# Ferrous Pipe Plugs, Bushings, and Locknuts With Pipe Threads

AN AMERICAN NATIONAL STANDARD



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AN AMERICAN NATIONAL STANDARD



The American Society of Mechanical Engineers

Two Park Avenue • New York, NY • 10016 USA

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### FOREWORD

In 1921, the American Engineering Standards Committee [later the American Standards Association (ASA)] organized Sectional Committee B16 to unify and develop standards for pipe flanges and fittings. Cosponsors of the B16 Committee were The American Society of Mechanical Engineers (ASME), the Heating and Piping Contractors National Association [now Mechanical Contractors Association of America (MCAA)], and the Manufacturers Standardization Society of the Valve and Fitting Industry (MSS). Cosponsors were later designated as Co-Secretariat organizations.

Threaded fittings were also included in the scope of the B16 Committee, and Subcommittee 2 (now Subcommittee B) was made responsible for threaded fittings other than steel.

The American Standard for Pipe Plugs was originally published as ASA B16e2-1936. It had been developed by Subcommittee 2 from material assembled and published by MSS as a standard practice in April 1929.

From the beginning, pipe plugs have been made with the American Standard Taper Pipe Thread but the design of the square heads necessary to screw them into fittings or any tapped hole has been a matter of manufacturers' and users' individual design. However, the use of pipe plugs soon broadened, and they came to be employed in many other service applications, e.g., automotive and industrial machinery. Accordingly, it was natural for the dimensions of the square heads and the sockets to conform to standard open wrench sizes and to maximum dimensions of standard hotrolled steel bars.

Pipe bushing and locknut dimensions were originally included in ASA B16c-1939, American Standard for 150 lb Malleable Iron Screwed Fittings. ASA B16c-1939 was adapted from information assembled and published by MSS in 1929.

In response to a demand for inclusion under one cover, B16.14-1943, American Standard for Ferrous Plugs, Bushings, Locknuts, and Caps, was developed from ASA B16c-1939 and ASA B16d-1941, American Standard for Cast Iron Screwed Fittings, 125 lb and 250 lb, and approved by ASA in October 1943. The section of B16.14 covering pipe caps was later removed because of differences in pressure ratings between caps made of cast iron and malleable iron.

This Standard was revised in 1948 and was approved by letter ballot vote of the Sectional Committee and, following the approval of the sponsor bodies, it was presented to ASA, now the American National Standards Institute (ANSI), with recommendations for approval as an American Standard. This approval and designation was given on April 6, 1949.

Subcommittee 2 began a review of the document in 1963 and completed its work in 1964. The Sectional Committee approved a number of minor changes in format and wording. Following approval by the sponsor organizations, ANSI approval was granted on November 12, 1965.

In 1970, a review was initiated by Subcommittee 2 that resulted in the proposal to revise the document in several minor areas and update referenced standards. Following approvals by the Standards Committee and Co-Secretariat, ANSI granted approval on November 1, 1971.

In 1975, review was again initiated by Subcommittee B (formerly Subcommittee 2), and it was determined that the only significant changes needed were the updating of referenced standards and the addition of metric equivalents. Approvals were granted by the Standards Committee, Co-Secretariat, and ANSI, the latter on February 4, 1977.

In 1982, the American National Standards Committee was reorganized as an ASME Committee operating under procedures accredited by ANSI. That same year, Subcommittee B once again began review of the Standard. The only changes made were the dates of the referenced standards. After approval by the Standards Committee and ASME, final approval was granted by ANSI in July 1983.

In the 1991 edition of B16.14, U.S. customary units were established as the standard and updates were made to the referenced standards and minimum specifications for steel fittings. Following approval by the Standards Committee and ASME, approval as an American National Standard was given by ANSI on January 4, 1991, with the new designation ASME B16.14-1991.

In the 2010 edition of B16.14, metric dimensions became the primary units, and inch dimensions were incorporated into the Standard as secondary units and shown in parentheses. Following approval by the Standards Committee and the ASME Board, the revision to the 1991 edition of the Standard was approved as an American National Standard by ANSI on April 1, 2010, and designated as ASME B16.14-2010.

In this 2013 edition, section 8 was revised and now requires threads and gaging practices to be as per, and identical with, ASME B1.20.1 and other B16 standards. Following approval by the ASME B16 Standards Committee, approval as an American National Standard was given by ANSI on July 29, 2013, with the new designation ASME B16.14-2013.

## ASME B16 COMMITTEE Standardization of Valves, Flanges, Fittings, and Gaskets

(The following is the roster of the Committee at the time of approval of this Standard.)

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## **CORRESPONDENCE WITH THE B16 COMMITTEE**

**General.** ASME Standards are developed and maintained with the intent to represent the consensus of concerned interests. As such, users of this Standard may interact with the Committee by requesting interpretations, proposing revisions, and attending Committee meetings. Correspondence should be addressed to:

Secretary, B16 Standards Committee The American Society of Mechanical Engineers Two Park Avenue New York, NY 10016-5990

As an alternative, inquiries may be submitted via e-mail to SecretaryB16@asme.org.

**Proposing Revisions.** Revisions are made periodically to the Standard to incorporate changes that appear necessary or desirable, as demonstrated by the experience gained from the application of the Standard. Approved revisions will be published periodically.

The Committee welcomes proposals for revisions to this Standard. Such proposals should be as specific as possible, citing the paragraph number(s), the proposed wording, and a detailed description of the reasons for the proposal, including any pertinent documentation.

**Interpretations.** Upon request, the B16 Committee will render an interpretation of any requirement of the Standard. Interpretations can only be rendered in response to a written request sent to the Secretary of the B16 Standards Committee.

The request for interpretation should be clear and unambiguous. It is further recommended that the inquirer submit his/her request in the following format:

Subject:	Cite the applicable paragraph number(s) and the topic of the inquiry.
Edition:	Cite the applicable edition of the Standard for which the interpretation is
	being requested.
Question:	Phrase the question as a request for an interpretation of a specific requirement
	suitable for general understanding and use, not as a request for an approval
	of a proprietary design or situation. The inquirer may also include any plans
	or drawings that are necessary to explain the question; however, they should
	not contain proprietary names or information.

Requests that are not in this format will be rewritten in this format by the Committee prior to being answered, which may inadvertently change the intent of the original request.

