KEYS AND KEYSEATS

USAS B17.1 - 1967

REAFFIRMED 2003

FOR CURRENT COMMITTEE PERSONNEL PLEASE E-MAIL CS@asme.org

Sponsor

The American Society of Mechanical Engineers

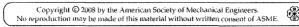
Published by

THE AMERICAN SOCIETY OF MECHANICAL ENGINEERS
United Engineering Center 345 East 47th Street New York, N. Y. 10017

This USA Standard is one of nearly 3000 standards approved as American Standards by the American Standards Association. On August 24, 1966, the ASA was reconstituted as the United States of America Standards Institute. Standards approved as American Standards are now designated USA Standards. There is no change in their index identification or technical content.

UDC 621.886.6

Copyright, 1968, by
THE AMERICAN SOCIETY OF MECHANICAL ENGINEERS
Printed in U.S.A.



Foreword

THE individual USA Standards on Keys were first issued in the late 1920's. These were consolidated with the shafting standards in the 1934 publication, Standard Shafting and Stock Keys. A revised issue was published in 1943 and withdrawn in 1955 as it was not being sufficiently supported. However, a separate standard on Woodruff Keys, Keyslots, and Cutters (B17f-1930) was reaffirmed in 1955.

In 1962 the USA Standards Committee B17 was reactivated and a sub-committee developed and established this standard based on current industry standards.

Following approval by the USA Standards Committee B17 and the sponsor, the proposed revision was approved on September 15, 1967 by the USA Standards Institute, and designated B17.1-1967.

USA Standards Committee B17

Keys and Keyseats

OFFICERS

U. P. Jennings, Chairman

Roger Daniels, Secretary

COMMITTEE MEMBERS

AMERICAN GEAR MANUFACTURERS ASSOCIATION

- G. L. Scott, American Gear Manufacturers Association, Washington, D.C.
- D. F. Wild, General Electric Company, Paterson, New Jersey

AMERICAN SOCIETY OF MECHANICAL ENGINEERS, THE

U. P. Jennings, E. I. DuPont de Nemours & Company, Inc., Wilmington, Delaware

AMERICAN SOCIETY OF TOOL AND MANUFACTURING ENGINEERS

W. M. Houck, New Holland Machine Company, New Holland, Pennsylvania

AMERICAN SPROCKET CHAIN MANUFACTURERS ASSOCIATION

R. A. Anderson, Roller Chain Division, Rex Chainbelt Company, Worcester, Massachusetts

CONVEYOR EQUIPMENT MANUFACTURERS ASSOCIATION

- C. R. Davis, Rex Chainbelt Inc., Milwaukee, Wisconsin
- A. W. Lemmon, Alternate, The Jeffrey Manufacturing Company, Columbus, Ohio
- R. C. Sollenberger, Alternate, Conveyor Equipment Manufacturers Association, Washington, D.C.

MECHANICAL POWER TRANSMISSION ASSOCIATION

D. C. Currier, Dodge Manufacturing Company, Mishawaka, Indiana

METAL CUTTING TOOL INSTITUTE

Perry L. Houser, Metal Cutting Tool Institute, New York, N.Y.

NATIONAL ELECTRICAL MANUFACTURERS ASSOCIATION

C. F. Schwan, Reliance Electric & Engineering, Cleveland, Ohio

SOCIETY OF AUTOMOTIVE ENGINEERS

E. J. Flewelling, Massey-Fergusson, Inc., Detroit, Michigan

INDIVIDUAL COMPANIES

- J. W. Gillen, John Gillen Company, Cicero, Illinois
- G. W. Andrews, Alternate, John Gillen Company, Cicero, Illinois

Roger Daniels, Formsprag Company, Warren, Michigan

W. C. Sheers, Standard Horse Nail Corporation, New Brighton, Pennsylvania

Personnel of Subcommittee 1, Rectangular Keys

- C. R. Davis, Chairman, Manager of Engineering, Conveyor Equipment, Rex Chainbelt Inc., 4701 West Greenfield Avenue, Milwaukee, Wisconsin 53214
- J. W. Gillen, President, John Gillen Co., Subsidiary of Stanray Corp., 2540 South 50th Avenue, Cicero, Illinois 60650
- C. F. Schwan, Chief Mechanical Engineer, Reliance Electric & Engineering Co., 24701 Euclid Avenue, Cleveland, Ohio 44117
- D. F. Wild, Manager-Engineering, Gearmotor Business Section, General Electric Co., 845 East 25th Street, Paterson, New Jersey 07513

