	16.00	15.73	9.0	8.0

GENERAL NOTE: Dimensions are in millimeters.

NOTE:

(1) Shoulder is mandatory. See para. 15.

TABLE 7 LENGTH TOLERANCES

Nominal Length, mm	Length Tolerance, mm
Over 6 to 10	±0.29
Over 10 to 18	±0.35
Over 18 to 30	±0.42
Over 30 to 50	±0.5
Over 50 to 80	±0.6
Over 80 to 120	±0.7
Over 120 to 180	±0.8
Over 180 to 250	±0.925
Over 250 to 300	±1.05

For screws with additive finish, size limits for tolerance Class 6g apply before coating, and the thread profile after coating is subject to acceptance using a 6h GO thread gage and tolerance Class 6g thread gage for either minimum material, LO or NOT GO.

19.2 Thread Gaging

Unless otherwise specified, dimensional acceptability of screw threads shall be determined based on System 21, ASME B1.3M.

TABLE 8 DIMENSIONS OF POINTS

Nominal Screw Diameter and Thread Pitch	Approximate Point Radius, <i>R_e</i>	Max. Point Length, <i>U</i>		
M5 × 0.8	7.0	1.6		
M6 × 1	8.4	2.0		
M8 × 1.25	11.2	2.5		
M10 × 1.5	14.0	3.0		
M12 × 1.75	16.8	3.5		
$M14 \times 2$	19.6	4.0		
M16 × 2	22.4	4.0		

GENERAL NOTES:

- (a) Dimensions are in millimeters.
- (b) R_e equals 1.4 times the nominal screw diameter, and conforms with ISO 4753.
- (c) U max. equals two times the thread pitch.

20 THREAD LENGTH

The length of thread on screws is controlled by the maximum grip length, L_g maximum, and the minimum body length, L_s minimum, as set forth in paras. 20.1 through 20.4.

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TABLE 9 MAXIMUM GRIP LENGTHS, L_a , AND MINIMUM BODY LENGTHS, L_s

					l	Nominal	Diamete	r and Ti	hread Pit	ch				
	M5 >	< 0.8	M6	M6 × 1		M8 × 1.25		× 1.5	M12 × 1.75		M14 × 2		M16 × 2	
Nominal Length,	L _g Max.	<i>L₅</i> Min.	L _g Max.	<i>L₅</i> Min.	L _g Max.	L _s Min.	L _g Max.	<i>L₅</i> Min.	L _g Max.	L _s Min.	L _g Max.	<i>L₅</i> Min.	L _g Max.	<i>L₅</i> Min.
8	1.2													
10	2.4		1.5											
12	2.4		3.0		1.9									
16	2.4		3.0		4.0		2.2							
20	2.4		3.0		4.0		4.5		2.6		3.0			
25	9.0	5.0	3.0		4.0		4.5		5.3		3.0		3.0	
30	14.0	10.0	12.0	7.0	4.0		4.5		5.3		6.0		3.0	<u> </u>
35	19.0	15.0	17.0	12.0	13.0	6.75	4.5		5.3		6.0		6.0	
40	24.0	20.0	22.0	17.0	18.0	11.75	14.0	6.5	5.3		6.0		6.0	
45	29.0	25.0	27.0	22.0	23.0	16.75	19.0	11.5	15.0	6.25	6.0		6.0	
50	34.0	30.0	32.0	27.0	28.0	21.75	24.0	16.5	20.0	11.25	16.0	6.0	6.0	
55			37.0	32.0	33.0	26.75	29.0	21.5	25.0	16.25	21.0	11.0	17.0	7.0
60			42.0	37.0	38.0	31.75	34.0	26.5	30.0	21.25	26.0	16.0	22.0	12.0
65					43.0	36.75	39.0	31.5	35.0	26.25	31.0	21.0	27.0	17.0
70					48.0	41.75	44.0	36.5	40.0	31.25	36.0	26.0	32.0	22.0
80					58.0	51.75	54.0	46.5	50.0	41.25	46.0	36.0	42.0	32.0
90							64.0	56.5	60.0	51.25	56.0	46.0	52.0	42.0
100							74.0	66.5	70.0	61.25	66.0	56.0	62.0	52.0
110									80.0	71.25	76.0	66.0	72.0	62.0
120									90.0	81.25	86.0	76.0	82.0	72.0
130											90.0	80.0	86.0	76.0
140											100.0	90.0	96.0	86.0
150													106.0	96.0
160													116.0	106.0

GENERAL NOTES:

- (a) Dimensions are in millimeters.
- (b) Diameter-length combinations between the thin stepped lines are recommended.
- (c) Screws with lengths above the solid thick stepped lines are threaded full length. See Table 10.
- (d) For screws with longer lengths, L_g and L_s values shall be computed from formulas as given in para. 20.