ASME procedures provide for reconsideration of any interpretation when or if additional information that might affect an interpretation is available. Further, persons aggrieved by an interpretation may appeal to the cognizant ASME Committee or Subcommittee. ASME does not "approve," "certify," "rate," or "endorse" any item, construction, proprietary device, or activity.

**Attending Committee Meetings.** The B16 Standards Committee regularly holds meetings that are open to the public. Persons wishing to attend any meeting should contact the Secretary of the B16 Standards Committee.

## ASME B16.14-2013 SUMMARY OF CHANGES

Following approval by the B16 Committee and ASME, and after public review, ASME B16.14-2013 was approved by the American National Standards Institute on July 29, 2013.

ASME B16.14-2013 includes editorial changes, revisions, and corrections, which are identified by a margin designator, **(13)**, placed next to the affected area.

Page	Location	Change
2	8	Revised

## FERROUS PIPE PLUGS, BUSHINGS, AND LOCKNUTS WITH PIPE THREADS

#### 1 SCOPE

This Standard covers the following:

- (*a*) pressure–temperature ratings
- (b) size
- (c) marking
- (d) materials
- (e) dimensions and tolerances
- (f) threading
- (g) pattern taper

#### 2 GENERAL

#### 2.1 References

Standards and specifications adopted by reference are shown in Mandatory Appendix II. It is not considered practical to identify the edition of each referenced standard and specification in the text. Instead, the editions of the referenced standards and specifications are listed in Mandatory Appendix II.

#### 2.2 Quality Systems

Requirements relating to the product manufacturer's quality system programs are described in Nonmandatory Appendix A.

#### 2.3 Relevant Units

This Standard states values in both SI (Metric) units and U.S. Customary units. These systems of units are to be regarded separately as standard. Within the text, the U.S. Customary units are shown in parentheses or in separate tables that appear in Mandatory Appendix I. The values stated in each system are not exact equivalents; therefore, it is required that each system of units be used independently of the other. Combining values from the two systems constitutes nonconformance with the Standard.

#### 2.4 Service Conditions

Criteria for selection of materials suitable for particular fluid service are not within the scope of this Standard.

#### 2.5 Convention

For determining conformance with this Standard, the convention for fixing significant digits where limits (maximum and minimum values) are specified shall be as defined in ASTM E29. This requires that an observed or calculated value be rounded off to the nearest unit in the last right-hand digit used for expressing the limit. Decimal values and tolerances do not imply a particular method of measurement.

#### 2.6 Denotation

**2.6.1 Pressure Rating Designation.** Class followed by a dimensionless number is the designation for pressure–temperature ratings.

**2.6.2 Size.** Nominal pipe size (NPS) followed by a dimensionless number is the designation for nominal flange or flange fitting size. NPS is related to the reference nominal diameter, DN, used in international standards. The relationship is, typically, as follows:

NPS	DN
1	25
$1\frac{1}{4}$	32
$1\frac{1}{2}$	40
2	50
$2\frac{1}{2}$	65
3	80
$3\frac{1}{2}$	90
4	100

For NPS  $\geq$  4, the related DN = 25 × (NPS)

#### **3 PRESSURE-TEMPERATURE RATINGS**

(*a*) Pressure–temperature ratings for plugs and bushings shall be in accordance with ASME B16.4 if made of cast iron, or ASME B16.3 if made of malleable iron, ductile iron, or steel. Use of cored plugs or hexagon head bushings should be limited to Class 125 cast iron and Class 150 malleable iron threaded fittings. Solid plugs and face bushings are recommended for use with Class 250 cast iron fittings and Class 300 malleable iron fittings.

(b) Locknuts are not pressure-temperature rated.

#### **4 NOMINAL PIPE SIZE**

As applied in this Standard, the use of the phrase "nominal pipe size" or the designation NPS followed by a dimensionless number is for identifying the end connection of fittings. The number is not necessarily the same as the fitting inside diameter.

#### 5 MARKING

Each fitting shall be marked for identification with the manufacturer's name or trademark, except where a marking is impractical.

#### **6 MATERIALS**

(*a*) These fittings are furnished in gray iron, malleable iron, ductile iron, or steel as indicated in the individual tables.

(*b*) The chemical and mechanical properties of cast material shall equal or exceed those properties listed in ASTM A126, as applicable. Steel castings, forgings, or machined bar properties shall correspond to the comparable ASTM casting material specifications. The fittings manufacturer shall be prepared to certify conformance based on test data.

#### 7 DIMENSIONS AND TOLERANCES

(*a*) Dimensions in metric units are given in Tables 1 through 5 for various types of fittings. Corresponding U.S. Customary values are given in Tables I-1 through I-5 in Mandatory Appendix I.

(*b*) At no point in the component wall shall the metal thickness be less than 90% of the values listed in the tables.

#### (13) 8 THREADS

#### 8.1 Thread Form

All threads shall be in accordance with ASME B1.20.1.

**8.1.1 Countersinks and Chamfers.** All internal taper pipe threads shall be countersunk or chamfered a distance of not less than one-half the pitch of the thread at an angle of approximately 45 deg with the axis of the thread. External taper pipe threads shall be chamfered at an angle between 30 deg and 45 deg with the axis,

for easier entrance in making a joint and protection of the thread. Countersinking and chamfering shall be concentric with the threads. The length of threads specified in all tables shall be measured to include the countersink or chamfer.

**8.1.2 Alignment.** The maximum allowable variation in the alignment of threads of all openings shall be 5.0 mm/m (0.06 in./ft).

**8.1.3 Internal Threading.** All fittings with internal threads, except locknuts, shall be threaded with ASME B1.20.1 NPT threads. The reference point for gaging is the starting end of the fitting, provided the chamfer does not exceed the major diameter of the internal thread. When a chamfer on the internal thread exceeds this limit, the reference point becomes the last thread scratch on the chamfer cone.

**8.1.4 External Threading.** All externally threaded fittings shall be threaded with ASME B1.20.1 NPT threads. The reference point for gaging is the end of the thread, provided the chamfer is not smaller than the minor diameter of the external thread. When a chamfer on the external thread exceeds this limit, the reference point becomes the last thread scratch on the chamfer cone.