Donald C. Currier, Chief of Standards and Statistics, Dodge Manufacturing Corporation, Mishawaka, Indiana 46544

TABLE OF CONTENTS

P	age
1. Introduction	. 1
2. Definitions	1
3. Key Size Versus Shaft Diameter	
4. Key Dimensions and Tolerances	3
5. Depth Control Formulas	
6. Key-Keyseat Assemblies	
7. Alignment Tolerances	
8. Chamfered Keys and Filleted Keyseats	
9. Set Screws For Use Over Keys	
Table 1 Key Size Versus Shaft Diameter	2
Table 2 Key Dimensions and Tolerances.	. 3
Table 2A Gib Head Nominal Dimensions	. 4
Table 3 Depth Control Values-Values for S and T	
Table 4 Class I - Fit for Parallel Keys	
Table 5 Class 2-Fit for Parallel and Taper Keys	
Table 6 Keyseat Tolerances for Electric Motor and Generator Shaft Extensions	
Table 7 Suggested Fillet Radii and Key Chamfer	
Table 8 Set Screws for Use Over Keys.	

Keys and Keyseats

1. INTRODUCTION

This standard establishes a uniform relationship between shaft size and key size for parallel and taper keys retaining similar basic sizing as found in the withdrawn B17.1-1943 standard.

This standard covers the size, type and tolerances of parallel and taper keys and keyseats, and their relationship to shaft diameters and bore diameters. The sizes and tolerances contained in this standard are intended for single key applications only.

Key strength and steel analysis entering into the makeup of stock for keys is not within the scope of this standard. Shaft diameters are listed for identification of various key sizes and are not intended to establish shaft dimensions, tolerances or selections.

This standard recognizes that there are two classes of stock for parallel keys presently used by industry. One is a broad negative toleranced bar stock and the other is a close plus toleranced keystock. Each is combined with appropriate keyseat tolerances to establish assemblies, respectively, designated as Classes 1 and 2.

Taper keys are established for Class 2 assembly only.

2. DEFINITIONS

2.1 Key - A demountable machinery part which, when assembled into keyseats, provides a posi-

tive means for transmitting torque between the shaft and hub. Tolerances are shown in Table 2.

- 2.1.1 Bar Stock General purpose, negative toleranced, cold finished steel stock for parallel keys.
- 2.2. 2 Keystock A close plus toleranced, cold finished steel stock for parallel keys.
- 2.2. Keyseat An axially located rectangular groove in a shaft or hub. This may also be written as shaft keyseat or hub keyseat when describing the exact application. Hub keyseat has been sometimes referred to as a keyway.
- 2.3 Keyscating The operation of producing keyseats.

3. KEY SIZE VERSUS SHAFT DIAMETER

For a stepped shaft, the size of a key is determined by the diameter of the shaft at the point of location of the key, regardless of the number of different diameters on the shaft.

Square keys are preferred through 6½-inch diameter shafts and rectangular keys for larger shafts. Sizes and dimensions in unshaded area are preferred.

If special considerations dictate the use of a keyseat in the hub shallower than the preferred nominal depth shown in Table 1, it is recommended that the tabulated preferred nominal standard keyseat be used in the shaft in all cases.