20.1 Grip Length, L_q

The grip length, L_g , is the distance, measured parallel to the axis of the screw, from the plane of the bearing circle diameter to the face of a noncounterbored, noncountersunk GO thread ring gage assembled by hand as far as the thread will permit. For standard diameter-length combinations of screws, the values for L_g maximum are specified in Table 9. For diameter-length combinations not listed in Table 9, the maximum grip length for long screws that are not threaded full length is equal to the nominal screw length, L, minus the

reference thread length, (B), as specified in Table 10: L_g maximum = L nominal – (B).

For short screws of nominal lengths, L, that are shorter than the lengths specified in Table 10 for screws threaded full length, L_g maximum = A maximum as specified in Table 10.

20.2 Body Length, L_s

Body length, L_s , on long screws that are not threaded full length is the distance, parallel to the axis of the screw, from the plane of the bearing circle to the last

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TABLE 10 THREAD LENGTHS

	Threa	d Length, (E	3, Ref.)		Screws Threaded Full Length						
Nominal	S	crew Length	, L		Screw	Unthreaded Length			Unthreaded Length		
Screw Diameter		>125		Transition Thread	Length, <i>L</i>	Under Head, <i>A</i>	Screw	Length, <i>L</i>	Under Head, <i>A</i>		
and Thread Pitch	≤125	and ≤200	>200	Length (X, Ref.)	Under	Max.	At Least	Under	Max.		
M5 × 0.8	16	22	35	4.0	10	1.2	10	25	2.4		
$M6 \times 1$	18	24	37	5.0	12	1.5	12	30	3.0		
M8 × 1.25	22	28	41	6.25	16	1.9	16	35	4.0		
M10 × 1.5	26	32	45	7.5	20	2.2	20	40	4.5		
M12 × 1.75	30	36	49	8.75	24	2.6	24	45	5.3		
$M14 \times 2$	34	40	53	10.0	28	3.0	28	50	6.0		
M16 × 2	38	44	57	10.0	32	3.0	32	55	6.0		

GENERAL NOTE: Dimensions are in millimeters.

scratch of thread or top of the extrusion angle, whichever is closer to the head. For standard diameter-length combinations of screws, the values for L_s minimum are specified in Table 9. For diameter-length combinations not listed in Table 9, the minimum body length on screws that are not threaded full length is equal to the maximum grip length, as determined above, minus the transition thread length, (X), as specified in Table 10: L_s minimum = L_g maximum – (X).

20.3 Thread Length, (B)

The thread length, (B), specified in Table 10, is a reference dimension intended for calculation purposes only, and is the distance, parallel to the axis of the screw, from the extreme end of the screw to the last complete (full-form) thread.

20.4 Transition Thread Length, (X)

The transition thread length, (X), specified in Table 10, is a reference dimension intended for calculation purposes only. It includes the length of incomplete threads and tolerances on grip length and body length. The transition from full-form thread to incomplete thread shall be smooth and uniform. The major diameter of the incomplete threads shall not exceed the actual major diameter of the complete (full-form) threads. The transition threads shall have a rounded root contour.

21 MATERIALS AND MECHANICAL PROPERTIES

21.1 Steel

Unless otherwise specified, steel screws shall conform to the requirements for Property Class 9.8 or Property Class 10.9 as specified in ASTM F 568M, or SAE J1199.

21.2 Corrosion-Resistant Steels

Unless otherwise specified, corrosion-resistant steel screws shall conform to the requirements of ASTM F 738M.

21.3 Nonferrous Metals

Unless otherwise specified, nonferrous screws shall conform to the requirements of ASTM F 468M.

22 IDENTIFICATION SYMBOLS

Markings shall be raised or recessed on the top of the head or raised on the top of the flange unless otherwise specified by the purchaser. Markings shall be legible to the unaided eye with the exception of corrective lenses. When raised, markings shall project not less than 0.1 mm for M14 and smaller screws, and 0.3 mm for M16 screws, above the top surface of the head or flange; and total head height (head plus markings) shall not exceed the specified maxi-

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mum head height, *K* maximum, plus 0.1 mm for M5 and M6 screws, 0.2 mm for M8 and M10 screws, 0.3 mm for M12 and M14 screws, and 0.4 mm for M16 screws.

22.1 Property Class Symbols

Each screw shall be marked in accordance with the requirements of the applicable specification for its material and mechanical properties.

22.2 Source Symbols

Each screw shall be marked to identify its source (manufacturer or private label distributor).

23 FINISH

Unless otherwise specified, screws shall be supplied with a natural (as processed) finish, unplated or uncoated, in a clean condition, and lightly oiled.

24 WORKMANSHIP

Screws shall be free from surface imperfections such as burrs, seams, laps, loose scale, and other surface irregularities that could affect serviceability and shall conform to ASTM F 788/F 788M.