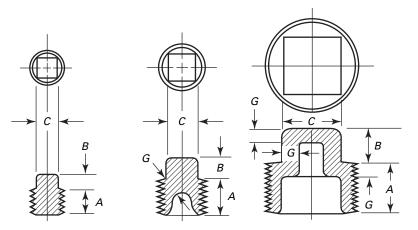
**8.1.5 Locknuts.** Locknuts shall be threaded with ASME B1.20.1 NPSL threads.

#### 8.2 Gaging Tolerances

For taper pipe threads, the variation in threading shall be limited to one turn large or small from the gaging notch on the plug or the gaging face of the ring when using working gages.

#### 9 PATTERN TAPER

Plug squares or hexagons, raised or countersunk, may have opposite sides tapered a maximum of 4 deg total.



#### Table 1 Dimensions of Square Head Plugs

(a) Solid Plugs

(b) Cored Plugs

NPS [Notes (1), (2)]	Minimum Thread Length, A	Minimum Height of Square, <i>B</i>	Nominal Width Across Flats, <i>C</i> [Note (3)]	Nominal Metal Thickness, <i>G</i> [Note (4)]
1/8	9.4	6.1	9/ <sub>32</sub> 3/ <sub>8</sub>	
1/4	11.2	7.1	3/8	
3/8	12.2	7.9	7/16	
$\frac{1}{4}$ $\frac{3}{8}$ $\frac{1}{2}$	14.2	9.7	9/16	4.1
3/4	16.0	11.2	5/8	4.6
1	19.1	12.7	13/16 15/16	5.1
$1\frac{1}{4}$	20.3	14.2	15/16	5.6
$1^{1}/_{2}$	21.1	15.8	$1\frac{1}{8}$	6.1
2	22.4	17.3	15/16	6.6
2 <sup>1</sup> / <sub>2</sub>	27.2	18.8	$1^{1/2}$	7.4
3	28.7	20.3	$1^{11}/_{16}$	7.9
$3^{1}/_{2}$	30.0	21.8	17/8	8.6

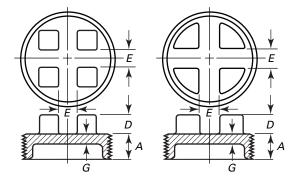
GENERAL NOTE: All dimensions are given in millimeters except nominal width across flats, which are given in inches. NOTES:

(1) Solid plugs are provided in NPS  $\frac{1}{8}$  to NPS  $\frac{3}{2}$ , incl.; cored plugs are provided in NPS  $\frac{1}{2}$  to NPS  $\frac{3}{2}$ , incl.

(2) For NPS 4 and larger, slotted or bar pattern plugs are provided (see Table 2).

(3) These dimensions are the nominal width across flats as given in Table 1 of ASME B18.2.1. Square head plugs are designed to fit these wrenches.

(4) Cored plugs have metal thickness at all points equal to dimension G, except at the end of the thread. For tolerance, see para. 7(b).



#### Table 2 Dimensions of Bar or Slotted Head Plugs

**Optional Designs** 

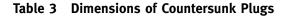
NPS	Minimum Thread Length, <i>A</i>	Minimum Height of Lug, D	Minimum Distance Between Lugs, <i>E</i>	Metal Thickness, <i>G</i> [Note (1)]
4	31.0	25.4	22.4	9.4
5	33.3	25.4	22.4	11.7
6	35.6	31.8	31.8	13.2
8	39.9	35.1	38.1	16.8

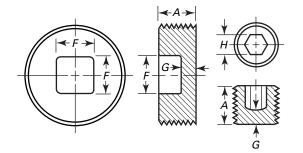
#### **GENERAL NOTES:**

- (a) All dimensions are given in millimeters.
- (b) For NPS  $3^1\!\!/_2$  and smaller, square head plugs are provided (see Table 1).

#### NOTE:

Cored plugs have metal thickness at all points equal to dimension *G*, except at the end of the thread. For tolerance, see para. 7(b).





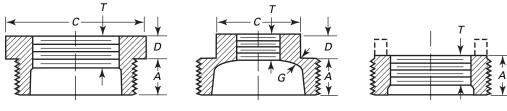
NPS	Minimum Thread Length, <i>A</i>	Nominal Size of Square Socket, <i>F</i> [Note (1)]	Size of Hexagon, <i>H</i> [Note (2)]	Metal Thickness, <i>G</i> [Note (3)]
$\frac{1}{8}$	9.4	•••	$\frac{3}{16}$	1.5
$\frac{1}{4}$ $\frac{3}{8}$ $\frac{1}{2}$ $\frac{3}{4}$	11.2	•••	1/4	2.3
/8	12.2	•••	5/16 3/8	3.3
1/2	14.2	3/8	1/8	4.1
3/4	16.0	1/2	9/16	4.6
1	19.1	$\frac{1}{2}$	5/8	5.1
$1^{1}/_{4}$	20.3	3/4		5.6
$1^{1}/_{2}$	21.1	3/4		6.1
2	22.4	7/8		6.6
2 <sup>1</sup> / <sub>2</sub>	27.2	$1^{1}/_{8}$		7.4
3	28.7	13/8		7.9
$3^{1}/_{2}$	30.0	$1^{1/2}$		8.6
4	31.0	2		9.4

GENERAL NOTE: All dimensions are given in millimeters except nominal sizes of square sockets and sizes of hexagons, which are given in inches.

NOTES:

- (1) Square socket within countersunk pattern shall have dimensions to fit commercial square bars of sizes indicated.
- (2) Hexagon socket within countersunk pattern shall have dimensions to fit regular wrenches used with hexagon socket set screws.
- (3) For metal thickness tolerance, see para. 7(b).