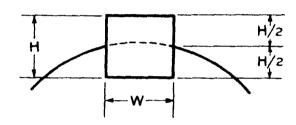


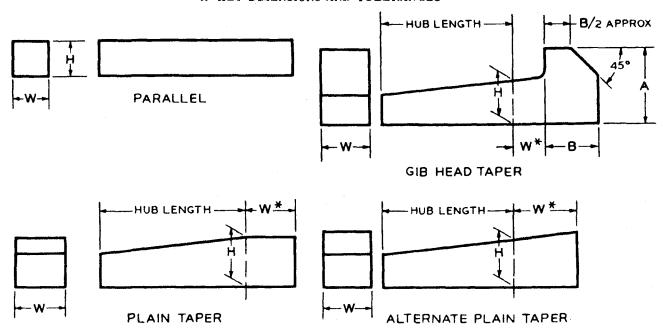
Table 1 Key Size Versus Shaft Diameter

NOMINAL SE	AFT DIAMETER		NOMINAL KEY SIZ	E	NOMINAL KEY	YSEAT DEPTH
Over	To (Incl)	Width, W	Width W Height, H		H,	/2
			Square	Rectangular	Square	Rectangular
5/16 7/16 9/16 7/8 1-1/4	7/16 9/16 7/8 1-1/4 1-3/8	3/32 1/8 3/16 1/4 5/16	3/32 1/8 3/16 1/4 5/16	3/32 1/8 3/16 1/4	3/64 1/16 3/32 1/8 5/32	3/64 1/16 3/32 1/8
1-3/8 1-3/4 2-1/4 2-3/4	1-3/4 2-1/4 2-3/4 3-1/4	3/8 1/2 5/8 3/4	3/8 1/2 5/8 3/4	1/4 3/8 7/16 1/2	3/16 1/4 5/16 3/8	1/8 3/16 7/32 1/4
3-1/4 3-3/4 4-1/2 5-1/2	3-3/4 4-1/2 5-1/2 6-1/2	7/8 1 1~1/4 1~1/2	7/8 1 1-1/4 1-1/2	5/8 3/4 7/8	7/16 1/2 5/8 3/4	5/16 3/8 7/16 1/2
6-1/2 7-1/2 9 11	7-1/2 9 11 13	1-3/4 2 2-1/2 3	1-3/4 2 2-1/2 3	1-1/2* 1-1/2 1-3/4 2	7/8 1 1-1/4 1-1/2	3/4 3/4 7/8
13 15 18 22 26	15 18 22 26 30	3-1/2 4 5 6 7	3-1/2	2-1/2 3 3-1/2 4 5	1-3/4	1-1/4 1-1/2 1-3/4 2 2-1/2

^{*}Some key standards show 1-1/4 in. Preferred size is 1-1/2 in. Shaded areas: See Part 3, page 1.
All dimensions given in inches.



4. KEY DIMENSIONS AND TOLERANCES



Plain and Gib Head Taper Keys Have a 1/8" Taper in 12"

Table 2 Key Dimensions and Tolerances

	_			L KEY SIZE	TOLE	RANCE
	KEY		Wic	ith, W	Width, W	Height, H
			Over	To (Incl)	# tdtii, "	ireight, ir
Square		Bar Stock	3/4 1-1/2 2-1/2	3/4 1-1/2 2-1/2 3-1/2	+0.000 -0.002 +0.000 -0.003 +0.000 -0.004 +0.000 -0.006	+0.000 -0.00 +0.000 -0.00 +0.000 -0.00 +0.000 -0.00
D - 21 - 4		Keystock	1-1/4 3	1-1/4 3 3-1/2	+0.001 -0.000 +0.002 -0.000 +0.003 -0.000	+0.001 -0.00 +0.002 -0.00 +0.003 -0.00
Parallel	Parallel Rectangular	Bar Stock	3/4 1-1/2 3 4 6	3/4 1-1/2 3 4 6 7	+0.000 -0.003 +0.000 -0.004 +0.000 -0.005 +0.000 -0.006 +0.000 -0.008 +0.000 -0.013	+0.000 -0.00 +0.000 -0.00 +0.000 -0.00 +0.000 -0.00 +0.000 -0.00 +0.000 -0.01
	Keystock	1-1/4 3	1-1/4 3 7	+0.001 -0.000 +0.002 -0.000 +0.003 -0.000	+0.005 -0.00 +0.005 -0.00 +0.005 -0.00	
Taper	Plain or Gib Head Square or Rectangular		1-1/4 3	1-1/4 3 7	+0.001 -0.000 +0.002 -0.000 +0.003 -0.000	+0.005 -0.00 +0.005 -0.00 +0.005 -0.00

^{*}For locating position of dimension H. Tolerance does not apply.

See Table 2A for dimensions on gib heads. All dimensions given in inches.