25 INSPECTION AND QUALITY ASSURANCE

Unless otherwise specified, acceptability of screws shall be determined in accordance with ASME B18.18.1M.

26 DIMENSIONAL CONFORMANCE

Products shall conform to the specified dimensions. Unless otherwise specified, the following provisions shall apply for inspection of dimensional characteristics:

(a) Unless otherwise specified, the following designated dimensional characteristics shall be inspected to the inspection levels shown according to ASME B18.18.2M, and shall be within their specified limits.

Characteristic	Inspection Level
Thread acceptability	С
Body diameter, D_s	C
Gaging of hex flange head	C
Grip length, L_g max.	C
Screw length, L	C
Visual inspection [Note (1)]	C

NOTE:

(1) Visual inspection shall include property class marking, source marking, fillet, and workmanship.

If verifiable in-process inspection is used, inspection sample sizes and reporting shall be in accordance with the applicable ASME, ASTM, or SAE quality system consensus standard.

(b) For nondesignated dimensional characteristics, the provisions of ASME B18.18.1M shall apply. Should a nondesignated dimension be determined to be outside its specified limits, it shall be deemed conforming to this Standard if the user, who is the installer, accepts the dimension, based on form, fit, and function considerations.

27 CLEARANCE HOLES

The recommended sizes of clearance holes in material to be assembled using hex flange screws are the normal series given in ASME B18.2.8.

28 DESIGNATION

(a) Hex flange screws shall be designated by the following data, preferably in the sequence shown: product name and designation of the standard, nominal diameter and thread pitch, nominal length, steel property class or material identification, and protective coating, if required.

EXAMPLES:

- (1) Hex flange screw B18.2.3.4M, M10 \times 1.5 \times 50, Class 9.8, zinc plated per ASTM F 1941M and ASTM B 633 Fe/Zn 5 Type II.
- (2) Hex flange screw B18.2.3.4M, M6 \times 2 \times 100, Class 10.9, phosphate/oil, ASTM F 1137 Grade I.
- (3) Hex flange screw B18.2.3.4M, reduced diameter body, fillet Type U, M16 \times 2 \times 80, Class 10.9 oiled.
- (4) Hex flange screw B18.2.3.4M, M16 \times 1 \times 30, A1-70 ASTM F 738M.

NOTE: It is common practice in ISO standards to omit thread pitch from the nominal size designation when screw threads are the metric coarse thread series; e.g., M10 is M10 \times 1.5.

(b) For a recommended part identifying number (PIN) system for metric hex flange screws, see ASME B18.24.1.

NONMANDATORY APPENDIX A COMPARISON WITH ISO STANDARDS

This Appendix describes the technical differences between ASME B18.2.3.4M-2001 and ISO 4162: 1990, its draft revision, ISO/DIS 4162.2: 1998, and related ISO standards.

A1 PRODUCT NAMES BOLT AND SCREW

Of the products named *hex flange screw* per ASME ASME B18.2.3.4M, those that are threaded full length would be named *hexagon screw with flange* per ISO 1891, but those that are not threaded full length are named *hexagon flange bolt* per ISO 4162: 1990, and *hexagon bolt with flange* per ISO 1891 and ISO/DIS 4162.2: 1998.

A2 HEAD HEIGHT AND WRENCHING HEIGHT

ASME B18.2.3.4M Table 5 specifies maximum head heights for M6, M12, and M16, and minimum wrenching height for M16, in agreement with ISO/DIS 4162.2: 1998, which are increased from those specified in ISO 4162: 1990.

A3 MAXIMUM WIDTH ACROSS CORNERS

ASME B18.2.3.4M, Table 5 maximum width across corners is not specified in ISO 4162. Instead, ISO/DIS 4759-1: May 1998 would control the shape of the hexagon by zero positional tolerance at maximum material condition, which is not specified in ASME B18.2.3.4M.

A4 GAGING OF HEX FLANGE HEAD

ISO/DIS 4162.2 would specify a gage C for gaging flange thickness, which is not included in ASME B18.2.3.4M.

A5 POSITION OF HEAD

ASME B18.2.3.4M, para. 11 and Table 2 specify positional tolerance at maximum material condition, as in ISO/DIS 4759-1: May 1998, instead of regardless

of feature size, as in ISO 4759/I-1978. ISO/DIS 4759-1: May 1998 also specifies position of the flange diameter with respect to the shank, which is not specified in ASME B18.2.3.4M.