#### Table 4 Dimensions of Outside Head, Inside Head, and Face Bushings



(a) Outside Head

(b) Inside Head

(c) Face

NPS	Minimum Length of External Thread,	Minimum Length of	Minimum Height	Minimum Width <i>C</i> [Note (3)]		Nominal Metal Thickness,
[Note (1)]	<i>A</i> [Note (2)]	Internal Thread, <i>T</i>	of Head, D	Outside	Inside	G [Note (4)]
$\frac{1}{4} \times \frac{1}{8}$	11.2	6.6 [Note (5)]	3.6	16.3 [Note (6)]		
$\frac{3}{8} \times \frac{1}{4}$	12.2	10.2 [Note (5)]	4.1	17.3 [Note (6)]		
$\frac{3}{8} \times \frac{1}{8}$	12.2	6.4	4.1	17.3 [Note (6)]		
$\frac{1}{2} \times \frac{3}{8}$	14.2	10.4 [Note (5)]	4.8	22.1 [Note (6)]		
$\frac{1}{2} \times \frac{1}{4}$	14.2	8.1	4.8	22.1 [Note (6)]		
$\begin{array}{c} 74 \times 78 \\ 3/8 \times 1/4 \\ 3/8 \times 1/8 \\ 1/2 \times 3/8 \\ 1/2 \times 1/4 \\ 1/2 \times 1/4 \\ 1/2 \times 1/8 \end{array}$	14.2	6.4	4.8	22.1 [Note (6)]	•••	
$3/4 \times 1/2$ $3/4 \times 3/8$ $3/4 \times 1/4$ $3/4 \times 1/8$	16.0	13.5 [Note (5)]	5.6	29.2 [Note (6)]		
$\frac{3}{4} \times \frac{3}{8}$	16.0	9.1	5.6	29.2 [Note (6)]		
$\frac{3}{4} \times \frac{1}{4}$	16.0	8.1	5.6	29.2 [Note (6)]		
$\frac{3}{4} \times \frac{1}{8}$	16.0	6.4	5.6	29.2 [Note (6)]		
$1 \times \frac{3}{4}$	19.1	12.7	6.4	36.1 [Note (6)]		
$1 \times \frac{1}{2}$	19.1	10.9	6.4	36.1 [Note (6)]		
$1 \times \frac{3}{8}$	19.1	9.1	7.6		28.5	
$1 \times \frac{1}{4}$	19.1	8.1	7.6		28.5	
$1 \times \frac{1}{8}$	19.1	6.4	7.6		28.5	
$1\frac{1}{4} \times 1$	20.3	14.7	7.1	44.7		
$1^{1}/_{4} \times {}^{3}/_{4}$	20.3	12.7	7.1	44.7		
$1^{1}_{4} \times 1^{1}_{2}$ $1^{1}_{4} \times 3^{3}_{8}$	20.3	10.9	8.6		34.0	4.8
$1^{1}/_{4} \times \frac{3}{8}$	20.3	9.1	8.6		28.5	4.8
$1^{1}/_{4} \times \frac{1}{4}$	20.3	8.1	8.6	•••	28.5	4.8
$1^{1}/_{2} \times 1^{1}/_{4}$	21.1	18.0 [Note (5)]	7.9	50.8		
$1^{1}/_{2} \times 1$	21.1	14.7	7.9	50.8		•••
$1^{1}/_{2} \times {}^{3}/_{4}$	21.1	12.7	9.4		41.4	5.1
$1\frac{1}{2} \times \frac{1}{2}$	21.1	10.9	9.4		34.0	5.1
$1^{1}/_{2} \times \frac{3}{8}$	21.1	9.1	9.4		28.5	5.1
$1^{1/2} \times 1^{1/4}$	21.1	8.1	9.4		28.5	5.1
$2 \times 1^{1/2}$	22.4	17.8	8.6	63.0		
$2 \times 1^{1}/_{4}$	22.4	17.0	8.6	63.0		• • •
2 × 1	22.4	14.7	10.4	•••	49.5	5.6
$2 \times \frac{3}{4}$	22.4	12.7	10.4		41.4	5.6
$2 \times \frac{1}{2}$	22.4	10.9	10.4	•••	34.0	5.6
$2 \times \frac{3}{8}$	22.4	9.1	10.4		28.5	5.6
$2 \times \frac{1}{4}$	22.4	8.1	10.4		28.5	5.6
$2^{1/2} \times 2$	27.2	19.1	9.4	75.7		
$2^{1}/_{2} \times 1^{1}/_{2}$	27.2	17.8	11.2	68.1		•••
$2\frac{1}{2} \times 1\frac{1}{4}$	27.2	17.0	11.2	•••	60.7	6.1
$2^{1/2} \times 1$	27.2	14.7	11.2	•••	49.5	6.1
$1\frac{1}{2} \times \frac{3}{4}$	27.2	12.7	11.2	•••	41.4	6.1
$2^{1/2} \times \frac{1}{2}$	27.2	10.9	11.2		34.0	6.1

	Minimum Length of External Thread,	Minimum Length of	Minimum Height	Minimum Width C	-	Nominal Meta Thickness,
NPS	Α	Internal Thread,	of Head,	[Note (3	3)]	G
[Note (1)]	[Note (2)]	Т	D	Outside	Inside	[Note (4)]
$3 \times 2^{1/2}$	28.7	23.4	10.2	98.0		
3 × 2	28.7	19.1	12.2	83.3		
$3 \times 1^{1/2}$	28.7	17.8	12.2		68.1	6.6
$3 \times 1^{1/4}$	28.7	17.0	12.2		60.7	6.6
3 × 1	28.7	14.7	12.2		49.5	6.6
$3 \times \frac{3}{4}$	28.7	12.7	12.2		41.4	6.6
$3 \times \frac{3}{4}$ $3 \times \frac{1}{2}$	28.7	10.9	12.2		34.0	6.6
$3^{1}/_{2} \times 3$	30.0	24.9	10.9	117.3		
$3^{1}/_{2} \times 2^{1}/_{2}$	30.0	23.4	13.2	98.0		
$3^{1}/_{2} \times 2$	30.0	19.1	13.2		83.3	7.1
$3^{1}/_{2} \times 1^{1}/_{2}$	30.0	17.8	13.2		68.1	7.1
$3\frac{1}{2} \times 1\frac{1}{4}$	30.0	17.0	13.2		60.7	7.1
$3^{1}/_{2} \times 1$	30.0	14.7	13.2	•••	49.5	7.1
$4 \times 3^{1/_{2}}$	31.0	26.2	12.7	132.1		
4 × 3	31.0	24.9	12.7	117.3		
$4 \times 2^{1/2}$	31.0	23.4	15.2		98.0	7.9
4 × 2	31.0	19.1	15.2		83.3	7.9
$4 \times 1^{1/2}$	31.0	17.8	15.2		68.1	7.9
$4 \times 1^{1/_{4}}$	31.0	17.0	15.2		60.7	7.9
4 × 1	31.0	14.7	15.2	•••	49.5	7.9
5 × 4	33.3	27.4	12.7	147.1		
5 × 3 <sup>1</sup> / <sub>2</sub>	33.3	26.2	15.2	132.1		
5 × 3	33.3	24.9	15.2		117.3	9.7
$5 \times 2^{1/2}$	33.3	23.4	15.2		98.0	9.7
5 × 2	33.3	19.1	15.2	•••	83.3	9.7
6 × 5	35.6	30.0	16.0	179.1		
6 × 4	35.6	27.4	19.1		147.1	10.9
6 × 3 <sup>1</sup> / <sub>2</sub>	35.6	26.2	19.1		132.1	10.9
6 × 3	35.6	24.9	19.1		117.3	10.9
$6 \times 2^{1/2}$	35.6	23.4	19.1		98.0	10.9
6 × 2	35.6	19.1	19.1	•••	83.3	10.9
8 × 6	39.9	32.5	21.1	210.3		
8 × 5	39.9	30.0	21.1		179.1	14.0
8 × 4	39.9	27.4	21.1		147.1	14.0
8 × 3 <sup>1</sup> / <sub>2</sub>	39.9	26.2	21.1		132.1	14.0
8 × 3	39.9	24.9	21.1		117.3	14.0