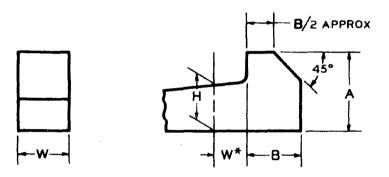


Table 2A Gib Head Nominal Dimensions

Nominal Key Size	SQUARE			REC	TANGUI	AR
Width, W	Н	Α	В	Н	Α	В
1/8	1/8	1/4	1/4	3/32	3/16	1/8
3/16	3/16	5/16	5/16	1/8	1/4	1/4
1/4	1/4	7/16	3/8	3/16	5/16	5/16
5/16	5/16	1/2	7/16	1/4	7/16	3/8
3/8	3/8	5/8	1/2	1/4	7/16	3/8
1/2	1/2	7/8	5/8	3/8	5/8	1/2
5/8	5/8	1	3/4	7/16	3/4	9/16
3/4	3/4	1-1/4	7/8	1/2	7/8	5/8
7/8	7/8	1-3/8	1	5/8	1	3/4
1	1	1-5/8	1-1/8	3/4	1-1/4	7/8
1-1/4	1-1/4	2	1-7/16	7/8	1-3/8	1
1-1/2	1-1/2	2-3/8	1-3/4	1	1-5/8	1-1/8
1-3/4	1-3/4	2-3/4	2	1-1/2	2-3/8	1-3/4
2	2	3-1/2	2-1/4	1-1/2	2-3/8	1-3/4
2-1/2	2-1/2	4	3	1-3/4	2-3/4	2
3 3-1/2	3 3-1/2	5	3-1/2 4	2-1/2	3-1/2 4	2-1/4 3

^{*}For locating position of dimension H.

For larger sizes the following relationships are suggested as guides for establishing \boldsymbol{A} and $\boldsymbol{B}_{\!\!\boldsymbol{c}}$

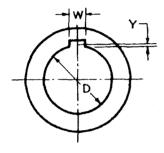
$$A = 1.8 H$$

B = 1.2 H

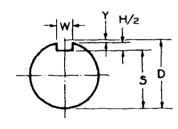
All dimensions given in inches,

5. DEPTH CONTROL FORMULAS

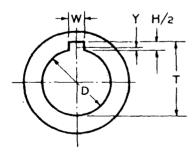
CHORDAL HEIGHT



DEPTH OF SHAFT KEYSEAT



DEPTH OF HUB KEYSEAT



The chordal height Y is determined from the following formula:

$$Y = \frac{D - \sqrt{D^2 - W^2}}{2}$$

The distance from the bottom of the shaft keyseat to the opposite side of the shaft is specified by dimension S. The following formula may be used for calculating this dimension:

$$S = D - Y - \frac{H}{2} = \frac{D - H + \sqrt{D^2 - W^2}}{2}$$

Tabulated values of S for specific shaft diameters are given in Table 3.

The distance from the bottom of the hub keyseat to the opposite side of the hub bore is specified by dimension T. For taper keyseats, Tis measured at the deeper end. The following formula may be used for calculating this dimension:

$$T = D - Y + \frac{H}{2} + C = \frac{D + H + \sqrt{D^2 - W^2}}{2} + C$$

Tabulated values of T for parallel and taper keyseats for specific bores are given in Table 3.