A6 BEARING SURFACE

ASME B18.2.3.4M, para. 13 and Table 5 specify bearing surface concavity of 0.75±0.05 deg, as in ISO/DIS 4162.2: 1998, which is a draft revision. ISO 4162: 1990 specifies 0 to 1 degrees 30 minimum (0.75±0.75 degrees). ISO/DIS 4759-1: May 1998 specifies straightness of radial lines on the bearing surface, which is not specified in ASME B18.2.3.4M. ASME B18.2.3.4M, Table 2 values for circular runout of the bearing circle are greater than those in ISO 4759/I-1978 or ISO/DIS 4759-1: May 1998.

A7 FILLETS

ASME B18.2.3.4M, Table 3 minimum fillet transition diameters and shorter maximum fillet lengths for screws threaded full length are not specified in ISO/DIS 4162. For Type U fillet, the radius of the fillet extension, r_4 reference in ISO/DIS 4162, is not specified in ASME B18.2.3.4M.

A8 BODY DIAMETER FOR SCREWS THREADED FULL LENGTH

ASME B18.2.3.4M, Tables 3 and 4 minimum diameters of unthreaded shank for screws threaded full length are not specified in ISO/DIS 4162.

A9 REDUCED DIAMETER BODY

ASME B18.2.3.4M. Table 6 maximum and minimum diameters of reduced body differ from ISO 4162 " d_2 is approximately equal to the pitch diameter (rolling diameter)." The limits specified in B18.2.3.4M Table 6 were proposed to ISO/TC2/WG2 in May 1995, but

ASME B18.2.3.4M-2001 NONMANDATORY APPENDIX A

ISO/TC2/WG2 was not willing to specify limits for the rolling diameter.

A10 POINT

ASME B18.2.3.4M, para. 17 and Table 5 optional rounded point is not in ISO 4162, but Table 8 agrees with ISO 4753.

A11 STRAIGHTNESS

ASME B18.2.3.4M specifies straightness at maximum material condition, as in ISO/DIS 4759-I: May 1998, instead of regardless of feature size, as in ISO 4759/I-1978. The formula in ASME B18.2.3.4M results in straightness tolerance zone diameters that are smaller for lengths 12 mm and shorter, and larger for lengths 16 mm and longer, than those in ISO 4759/I-1978 or ISO/DIS 4759-1: May 1998.

A12 GRIP LENGTH FOR SCREWS THREADED FULL LENGTH

ASME B18.2.3.4M, Table 10 maximum unthreaded lengths under the head of screws shorter than 2*D* differ from ISO 3508.

A13 TRANSITION THREADS

ASME B18.2.3.4M, para. 20.4 specification on root contour of transition threads is not in ISO 4162.

A14 POSITION OF BODY-TO-THREAD

Coaxiality or position of the body with respect to the thread, specified in ISO 4759/I or ISO/DIS 4759-1, is not specified in ASME B18.2.3.4M.

A15 MARKING

ASME B18.2.3.4M, para. 22 allows markings on the top of the flange but not on the side of the head, as alternatives to markings on the top of the head, while ISO 898-1 allows markings on the side of the head but not on the top of the flange. ASME B18.2.3.4M, para. 22 dimensions of head markings are not specified in ISO 4162 or ISO 898-1.

A16 INSPECTION AND QUALITY ASSURANCE

ASME B18.2.3.4M, paras. 25 and 26 inspection and quality assurance provisions differ from those in ISO 4162.

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NONMANDATORY APPENDIX B SCREW STRAIGHTNESS GAGE AND GAGING PROCEDURE

The conformance of screws to shank straightness or camber limitations set forth in para. 18 may be checked by using the gage illustrated below in accordance with the following procedure:

- (a) Allowable total camber on the product to be inspected shall be calculated in accordance with para. 18.
- (b) The total camber thus derived shall be added to the specified maximum body diameter, and the movable rail of gage shall be adjusted to provide a parallel

space between the rails equal to this distance by obtaining common readings on both micrometer heads.

- (c) The movable rail shall then be locked in place by tightening securing screws.
- (d) The product shall then be inserted between rails and shall be rotated by hand through full 360 degrees. Any interference occurring between the product and the gage that is sufficient to prevent rotation shall indicate excessive camber.

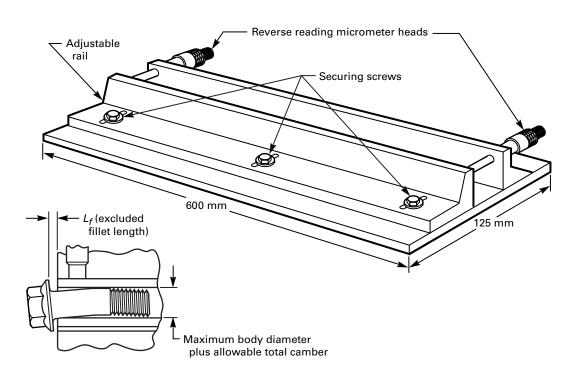


FIG. B1 TYPICAL STRAIGHTNESS GAGE

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