Table 4 Dimensions of Outside Head, Inside Head, and Face Bushings (Cont'd)

GENERAL NOTES:

(a) All dimensions are given in millimeters.

(b) Cored bushings have minimum metal thickness at all points equal to dimension *G*, except at the end of the thread. NOTES:

(1) Hexagon head or octagon head bushings NPS 2<sup>1</sup>/<sub>2</sub> and smaller reducing one NPS shall not be made of gray cast iron. Other NPS may be made of either cast iron or malleable iron, ductile iron, or steel. Face bushings NPS 2<sup>1</sup>/<sub>2</sub> and smaller shall not be made of gray cast iron. Face bushings NPS 3 and larger reducing one NPS shall not be made of gray cast iron. Face bushings NPS 3 and larger reducing two NPS or more may be made of either cast or malleable iron, ductile iron, or steel.

(2) In the case of outside head bushings, length A includes provisions for imperfect threads (see also section 8).

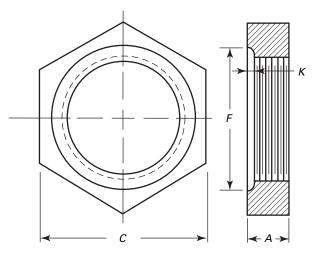
(3) Heads of bushings shall be hexagonal or octagonal, except that on the larger sizes of outside head bushings, the heads may be made round with lugs instead of hexagonal or octagonal.

(4) G is the same as metal thickness for Class 125 Cast Iron Threaded Fittings (ASME B16.4). For tolerance, see para. 7(b).

(5) To provide proper metal thickness, these NPS shall not be cored out to diameters greater than the root diameter of the internal thread. The length of the internal thread may be equal to the minimum dimension, *T*, or greater up to the full length of bushing.

(6) When made of bar stock, the dimensions may be  $\frac{5}{8}$  in.,  $\frac{11}{16}$  in.,  $\frac{7}{8}$  in.,  $\frac{11}{8}$  in., and  $\frac{17}{16}$  in., respectively, in order to use regular bar stock sizes.

#### Table 5 Dimensions of Locknuts



#### Minimum Width Across Flats,

	Minimum Nominal	C [Note (1)]		Minimum Diameter of Packing Recess,	Depth of Packing Recess,	
NPS	NPS	Thickness, A	Malleable Iron, Ductile Iron, Steel	Cast Iron	<i>F</i> [Note (2)]	<i>K</i> [Note (2)]
<sup>1</sup> / <sub>8</sub>	4.8	17.5 [Note (3)]		12.7	1.0	
<sup>1</sup> / <sub>4</sub>	6.4	21.3 [Note (3)]		16.8	1.5	
3/8	7.1	25.4 [Note (3)]		19.6	1.5	
$\frac{1}{4}$ $\frac{3}{8}$ $\frac{1}{2}$	7.9	30.0 [Note (3)]		24.6	1.5	
3/4	8.6	36.3 [Note (3)]		31.2	1.5	
1	9.7	44.5		38.1	1.5	
$1^{1}/_{4}$	10.7	53.3		47.2	1.5	
$1^{1}/_{2}$	11.9	59.7		53.9	1.5	
2	13.5	73.2		66.8	2.3	
$2^{1}/_{2}$	15.0	88.9	98.0	80.8	2.3	
3	17.0	108.5	117.3	97.5	2.3	
$3^{1}/_{2}$	18.5	122.9	132.1	111.3	3.3	
4	20.3	136.7	147.1	127.0	3.3	

GENERAL NOTE: All dimensions are given in millimeters.

NOTES:

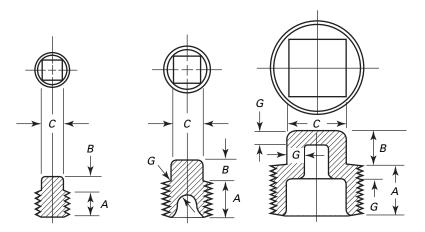
(1) NPS  $3^{1}/_{2}$  and smaller are hexagonal; NPS 4 may be either hexagonal or octagonal.

(2) The packing recess surface may be as cast. When made from bar stock, the recess may be tool finished.
(3) Locknuts in these NPS may be made from bar stock, in which case dimension C may be <sup>11</sup>/<sub>16</sub> in., <sup>7</sup>/<sub>8</sub> in., 1 in., 1<sup>3</sup>/<sub>16</sub> in., and 1<sup>7</sup>/<sub>16</sub> in., respectively, in order to conform with regular hexagon bar stock sizes.

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## MANDATORY APPENDIX I DIMENSIONS OF FITTINGS IN U.S. CUSTOMARY UNITS

Tables I-1 through I-5 are presented on the following pages.