Symbols

C = Allowance

+ 0.005 inch clearance for parallel keys

- 0.020 inch interference for taper keys

D = Nominal shaft or bore diameter, inches

H = Nominal key height, inches

W = Nominal key width, inches

Y = Chordal height, inches





USA STANDARD DEPTH CONTROL VALUES Table 3 Values for S and T

	Table 3 Values for S and T						
Nominal Shaft Diameter	-) \$	(() i	
	Parallel	and Taper	Para	allel	T	aper	
	Square	Rectangular	Square	Rectangular	Square	Rectangular	
	S	5	T	T	T	T	
1/2	0.430	0.445	0.560	0.544	0.535	0.519	
9/16	0.493	0.509	0.623	0.607	0.598	0.582	
5/8	0.517	0.548	0.709	0.678	0.684	0.653	
11/16 3/4	0.581 0.644	0.612 0.676	0.773 0.837	0.742 0.806	0.748	0.717	
		0.676	0.837	0.806	0.812	0.781	
13/16	0.708	0.739	0.900	0.869	0.875	0.844	
7/8	0.771	0.802	0.964	0.932	0.939	0.907	
15/16	0.796	0.827	1.051	1.019	1.026	0.994	
1	0.859	0.890	1.114	1.083	1.089	1.058	
1-1/16	0.923	0.954	1.178	1.146	1.153	1.121	
1-1/8	0.986	1.017	1.241	1.210	1.216	1.185	
1-3/16	1.049	1.080	1.304	1.273	1.279	1.248	
1-1/4	1.112	1.144	1.367	1.336	1.342	1.311	
1-5/16	1.137	1.169	1.455	1.424	1.430	1.399	
1-3/8	1.201	1.232	1.518	1.487	1.493	1.462	
1-7/16	1.225	1.288	1.605	1.543	1.580	1.518	
1-1/2	1.289	1.351	1,669	1.606	1.644	1,581	
1-9/16	1.352	1.415	1.732	1.670	1.707	1.645	
1-5/8	1,416	1.478	1.796	1.733	1.771	1.708	
1-11/16	1.479	1.541	1.859	1.796	1.834	1.771	
1-3/4	1.>42	1.605	1.922	1.860	1.897	1.835	
1-13/16	1.527	1.590	2.032	1.970	2.007	1.945	
1-7/8	1.591	1.654	2.096	2.034	2.071	2.009	
1-15/16	1.655	1.717	2.160	2.097	2.135	2.072	
2	1.718	1.781	2.223	2.161	2.198	2.136	
2-1/16	1.782	1.844	2.287	2.224	2.262	2.199	
2-1/8	1.845	1.908	2.350	2.288	2.325	2.263	
2-3/16	1.909	1.971	2.414	2.351	2.389	2.326	
2-1/4	1.972	2.034	2.477	2.414	2.452	2.389	
2-5/16	1.957	2.051	2.587	2.493	2.562	2.468	
2-3/8	2.021	2.114	2.651	2.557	2.626	2.532	
2-7/16	2.084	2.178	2.714	2.621	2.689	2.596	
2-1/2	2.148	2.242	2.778	2.684	2,753	2.659	
2-9/16	2.211	2.305	2.841	2.748	2.816	2.723	
2-5/8	2.275	2.369	2.905	2.811	2.880	2.786	
2-11/16	2.338	2.432	2.968	2.874	2.943	2.849	
2-3/4	2.402	2.495	3.032	2.938	3.007	2.913	
2-13/16	2.387	2.512	3.142	3.017	3.117	2.992	
2-7/8	2.450	2.575	3.205	3.080	3.180	3.055	
2-15/16	2.514	2.639	3.269	3.144	3.244	3.119	
3	2,577	2,702	3.332	3.207	3.307	3.182	
3-1/16	2.641	2.766	3.396	3.271	3.371	3.246	
3-1/8	2.704	2.829	3.459	3.334	3.434	3.309	

 $^{{\}cal S}$ and ${\cal T}$ Values are calculated from the formulas on page 5. See Tables 4 and 5 for telerances. All dimensions given in inches.



Table 3 Values for S and T = (Continued)

Nominal St. (
Shaft Diameter		
Parallel and Taper Parallel	Taper	
Square Rectangular Square Rectangular Square	Rectangular	
S S T T T	T	
	3.373 3.436	
3-5/16 2.816 2.941 3.696 3.571 3.671	3.546	
3-3/8 2.880 3.005 3.760 3.635 3.735 3-7/16 2.943 3.068 3.823 3.698 3.798	3.610	
	3.673	
3-1/2 3.007 3.132 3.887 3.762 3.862	3.737	
3-9/16 3.070 3.195 3.950 3.825 3.925	3.800	
3-5/8 3.134 3.259 4.014 3.889 3.989	3.864	
3-11/16 3.197 3.322 4.077 3.952 4.052	3.927	
3-3/4 3.261 3.386 4.141 4.016 4.116	3.991	
3-13/16 3.246 3.371 4.251 4.126 4.226	4.101	
3-7/8 3.309 3.434 4.314 4.189 4.289	4.164	
3-15/16 3.373 3.498 4.378 4.253 4.353	4.228	
4 3.436 3.561 4.441 4.316 4.416	4.291	
4-3/16 3.627 3.752 4.632 4.507 4.607	4.482	
4-1/4 3.690 3.815 4.695 4.570 4.670	4.545	
4-3/8 3.817 3.942 4.822 4.697 4.797	4.672	
4-7/16 3.880 4.005 4.885 4.760 4.860	4.735	
4-1/2 3.944 4.069 4.949 4.824 4.924	4.799	
4-3/4 4.041 4.229 5.296 5.109 5.271	5.084	
4-7/8 4.169 4.356 5.424 5.236 5.399	5.211	
4-15/16 4.232 4.422 5.487 5.300 5.462	5.275	
5 4.296 4.483 5.551 5.363 5.526	5.338	
5-3/16 4.486 4.674 5.741 5.554 5.716	5,529	
5-1/4 4.550 4.737 5.805 5.617 5.780	5.592	
5-7/16 4.740 4.927 5.995 5.807 5.970	5.782	
5-1/2 4.803 4.991 6.058 5.871 6.033	5.846	
5-3/4 4.900 5.150 6.405 6.155 6.380	6.130	
5-15/16 5.091 5.341 6.596 6.346 6.571	6.321	
6 5.155 5.405 6.660 6.410 6.635	6.385	
6-1/4 5.409 5.659 6.914 6.664 6.889	L .	
6-1/2 5.662 5.912 7.167 6.917 7.142	6.892	
6-3/4 5.760 *5.885 7.515 *7.390 7.490	*7.365	
7 6.014 *6.139 7.769 *7.644 7.744 7-1/4 6.268 *6.393 8.023 *7.898 7.998		
7-1/4 6.521 6.646 8.276 *8.151 8.251		
7-3/4 6.619 6.869 8.624 8.374 8.599		
8 6.873 7.123 8.878 8.628 8.853		
9 7.887 8.137 9.892 9.642 9.867		
10 8.591 8.966 11.096 10.721 11.071		
11 9.606 9.981 12.111 11.736 12.086		
12 10.309 10.809 13.314 12.814 13.289	12.789	
13 11.325 11.825 14.330 13.830 14.305		
14 12.028 12.528 15.533 15.033 15.508	Transfer of the contract of th	
15 13.043 13.543 16.548 16.048 16.523	16.023	

S and T Values are calculated from formulas on page 5. See Tables 4 and 5 for tolerances. All dimensions given in inches. $\pm 1-3/4 \times 1-1/2$ key.

6. KEY-KEYSEAT ASSEMBLIES

- 6.1 Class 1 A clearance or metal to metal side fit obtained by using bar stock keys and keyseat tolerances as shown in Table 4—a relatively free fit. This fit applies only to parallel keys.
- 6.2 Class 2 A side fit (possible interference or clearance) obtained by using keystock and keyseat tolerances as shown in Table 5—a relatively tight fit.
- 6.3 Class 3 A third type of fit not tabulated is an interference side fit. Since the degree of interference is not readily standardized for these applications, no specific values are included. However, it is suggested that the top and bottom fit range shown for parallel keys in Table 5 be used.

Table 4 Class 1 - Fit for Parallel Keys

Туре	KEY	WIDTH		SIDE FIT			TOP AND B	OTTOM FIT	
of	Over	To (Incl)	Width T	olerance	Fit	Е	epth Toleran		Fit
Key			Key	Keyseat	Range*	Key	Shaft Keyseat	Hub Keyseat	Range*
	_	1/2	+0.000 -0.002	+0.002 -0.000	0.004 CL 0.000	+0.000 -0.002	+0.000 -0.015	+0.010 -0.000	0.032 CL 0.005 CL
	1/2	3/4	+0.000 -0.002	+0.003 -0.000	0.005 CL 0.000	+0.000 -0.002	+0.000 -0.015	+0.010 -0.000	0.032 CL 0.005 CL
	3/4	1	+0.000 -0.003	+0.003 -0.000	0.006 CL 0.000	+0.000 -0.003	+0.000 -0.015	+0.010 -0.000	0.033 CL 0.005 CL
Square	1	1-1/2	+0.000 -0.003	+0.004 -0.000	0.007 CL 0.000	+0.000 -0.003	+0.000 -0.015	+0.010 -0,000	0.033 CL 0.005 CL
	1-1/2	2-1/2	+0.000 -0.004	+0.004 -0.000	0.008 CL 0.000	+0.000 -0.004	+0.000 -0.015	+0.010 -0.000	0.034 CL 0.005 CL
	2-1/2	3-1/2	+0.000 -0.006	+0.004 -0.000	0.010 CL 0.000	+0.000 -0.006	+0.000 -0.015	+0.010 -0.000	0.036 CL 0.005 CL
	_	1/2	+0.000 -0.003	+0.002 -0.000	0.005 CL 0.000	+0.000 -0.003	+0.000 -0.015	+0.010 -0.000	0.033 CL 0.005 CL
	1/2	3/4	+0.000 -0.003	+0.003 -0.000	0.006 CL 0.000	+0.000 -0.003	+0.000 0.015	+0.010 -0.000	0.033 CL 0.005 CL
	3/4	1	+0.000 -0.004	+0.003 -0.000	0.007 CL 0.000	+0.000 -0.004	+0.000 -0.015	+0.010 -0.000	0.034 CL 0.005 CL
Rec-	1	1-1/2	+0.000 -0.004	+0.004 -0.000	0.008 CL 0.000	+0.000 -0.004	+0.000 -0.015	+0.010 -0.000	0.034 CL 0.005 CL
tangular	1-1/2	3	+0.000 -0.005	+0.004 -0.000	0.009 CL 0.000	+0.000 -0.005	+0,000 -0.015	+0.010 -0.000	0.035 CL 0.005 CL
	3	4	+0.000 -0.006	+0.004 -0.000	0.010 CL 0.000	+0.000 -0.006	+0.000 -0.015	+0.010 -0.000	0.036 CL 0.005 CL
	4	6	+0.000 -0.008	+0.004 -0.000	0.012 CL 0.000	+0.000 -0.008	+0.000 -0.015	+0.010 -0.000	0.038 CL 0.005 CL
	6	7	+0.000 -0.013	+0.004 -0.000	0.017 CL 0.000	+0.000 -0.013	+0.000 -0.015	+0.010 -0.000	0.043 CL 0.005 CL