#### Table I-1 Dimensions of Square Head Plugs

(a) Solid Plugs

(b)	Cored	Plugs
-----	-------	-------

NPS [Notes (1), (2)]	Minimum Thread Length, A	Minimum Height of Square, <i>B</i>	Nominal Width Across Flats, <i>C</i> [Note (3)]	Nominal Metal Thickness, <i>G</i> [Note (4)]
1/8	0.37	0.24	9/ <sub>32</sub> 3/8	
1/4	0.44	0.28	3/8	
3/8	0.48	0.31	7/16	
$\frac{1}{4}$ $\frac{3}{8}$ $\frac{1}{2}$	0.56	0.38	9/16	0.16
3/4	0.63	0.44	5/8	0.18
1	0.75	0.50	<sup>13</sup> / <sub>16</sub>	0.20
11/4	0.80	0.56	<sup>15</sup> / <sub>16</sub>	0.22
$1^{1}/_{2}$	0.83	0.62	13/16 15/16 $1^{1}/8$	0.24
2	0.88	0.68	$1^{5}/_{16}$ $1^{1}/_{2}$	0.26
$2^{1}/_{2}$	1.07	0.74	$1^{1}/_{2}$	0.29
3	1.13	0.80	$1^{11}/_{16}$	0.31
$3^{1}/_{2}$	1.18	0.86	17/8	0.34

GENERAL NOTE: Dimensions are in inches.

NOTES:

(1) Solid plugs are provided in NPS  $\frac{1}{8}$  to NPS  $\frac{3}{2}$ , incl.; cored plugs, NPS  $\frac{1}{2}$  to NPS  $\frac{3}{2}$ , incl. (2) For NPS 4 and larger, slotted or bar pattern plugs are provided (see Table I-2).

(3) These dimensions are the nominal width across flats as given in Table 1 of ASME B18.2.1. Square head plugs are designed to fit these wrenches.

(4) Cored plugs have metal thickness at all points equal to dimension G, except at the end of the thread. For tolerance, see para. 7(b).

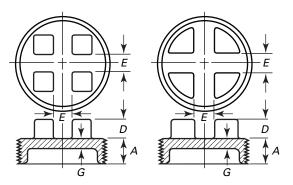


 Table I-2
 Dimensions of Bar or Slotted Head

 Plugs

**Optional Designs** 

NPS	Minimum Thread Length, A	Minimum Height of Lug, D	Minimum Distance Between Lugs, <i>E</i>	Metal Thickness, <i>G</i> [Note (1)]	
4	1.22	1.00	0.88	0.37	
5	1.31	1.00	0.88	0.46	
6	1.40	1.25	1.25	0.52	
8	1.57	1.38	1.50	0.66	

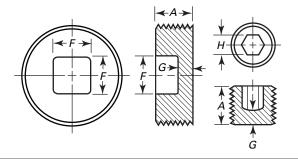
#### GENERAL NOTES:

- (a) Dimensions are given in inches.
- (b) For NPS  $3\frac{1}{2}$  and smaller, square head plugs are provided (see Table I-1).

#### NOTE:

Cored plugs have metal thickness at all points equal to dimension *G*, except at the end of the thread. For tolerance, see para. 7(b).

#### Table I-3 Dimensions of Countersunk Plugs

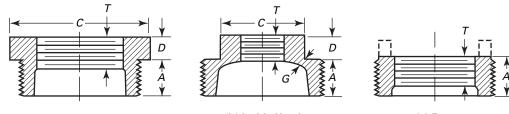


NPS	Minimum Thread Length, <i>A</i>	Nominal Size of Square Socket, <i>F</i> [Note (1)]	Size of Hexagon, <i>H</i> [Note (2)]	Metal Thickness, <i>G</i> [Note (3)]
1/8	0.37		3/16	0.06
<sup>1</sup> /4	0.44		1/4	0.09
3/8	0.48		5/16	0.13
<sup>1</sup> /2	0.56	3/8	3/8	0.16
3/4	0.63	1/2	9/16	0.18
1	0.75	1/2	5/8	0.20
$1^{1}/_{4}$	0.80	3/4		0.22
$1^{1}/_{2}$	0.83	3/4		0.24
2	0.88	7/8		0.26
2 <sup>1</sup> / <sub>2</sub>	1.07	11/8		0.29
3	1.13	13/8		0.31
$3^{1}/_{2}$	1.18	$1^{1}/_{2}$		0.34
4	1.22	2		0.37

GENERAL NOTE: Dimensions are in inches. NOTES:

- (1) Square socket within countersunk pattern shall have dimensions to fit commercial square bars of sizes indicated.
- (2) Hexagon socket within countersunk pattern shall have dimensions to fit regular wrenches used with hexagon socket set screws.
- (3) For metal thickness tolerance, see para. 7(b).