^{*}Limits of variation, CL = Clearance All dimensions given in inches.

Table 5 Class 2 - Fit for Parallel and Taper Keys

Туре	KEY	WIDTH		SIDE FIT			TOP AND B	OTTOM FIT	
of }		Width T	olerance	Fit		epth Toleran	ce	Fit	
Key	Over	To (Incl)	Kev	Keyset	Range*	Key	Shaft Keyseat	Hub Keyseat	Range*
Parallel	_	1-1/4	+0.001 -0.000	+0.002 -0.000	0.002 CL 0.001 INT	+0.001 -0.000	+0.000 -0.015	+0.010 -0.000	0.030 CL 0.004 CL
rataties	1-1/4	3	+0.002 0.000	+0.002 0.000	0.002 CL 0.002 INT	+0.002 -0.000	+0.000 -0.015	+0.010 -0.000	0.030 CL 0.003 CL
Square	3	3-1/2	+0.003 -0.000	+0.002 0.000	0.002 CL 0.003 INT	+0.003 -0.000	+0.000 -0.015	+0.010 -0.000	0.030 CL 0.002 CL
Parallel	-	1-1/4	+0.001 -0.000	+0.002 -0.000	0.002 CL 0.001 INT	+0.005 -0.005	+0.000 -0.015	+0.010 -0.000	0.035 CL 0.000 CL
Rectang-	1-1/4	3	+0.002 -0.000	+0.002 -0.000	0.002 CL 0.002 INT	+0.005 -0.005	+0.000 -0.015	+0.010 -0.000	0.035 CL 0.000 CL
ular	3	7	+0.003 0.000	+0.002 -0.000	0.002 CL 0.003 INT	+0.005 -0.005	+0.000 -0.015	+0.010 -0.000	0.035 CL 0.000 CL
	-	1-1/4	+0.001 -0.000	+0.002 -0.000	0.002 CL 0.001 INT	+0.005 -0.000	+0.000 -0.015	+0.010 -0.000	0.005 CL 0.025 INT
Тарег	1-1/4	3	+0.002 -0.000	+0.002 -0.000	0.002 CL 0.002 INT	+0.005 -0.000	+0.000 -0.015	+0.010 -0.000	0.005 CL 0.025 INT
	3	Δ	+0.003 -0.000	+0.002 -0.000	0.002 CL 0.003 INT	+0.005 -0.000	+0.000 -0.015	+0.010 -0.000	0.005 CL 0.025 INT

^{*}Limits of variation. CL= Clearance; INT = Interference

All dimensions given in inches.

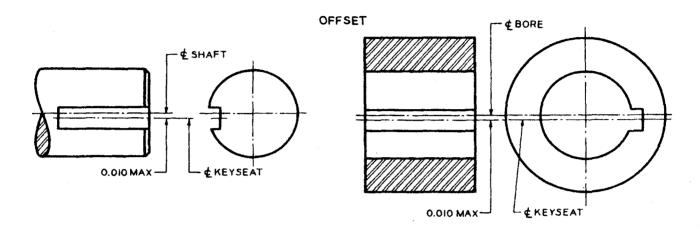
Table 6 Keyseat Tolerances for Electric Motor and Generator Shaft Extensions

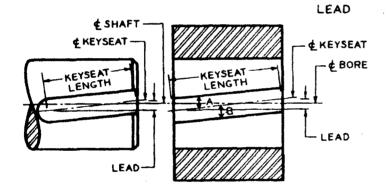
KEYSEA	AT WIDTH		Depth Tolerance	
Over	To (Incl)	Width Tolerance		
_	1/4	+0.001 -0.001	+0.000 -0.015	
1/4	3/4	+0.000 -0.002	+0.000 -0.015	
3/4	, 1-1/4	+0.000 -0.003	+0.000 -0.015	

All dimensions given in inches.