(a) Outside Head

(b) Inside Head

(c) Face

NPS [Note (1)]	Minimum Length of External Thread,	Minimum Length of Internal Thread, T	Minimum Height of Head, D	Minimum Width of Head, C [Note (3)]		Nominal Metal Thickness,
	<i>A</i> [Note (2)]			Outside	Inside	<i>G</i> [Note (4)]
<sup>1</sup> / <sub>4</sub> × <sup>1</sup> / <sub>8</sub>	0.44	0.26 [Note (5)]	0.14	0.64 [Note (6)]		
$\frac{3}{8} \times \frac{1}{4}$	0.48	0.40 [Note (5)]	0.16	0.68 [Note (6)]		
$\frac{3}{8} \times \frac{1}{8}$	0.48	0.25	0.16	0.68 [Note (6)]		
$\frac{1}{2} \times \frac{3}{8}$	0.56	0.41 [Note (5)]	0.19	0.87 [Note (6)]		
$\frac{1}{2} \times \frac{1}{4}$	0.56	0.32	0.19	0.87 [Note (6)]		
$3/8 \times 1/4$ $3/8 \times 1/4$ $3/8 \times 1/8$ $1/2 \times 3/8$ $1/2 \times 1/4$ $1/2 \times 1/8$	0.56	0.25	0.19	0.87 [Note (6)]		
$\frac{3}{4} \times \frac{1}{2}$ $\frac{3}{4} \times \frac{3}{8}$ $\frac{3}{4} \times \frac{1}{4}$ $\frac{3}{4} \times \frac{1}{8}$	0.63	0.53 [Note (5)]	0.22	1.15 [Note (6)]		
$\frac{3}{4} \times \frac{3}{8}$	0.63	0.36	0.22	1.15 [Note (6)]		
$\frac{3}{4} \times \frac{1}{4}$	0.63	0.32	0.22	1.15 [Note (6)]		
$\frac{3}{4} \times \frac{1}{8}$	0.63	0.25	0.22	1.15 [Note (6)]		•••
$1 \times \frac{3}{4}$	0.75	0.50	0.25	1.42 [Note (6)]		
$1 \times \frac{1}{2}$	0.75	0.43	0.25	1.42 [Note (6)]		
$1 \times \frac{3}{8}$	0.75	0.36	0.30		1.12	
$1 \times \frac{1}{4}$	0.75	0.32	0.30		1.12	
$1 \times \frac{1}{8}$	0.75	0.25	0.30		1.12	•••
$1^{1}/_{4} \times 1$	0.80	0.58	0.28	1.76		
$1^{1}/_{4} \times {}^{3}/_{4}$	0.80	0.50	0.28	1.76		
$1^{1}_{4} \times 1^{1}_{2}$ $1^{1}_{4} \times 3^{3}_{8}$	0.80	0.43	0.34	•••	1.34	0.18
$1\frac{1}{4} \times \frac{3}{8}$	0.80	0.36	0.34		1.12	0.18
$1\frac{1}{4} \times \frac{1}{4}$	0.80	0.32	0.34		1.12	0.18
$1\frac{1}{2} \times 1\frac{1}{4}$	0.83	0.77 [Note (5)]	0.31	2.00		
$1^{1}/_{2} \times 1$	0.83	0.58	0.31	2.00		
$1^{1}/_{2} \times {}^{3}/_{4}$	0.83	0.50	0.37		1.63	0.20
$1\frac{1}{2} \times \frac{1}{2}$	0.83	0.43	0.37		1.34	0.20
$1\frac{1}{2} \times \frac{3}{8}$	0.83	0.36	0.37		1.12	0.20
$1\frac{1}{2} \times \frac{1}{4}$	0.83	0.32	0.37		1.12	0.20
$2 \times 1^{1/2}$	0.88	0.70	0.34	2.48		
$2 \times 1^{1}/_{4}$	0.88	0.67	0.34	2.48		•••
2 × 1	0.88	0.58	0.41	•••	1.95	0.22
$2 \times \frac{3}{4}$	0.88	0.50	0.41	•••	1.63	0.22
$2 \times \frac{1}{2}$	0.88	0.43	0.41	•••	1.34	0.22
$2 \times \frac{3}{8}$	0.88	0.36	0.41	•••	1.12	0.22
$2 \times \frac{1}{4}$	0.88	0.32	0.41	• • •	1.12	0.22
$2^{1}/_{2} \times 2$	1.07	0.75	0.37	2.98		
$2\frac{1}{2} \times 1\frac{1}{2}$	1.07	0.70	0.44	2.68		
$2\frac{1}{2} \times 1\frac{1}{4}$	1.07	0.67	0.44		2.39	0.24
$2^{1}/_{2} \times 1$	1.07	0.58	0.44		1.95	0.24
$1\frac{1}{2} \times \frac{3}{4}$	1.07	0.50	0.44		1.63	0.24
$2^{1/2} \times \frac{1}{2}$	1.07	0.43	0.44		1.34	0.24

NDC	Minimum Length of External Thread,	Minimum Length of Internal Thread, T	Minimum Height of Head, D	Minimum Width of Head, <i>C</i> [Note (3)]		Nominal Metal Thickness,
NPS [Note (1)]	<i>A</i> [Note (2)]			Outside	Inside	G [Note (4)]
$3 \times 2^{1/2}$	1.13	0.92	0.40	3.86		
3 × 2	1.13	0.75	0.48	3.28		
$3 \times 1^{1/_{2}}$	1.13	0.70	0.48		2.68	0.26
$3 \times 1^{1/4}$	1.13	0.67	0.48		2.39	0.26
3 × 1	1.13	0.58	0.48		1.95	0.26
$3 \times \frac{3}{4}$	1.13	0.50	0.48		1.63	0.26
$3 \times \frac{1}{2}$	1.13	0.43	0.48		1.34	0.26
$3^{1}/_{2} \times 3$	1.18	0.98	0.43	4.62		
$3^{1}/_{2} \times 2^{1}/_{2}$	1.18	0.92	0.52	3.86		
$3^{1}/_{2} \times 2^{1}$	1.18	0.75	0.52		3.28	0.28
$3^{1}/_{2} \times 1^{1}/_{2}$	1.18	0.70	0.52		2.68	0.28
$3^{1}/_{2} \times 1^{1}/_{4}$	1.18	0.67	0.52		2.39	0.28
$3^{1}/_{2} \times 1$	1.18	0.58	0.52		1.95	0.28
$4 \times 3^{1/2}$	1.22	1.03	0.50	5.20		
4 × 3	1.22	0.98	0.50	4.62		
$4 \times 2^{1/2}$	1.22	0.92	0.60		3.86	0.31
4 × 2	1.22	0.75	0.60		3.28	0.31
$4 \times 1^{1/2}$	1.22	0.70	0.60		2.68	0.31
$4 \times 1^{1}/_{4}$	1.22	0.67	0.60		2.39	0.31
4 × 1	1.22	0.58	0.60		1.95	0.31
5 × 4	1.31	1.08	0.50	5.79		
5 × 3 <sup>1</sup> / <sub>2</sub>	1.31	1.03	0.60	5.20		
5 × 3	1.31	0.98	0.60		4.62	0.38
$5 \times 2^{1/2}$	1.31	0.92	0.60		3.86	0.38
5 × 2	1.31	0.75	0.60		3.28	0.38
6 × 5	1.40	1.18	0.63	7.05		
6 × 4	1.40	1.08	0.75		5.79	0.43
6 × 3 <sup>1</sup> / <sub>2</sub>	1.40	1.03	0.75		5.20	0.43
6 × 3	1.40	0.98	0.75		4.62	0.43
$6 \times 2^{1/2}$	1.40	0.92	0.75		3.86	0.43
6 × 2	1.40	0.75	0.75		3.28	0.43
8 × 6	1.57	1.28	0.83	8.28		
8 × 5	1.57	1.18	0.83		7.05	0.55
8 × 4	1.57	1.08	0.83		5.79	0.55
8 × 3 <sup>1</sup> / <sub>2</sub>	1.57	1.03	0.83		5.20	0.55
8 × 3	1.57	0.98	0.83		4.62	0.55

Table I-4 Dimensions of Outside Head, Inside Head, and Face Bushings (Cont'd)

#### **GENERAL NOTES:**

(a) Dimensions are in inches.