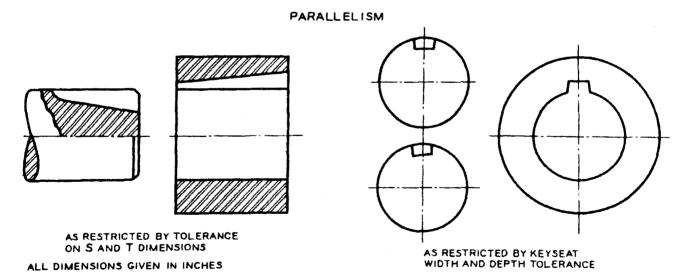
 $[\]Delta$ To (Incl) 3-1/2 Square and 7 Rectangular key widths.

7. ALIGNMENT TOLERANCES





KEYSEA	T LENGTH			
OVER	TO (INCL)	LEAD MAXIMUM		
	4	0.002		
4	10	0.0005 PER INCH		
10		0.005		



8. CHAMFERED KEYS AND FILLETED KEYSEATS

In general practice, chamfered keys and filleted keyseats are not used. However, it is recognized that fillets in keyseats decrease stress concentrations at corners. When used, fillet radii should be as large as possible without causing excessive bearing stresses due to reduced contact area between the key and its mating parts. Keys must be chamfered or rounded to clear fillet radii. Values in Table 7 assume general conditions and should be used only as a guide when critical stresses are encountered.

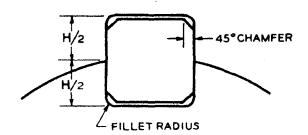


Table 7 Suggested Fillet Radii and Key Chamfer

H/2 KEYSE	AT DEPTH		0
Over	To (Incl)	Fillet Radius	45° Chamfer
1/8 1/4 1/2	1/4 1/2 7/8	1/32 1/16 1/8	3/64 5/64 5/32
7/8 1-1/4 1-3/4	1-1/4 1-3/4 2-1/2	3/16 1/4 3/8	7/32 9/32 13/32

All dimensions given in inches.

9. SET SCREWS FOR USE OVER KEYS

Set screw diameter selections listed below are offered as a guide but their use shall be dependent upon design considerations.

Table 8 Set Screws for Use Over Keys

NOMINAL SHAFT DIAMETER		Nominal Key Width	Set Screw Diameter
Over	To (Incl)		Jet Selew Diameter
5/16 7/16 9/16 7/8	7/16 9/16 7/8 1-1/4	3/32 1/8 3/16 1/4	#10 #10 1/4 5/16
1-1/4 1-3/8 1-3/4 2-1/4	1-3/8 1-3/4 2-1/4 2-3/4	5/16 3/8 1/2 5/8	3/8 3/8 1/2 1/2
2-3/4 3-1/4 3-3/4 4-1/2 5-1/2	3-1/4 3-3/4 4-1/2 5-1/2 6-1/2	3/4 7/8 1 1-1/4 1-1/2	5/8 3/4 3/4 7/8

All dimensions given in inches.

ADOPTION NOTICE 1 16 October 1992 for ASME B17.1-1967 September 15, 1967

ACCEPTANCE NOTICE

ASME B17.1-1967 was adopted on 16 October 1992 and is approved for use by the Department of Defense (DoD). The American Society of Mechanical Engineers has furnished the clearance required by existing regulations. Copies of the document are stocked at the Standardization Documents Order Desk, Bldg. 4D, 700 Robbins Avenue, Philadelphia, PA 19111-5094, for issue to DoD activities only. All other requestors must obtain copies from:

American Society of Mechanical Engineers United Engineering Center 345 East 47th Street New York, NY 10017

Title of Document:

Keys and Keyseats

Date of Specific Issue Adopted:

September 15, 1967, Reaffirmed 1989

Releasing Non-Government Standards Body:

American Society of Mechanical Engineers

Custodians:

Army - AR

Navy - YD Air Force - 99

Review Activity: DLA – IS Military Coordinating Activity:

Navy - YD

(Project 5340-2111)

FSC 5340





J00038