(b) Cored bushings have minimum metal thickness at all points equal to dimension G, except at the end of the thread.

NOTES:

(1) Hexagon head or octagon head bushings NPS 2<sup>1</sup>/<sub>2</sub> and smaller reducing one NPS shall not be made of gray cast iron. Other NPS may be made of either cast iron or malleable iron, ductile iron, or steel. Face bushings NPS 2<sup>1</sup>/<sub>2</sub> and smaller shall not be made of gray cast iron. Face bushings NPS 3 and larger reducing one NPS shall not be made of gray cast iron. Face bushings NPS 3 and larger reducing two NPS or more may be made of either cast iron or malleable iron, ductile iron, ductile iron, or steel.

(2) In the case of outside head bushings, length A includes provisions for imperfect threads (see also section 8).

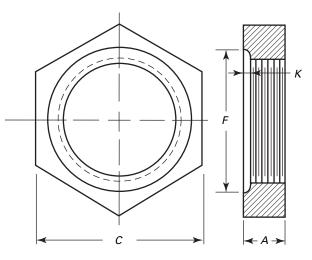
(3) Heads of bushings shall be hexagonal or octagonal, except that on the larger sizes of outside head bushings, the heads may be made round with lugs instead of hexagonal or octagonal.

(4) G is the same as metal thickness for Class 125 Cast Iron Threaded Fittings (ASME B16.4). For tolerance, see para. 7(b).

(5) To provide proper metal thickness, these NPS shall not be cored out to diameters greater than the root diameter of the internal thread. The length of the internal thread may be equal to the minimum dimension, *T*, or greater up to the full length of bushing.

(6) When made of bar stock, the dimensions may be  $\frac{5}{8}$  in.,  $\frac{11}{16}$  in.,  $\frac{7}{8}$  in.,  $1\frac{1}{8}$  in., and  $1\frac{7}{16}$  in., respectively, in order to use regular bar stock sizes.

#### Table I-5 Dimensions of Locknuts



		Minimum Wi Across Flat			
NPS	Minimum Nominal Thickness, <i>A</i>	C [Note (1)]		Minimum Diameter of Packing Recess,	Depth of Packing Recess,
		Malleable Iron, Ductile Iron, Steel	Cast Iron	F [Note (2)]	<i>K</i> [Note (2)]
<sup>1</sup> / <sub>8</sub>	0.19	0.69 [Note (3)]		0.50	0.04
1/4	0.25	0.84 [Note (3)]		0.66	0.06
3/8	0.28	1.00 [Note (3)]		0.77	0.06
<sup>1</sup> / <sub>4</sub> <sup>3</sup> / <sub>8</sub> <sup>1</sup> / <sub>2</sub>	0.31	1.18 [Note (3)]		0.97	0.06
3/4	0.34	1.43 [Note (3)]		1.23	0.06
1	0.38	1.75		1.50	0.06
$1^{1}/_{4}$	0.42	2.10		1.86	0.06
1 <sup>1</sup> / <sub>2</sub>	0.47	2.35		2.12	0.06
2	0.53	2.88		2.63	0.09
$2^{1}/_{2}$	0.59	3.50	3.86	3.18	0.09
3	0.67	4.27	4.62	3.84	0.09
$3^{1}/_{2}$	0.73	4.84	5.20	4.38	0.13
4	0.80	5.38	5.79	5.00	0.13

GENERAL NOTE: Dimensions are in inches.

NOTES:

(1) NPS  $3\frac{1}{2}$  and smaller are hexagonal; NPS 4 may be either hexagonal or octagonal.

(2) The packing recess surface may be as cast. When made from bar stock, the recess may be tool finished.
(3) Locknuts in these NPS may be made from bar stock, in which case dimension C may be <sup>11</sup>/<sub>16</sub> in., <sup>7</sup>/<sub>8</sub> in., 1 in., 1<sup>3</sup>/<sub>16</sub> in., and 1<sup>7</sup>/<sub>16</sub> in., respectively, in order to conform with regular hexagon bar stock sizes.

## MANDATORY APPENDIX II REFERENCES

The following is a list of standards and specifications referenced in this Standard:

- ANSI/ASME B1.20.1-1983 (R2001), Pipe Threads, General Purpose (Inch)
- ASME B16.3-2006, Malleable Iron Threaded Fittings: Classes 150 and 300
- ASME B16.4-2011, Gray Iron Threaded Fittings: Classes 125 and 250
- ASME B18.2.1-2012, Square, Hex, Heavy Hex, and Askew Head Bolts and Hex, Heavy Hex, Hex Flange, Lobed Head, and Lag Screws (Inch Series)
- Publisher: The American Society of Mechanical Engineers (ASME), Two Park Avenue, New York, NY

10016-5990; Order Department: 22 Law Drive, P.O. Box 2900, Fairfield, NJ 07007-2900 (www.asme.org)

- ASTM A126-04(2009), Specification for Gray Iron Castings for Valves, Flanges, and Pipe Fittings
- ASTM E29-08, Standard Practice for Using Significant Digits in Test Data to Determine Conformance With Specifications
- Publisher: American Society for Testing and Materials (ASTM International), 100 Barr Harbor Drive, P.O. Box C700, West Conshohocken, PA 19428-2959 (www.astm.org)

## NONMANDATORY APPENDIX A QUALITY SYSTEM PROGRAM

The products manufactured in accordance with this Standard shall be produced under a quality system program following the principles of the ISO 9000 series.<sup>1</sup> A determination of the need for registration and/or certification of the product manufacturer's quality system program by an independent organization shall be the responsibility of the manufacturer. The detailed documentation demonstrating program compliance shall be available to the purchaser at the manufacturer's facility. A written summary description of the program utilized by the product manufacturer shall be available to the purchaser upon request. The product manufacturer is defined as the entity whose name or trademark appears on the product in accordance with the marking or identification requirements in this Standard.

<sup>&</sup>lt;sup>1</sup> The series is also available from the American National Standards Institute (ANSI) and the American Society for Quality (ASQ) as American National Standards that are identified by the prefix "Q," replacing the prefix "ISO." Each standard of the series is listed under References in Mandatory Appendix II.

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